

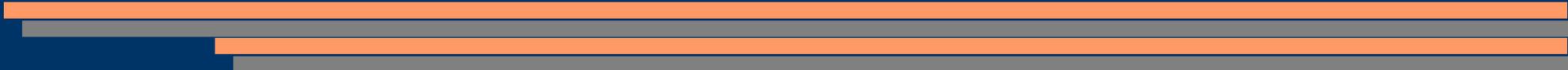
Building a System to Perform Fluid Dynamics Simulations and Experiments

***OU Supercomputing Symposium
Oct. 6, 2010***

Willy Duffle, Chris Kiser, Grant Armstrong, Andrew Baker,
Andrew Henderson, Alex Macharia, Lamar Williams, and
Evan Lemley

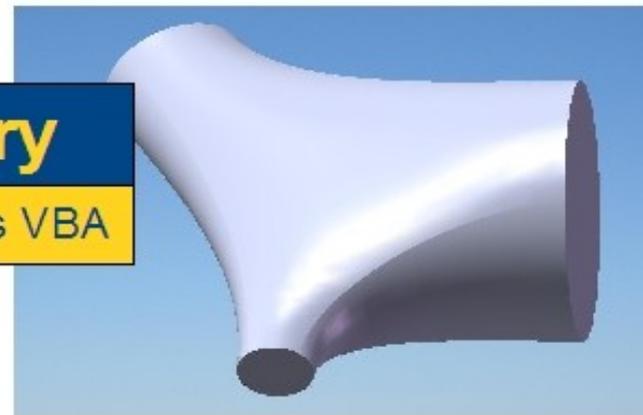
***Department of Engineering and Physics
University of Central Oklahoma***

Dimitrios Papavassiliou, Henry Neeman, Ed O'Rear
University of Oklahoma



Generate Junction Geometry

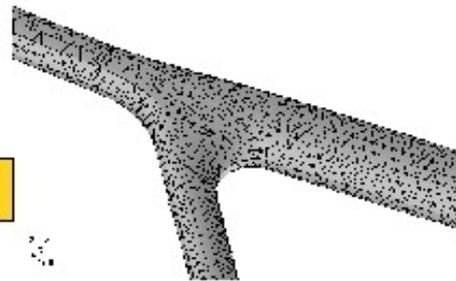
Junction code (C++) → SolidWorks VBA



Simulations

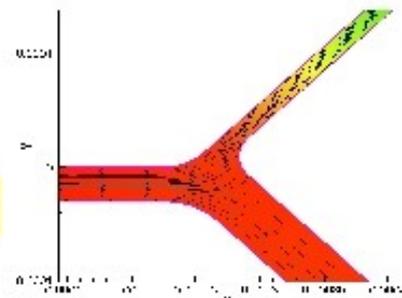
Generate Mesh

Junction Code → GAMBIT



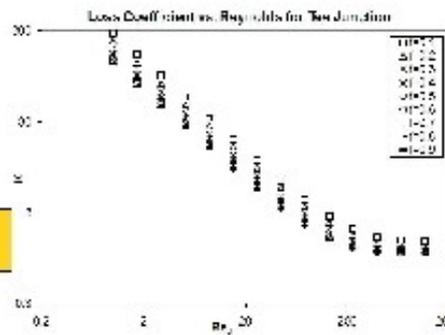
Simulate Flow

Junction Code → FLUENT



Analyze Results

Junction Code → tsv

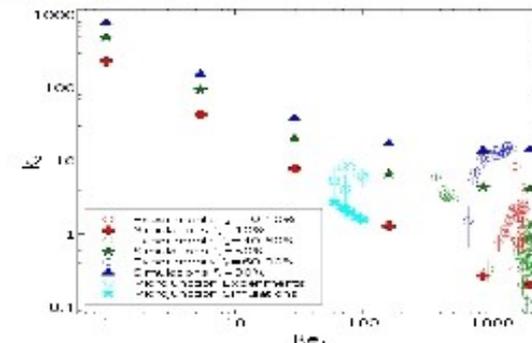
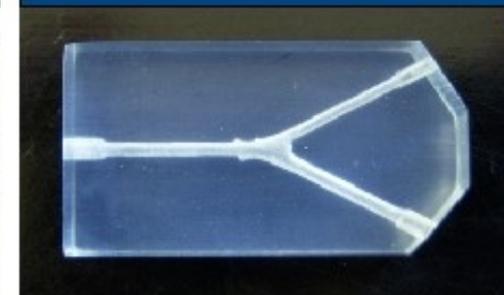


Experiments

Milliscale



Microscale



Overview

Background

Simulations

- Flow Through Porous Media (FTPM)

- Flow in Junctions/Microjunctions

- Microelbows

- Entrance Length in Microtubes

- Reynolds Number Dependence tees and wyes

Experimental

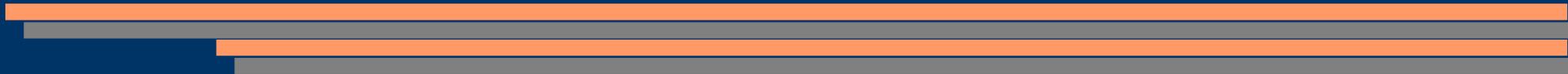
- Millijunctions & MicroJunctions

Current and Future Directions

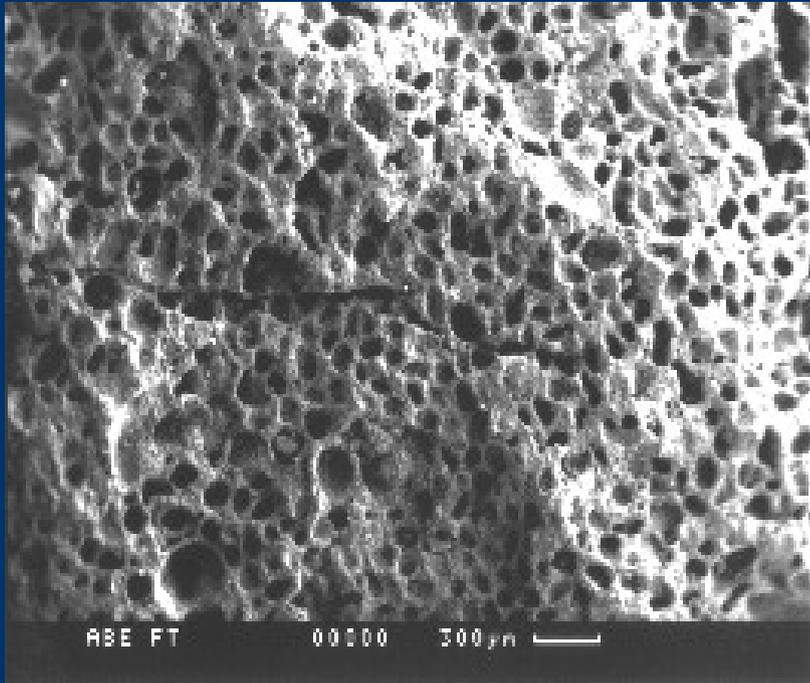
- Bio-Scaffolds

- Renal Artery Aneurysm

- Microtubes

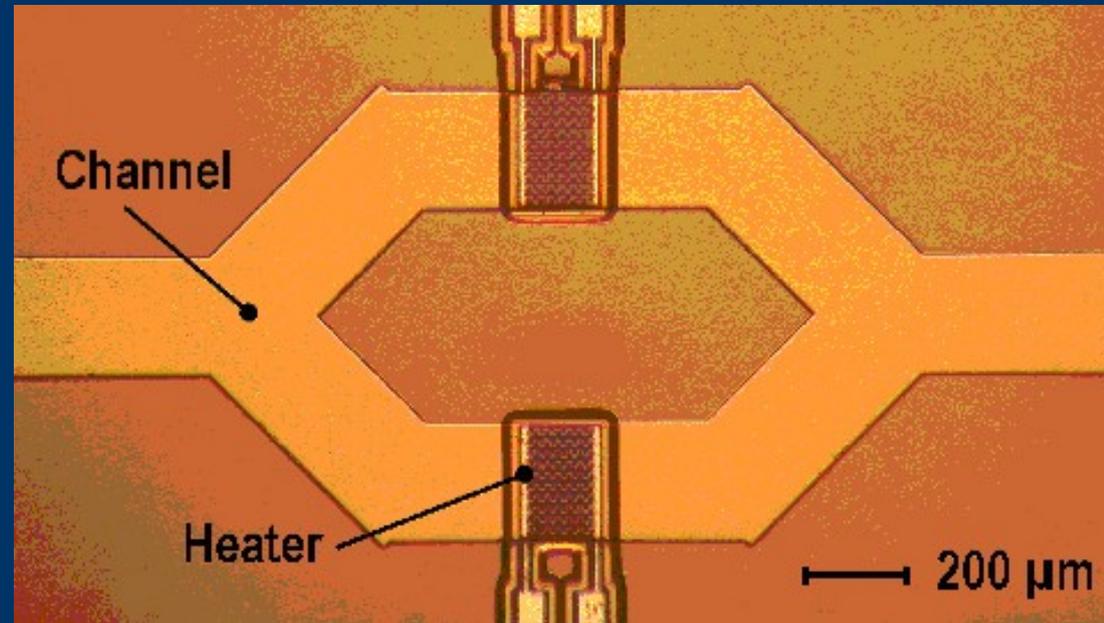


Porous Media



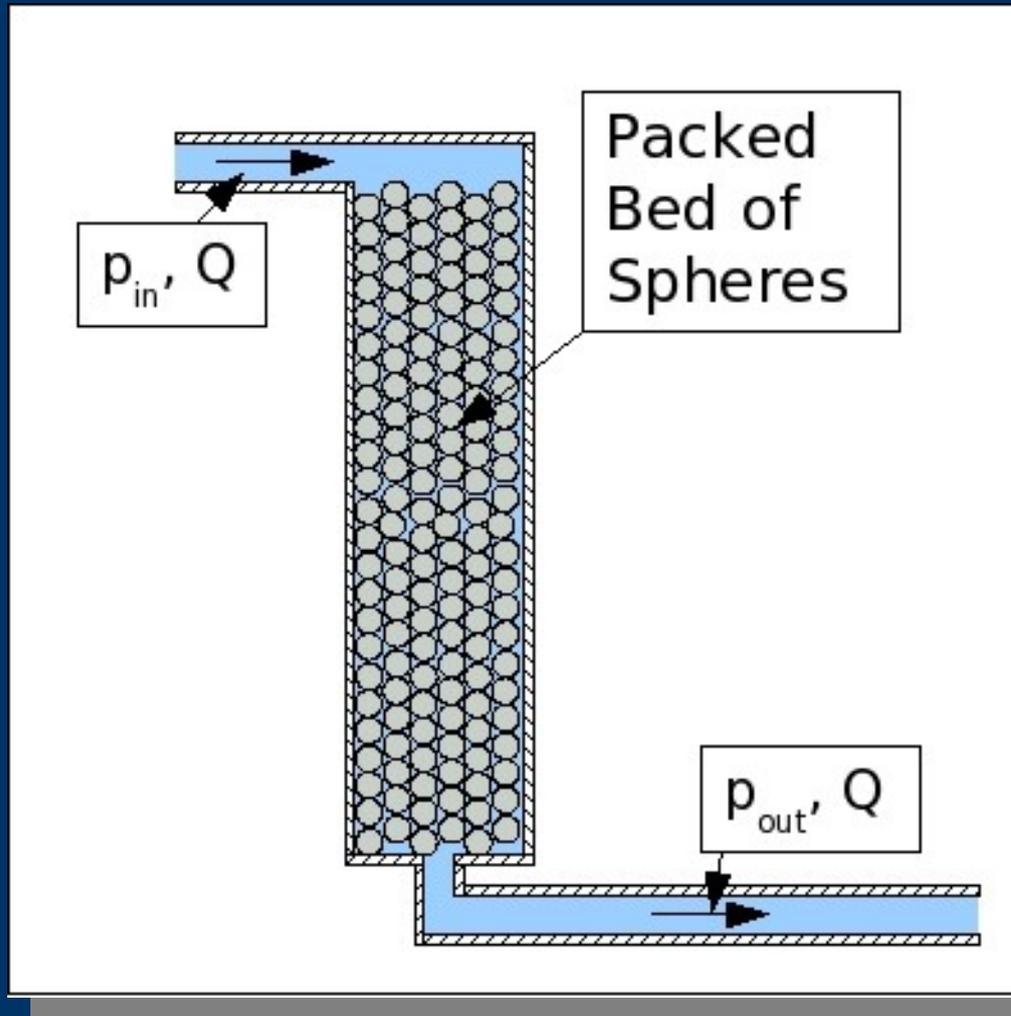
Highly porous magnesiumian limestone.
(www.dawntnicholson.org.uk)

Microfluidic Devices



Microfluidic Valve Structure.
(<http://www.cchem.berkeley.edu/sjmgrp/people/boris/boris.htm>)

Artificial Porous Media



Packed Beds, Gas and Liquid Filters
Sphere sizes μm to cm
Hold-up for chemical reaction, thermal processing, or filtering

Basics of Porous Media

Low Speed Flow – Darcy's Law

$$\frac{dp}{dx} = \frac{\mu}{\kappa} u$$

p = pressure x = position
 μ = viscosity κ = permeability
 u = filtration velocity

High Speed Flow – Forchheimer's Law

$$\frac{dp}{dx} = \frac{\mu}{\kappa} u + \rho \beta u^2$$

ρ = density
 β = Forchheimer's Coefficient

Packed Beds – Ergun's Equation (empirical)

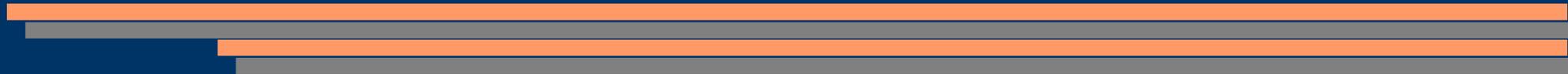
$$\frac{-\Delta p}{\Delta L} = \frac{150 \mu (1-\phi)^2}{d_p^2 \phi^3} u + \frac{175 \rho (1-\phi)}{d_p \phi^3} u^2$$

ϕ = porosity
 d_p = mean sphere diameter

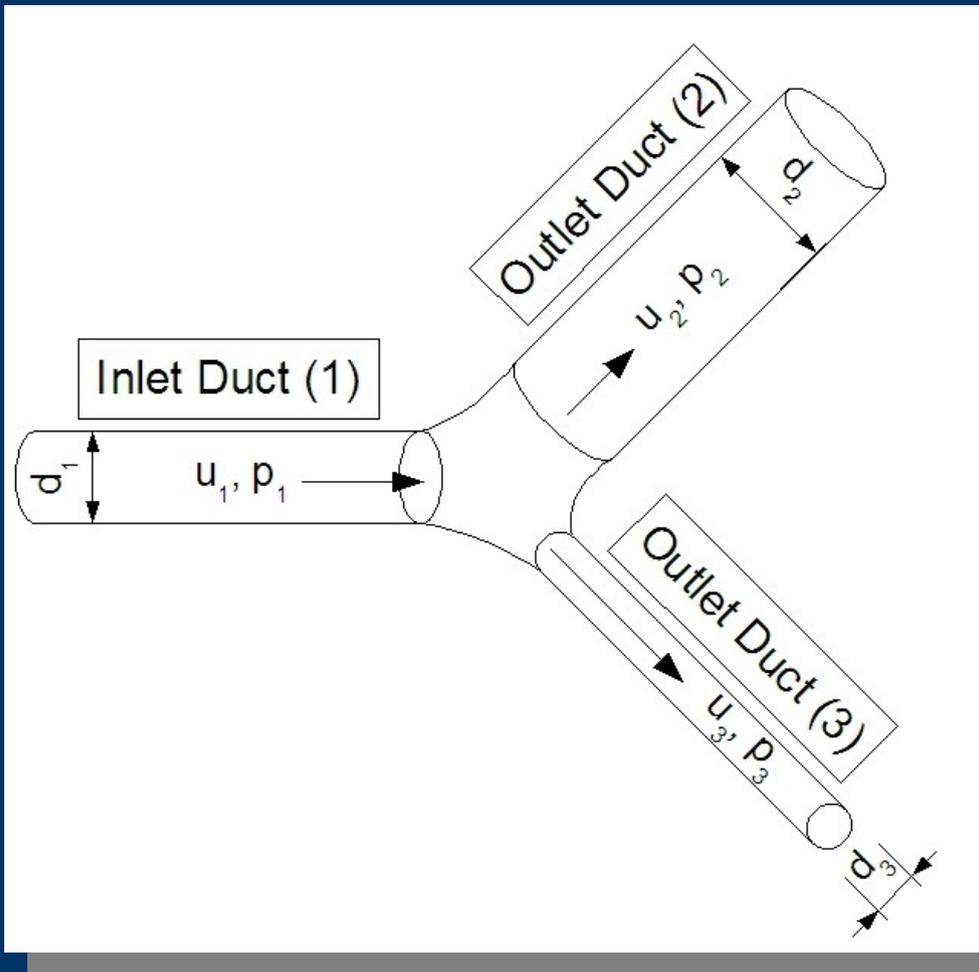
Flow Through Porous Media

Collaborative Effort with Dimitrios Papavassiliou and Henry Neeman from OU (began Fall 2004)
Simulation of Flow of Fluids through Porous Media

Code FTPM – Flow Through Porous Media.
Solves for velocity and pressure at pore junctions in a randomly generated pore network.

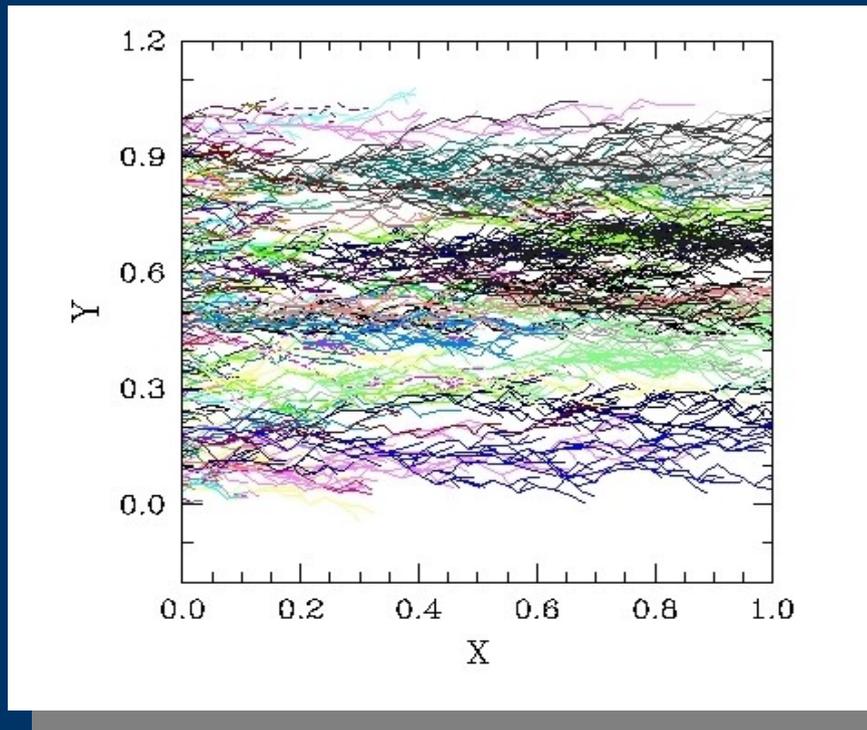


Flow Network Analysis



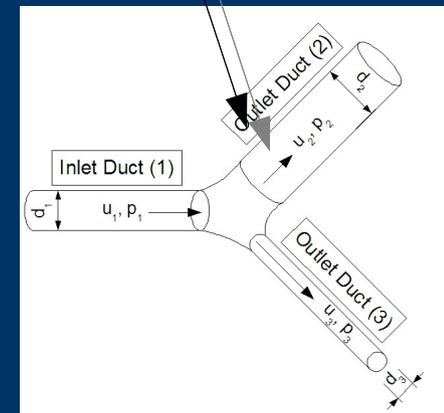
Design and Analysis of networks depends on knowledge of flow and energy losses in arbitrary branches. No systematic studies to generalize these bifurcations

Porous Network Simulator (Collaboration with Univ. of Oklahoma)

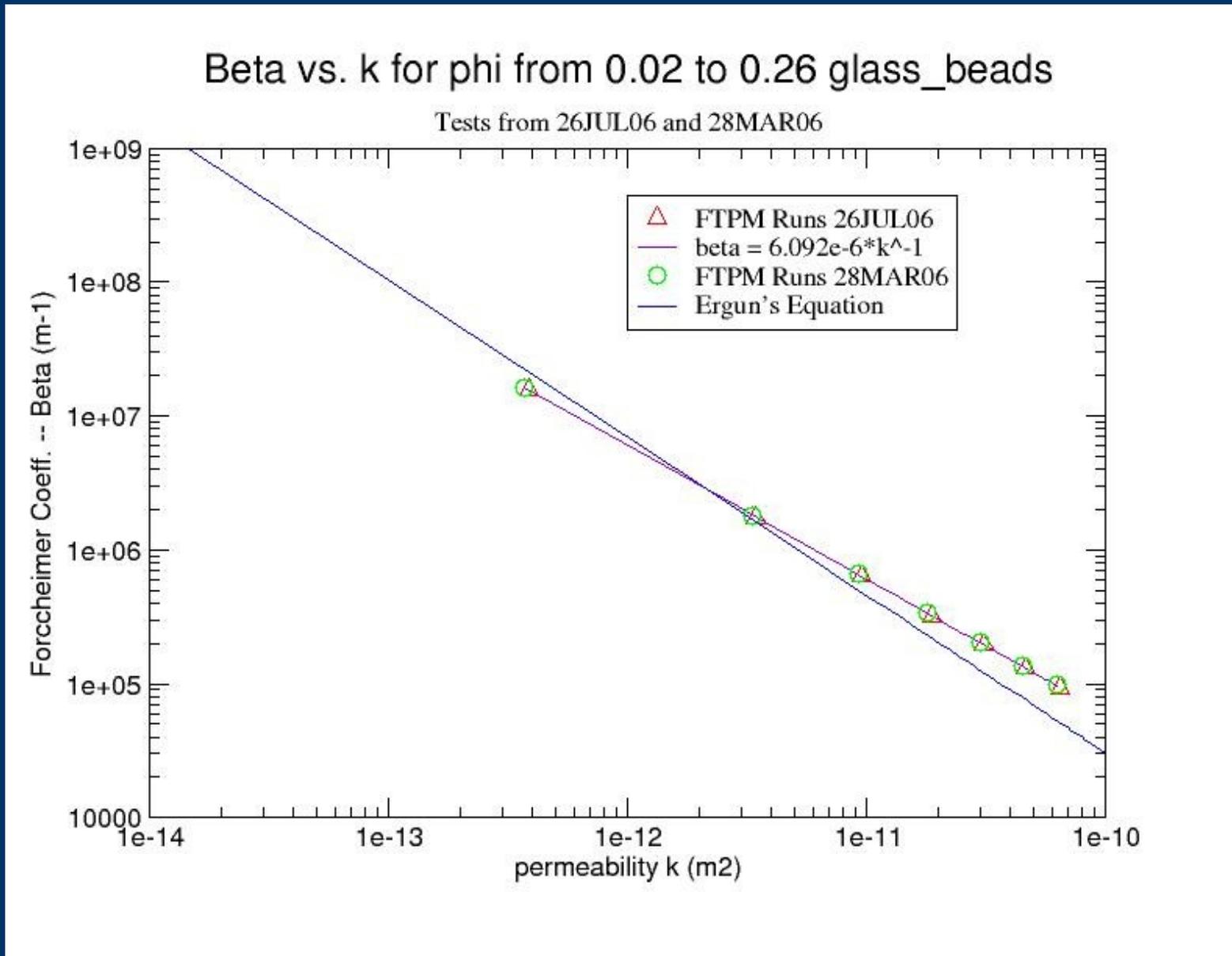


Projection on the xy plane of a 3D network that has 200 entry points at $x=0$, porosity equal to 10% and a range of $\pm 60^\circ$ relative to the x axis and $\pm 30^\circ$ relative to the y axis.

3D Monte Carlo networks from normal, beta, or empirical distribution (pore size pdf)
Coordination Number (1, 2, 3)
number of pores entering and leaving a junction
 $\theta \pm 90^\circ$

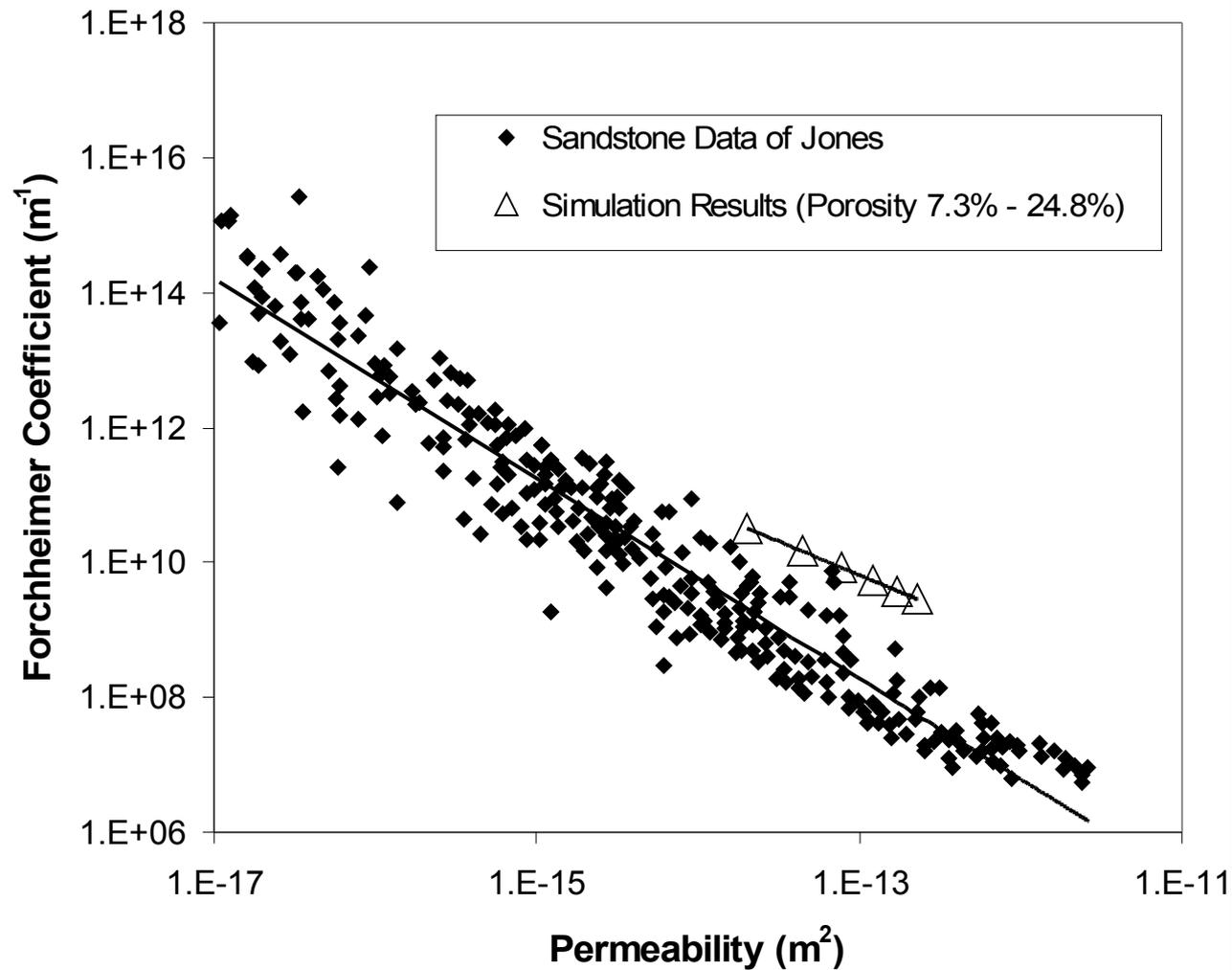


FTPMP Results



FTPMP Results

Forchheimer Coefficient versus Permeability Comparison of Simulation to Empirical Results



Literature

T's and Y's – limited configurations and most are for turbulent flow

Basset, M.D., Winterbone, D.E., and Pearson, R.J., 2001, "Calculation of Steady Flow Pressure Loss Coefficients for Pipe Junctions," Proc. Instn. Mech. Engrs., Part C, Journal of Mechanical Engineering Science, **215** (8), pp. 861-881.

W.H. Hager, 1984, "An Approximate Treatment of Flow in Branches and Bends," Proc. Instn. Mech. Engrs., Part C, Journal of Mechanical Engineering Science, **198**(4) pp. 63-9.

Blaisdell, F.W., and Manson, P.W., 1967, "Energy loss at pipe junctions," J. Irrig. and Drainage Div., ASCE, **93**(IR3), pp. 59-78.

Schohl, G.A., 2003, "Modeling of Tees and Manifolds in Networks," *Proceedings of the 4th ASME/JSME Joint Fluids Engineering Conference*, **2**, Part D, pp. 2779-2786.

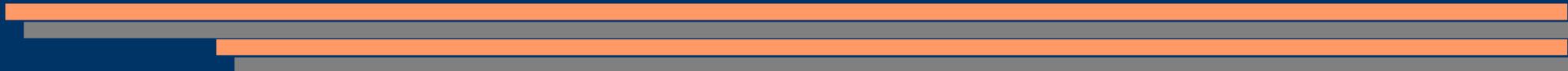
Bassett, M.D., Pearson, R.J., and Winterbone, D.E., 1998, "Estimation of Steady Flow Loss Coefficients for Pulse Converter Junctions in Exhaust Manifolds," *IMEchE Sixth International Conference on Turbocharging and Air Management Systems*, IMechE HQ, London, UK, **C554/002**, pp.209-218.

Ruus, E., 1970, "Head Losses in Wyes and Manifolds," J. Hyd. Div., ASCE, **96**(HY3), 593-608.

Laminar loss coefficients and elbows, reductions, contractions – much larger loss coefficients than turbulent case – strong dependence on Reynold's number.

Edwards, M.F., Jadallah, M.S.M., and Smith, R., 1985, "Head Losses in Pipe Fittings at Low Reynolds Numbers," Chem. Engr. Res. Des., **63**(1), pp. 43-50.

Importance of roughness at microscale



Problem Description

Stagnation Loss Coefficient

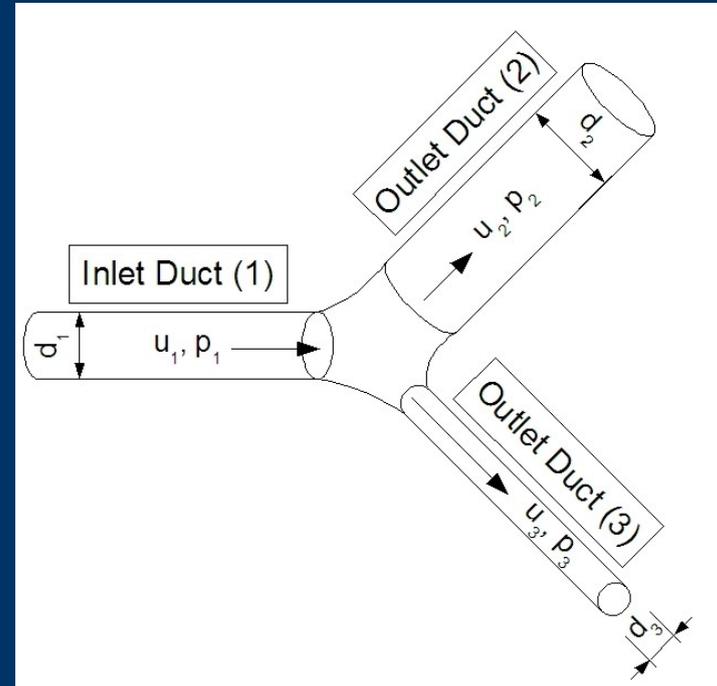
$$K_2 = \frac{\left[\left(\frac{p_1}{\rho} + \frac{u_1^2}{2} \right) - \left(\frac{p_2}{\rho} + \frac{u_2^2}{2} \right) \right]}{\frac{u_1^2}{2}}$$

Parameters:

$d_2/d_1, d_3/d_1$

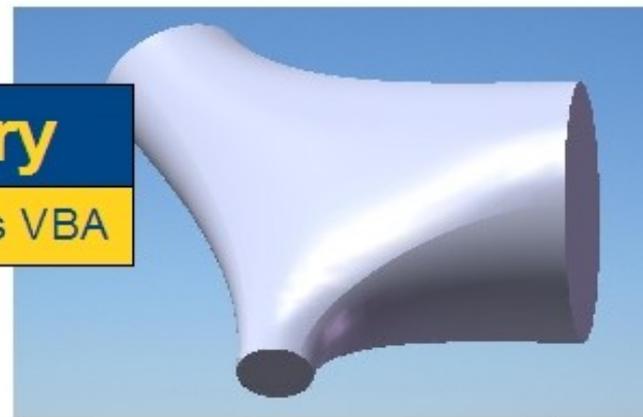
θ_2 and θ_3

f_2 (this sets f_3) – (why? other literature and possibility of simulations where this is unknown initially)



Generate Junction Geometry

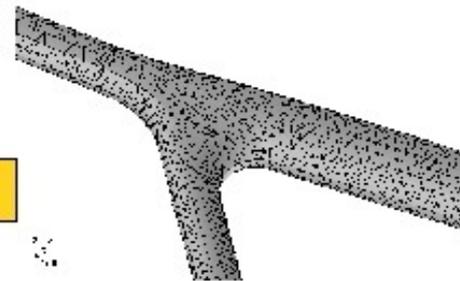
Junction code (C++) → SolidWorks VBA



Simulations

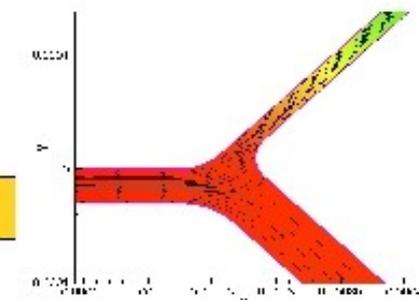
Generate Mesh

Junction Code → GAMBIT



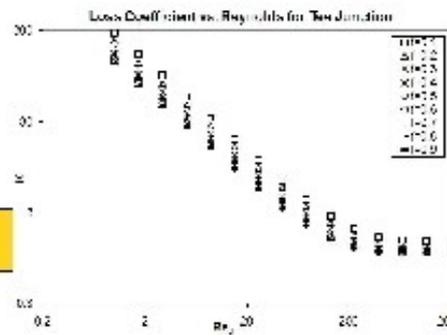
Simulate Flow

Junction Code → FLUENT



Analyze Results

Junction Code → tsv

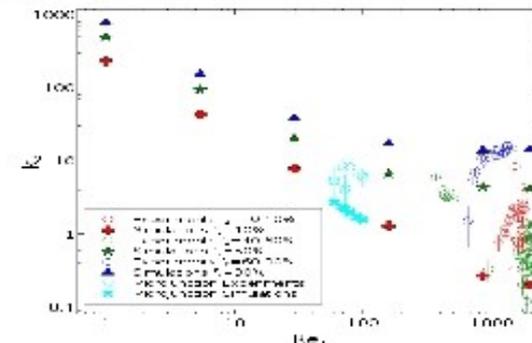
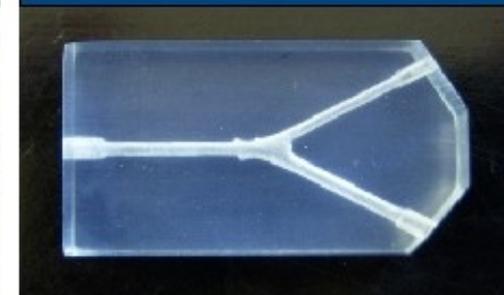


Experiments

Milliscale



Microscale



Automation of Geometry Generation and CFD Runs

Custom Code was written to:

(a) create GAMBIT journal files that instantiate the desired geometry based on existing 2D geometries.

(c) create a script that loads journal files into GAMBIT and meshes

(d) create all necessary preprocessing files for FLUENT.

(e) create post-processing files for FLUENT results and to tabulate results for a complete set of runs

Solution Methodology

2D Geometry Generalization

$$L_1 = 5 d_{\max}$$

$$L_2 \text{ and } L_3 = 10 d_{\max}$$

If $d_2 > d_{\text{avg}}$,
then $r_2 = 3d_2$;

else $r_2 = 2d_2$

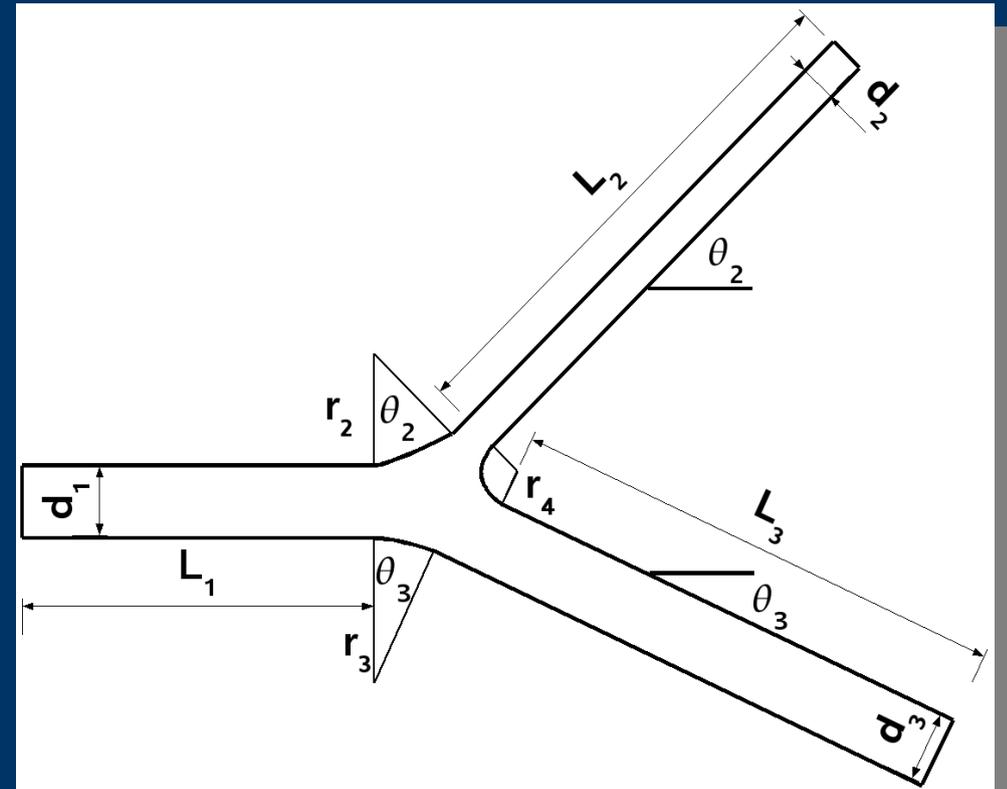
If $d_3 > d_{\text{avg}}$,
then $r_3 = 3d_3$;

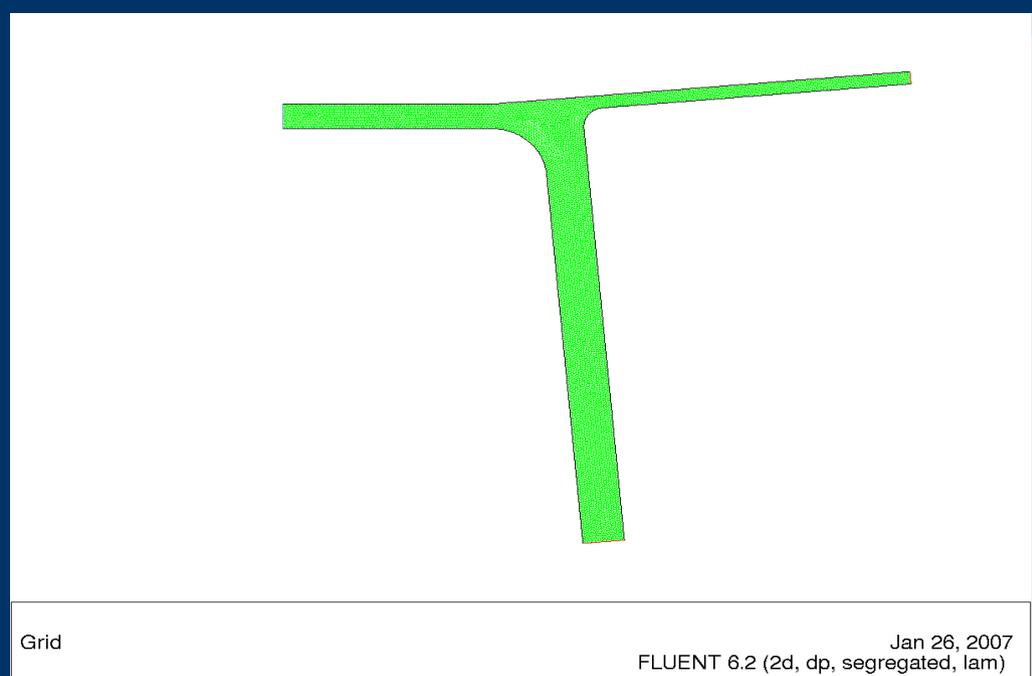
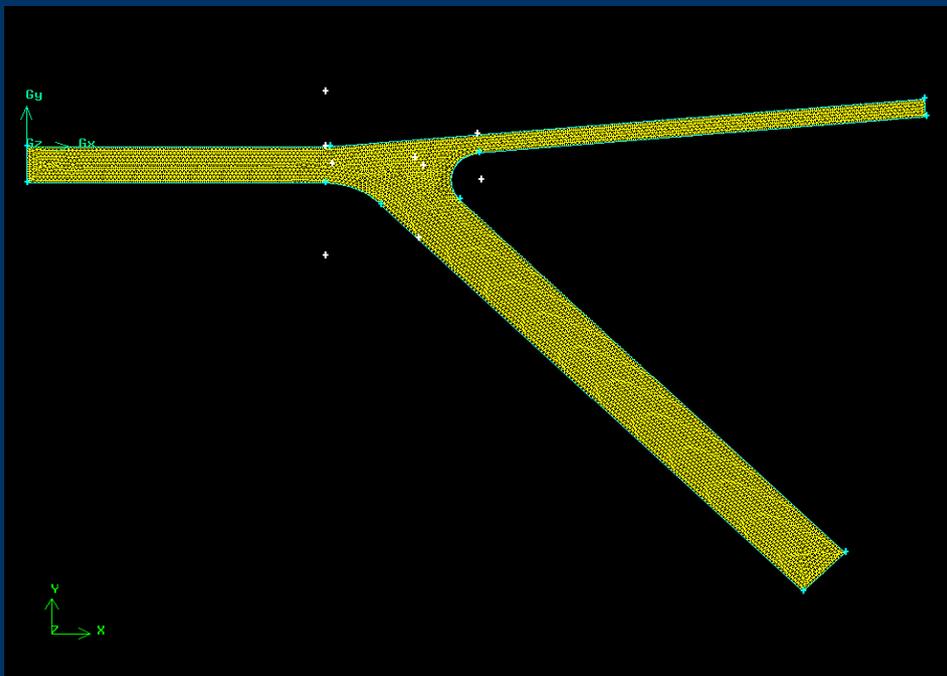
else $r_3 = 2d_3$

$$r_4 = d_{\max} / 2$$

Generalized Geometry

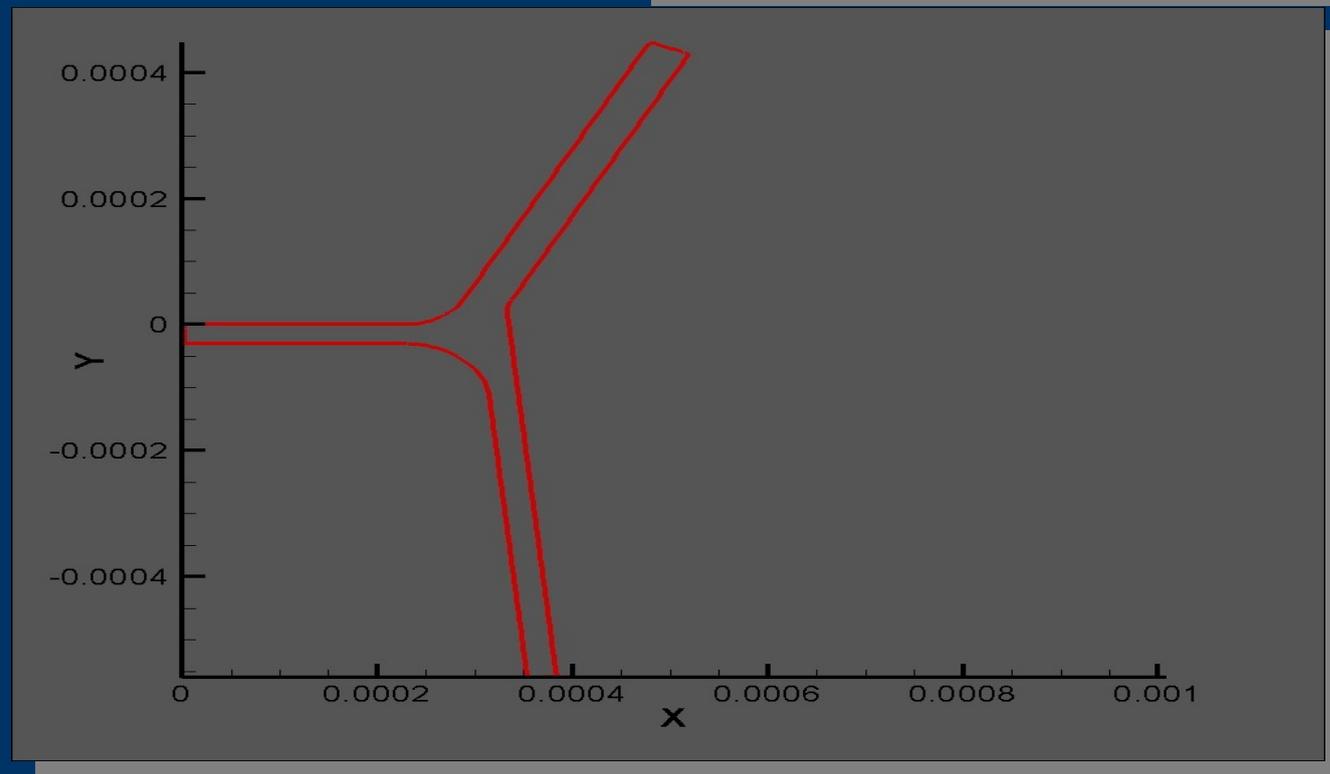
larger and smaller outlet ducts ; 2Dim. - 3Dim are underway
avoid sharp edges ; 5 – 90 degrees for angles



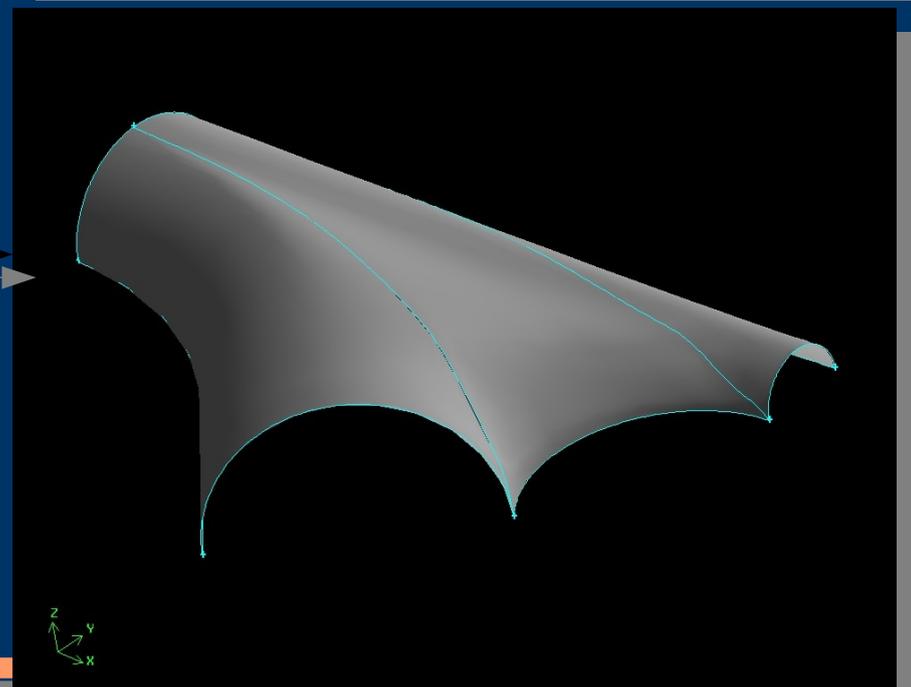
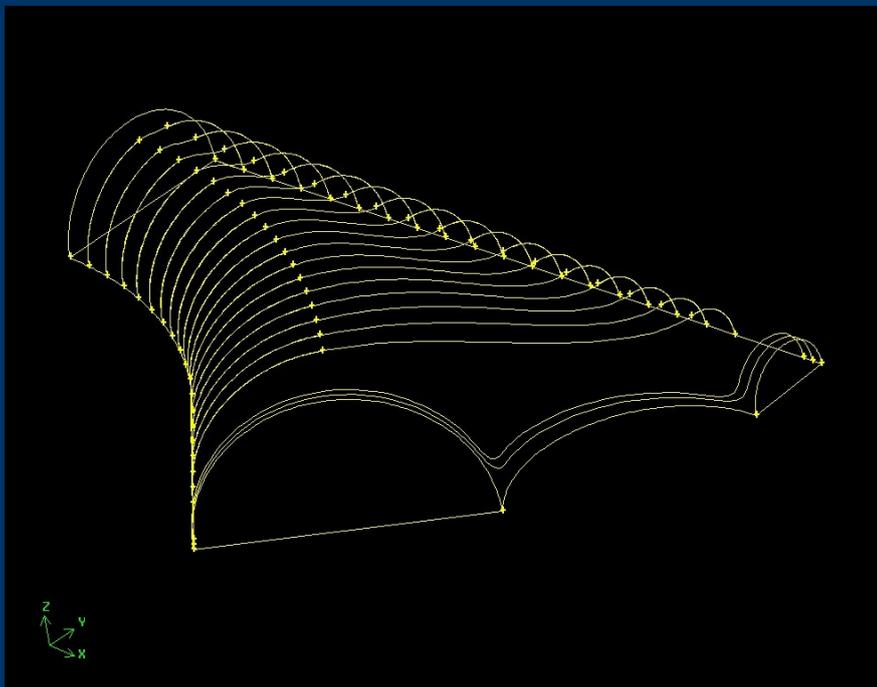
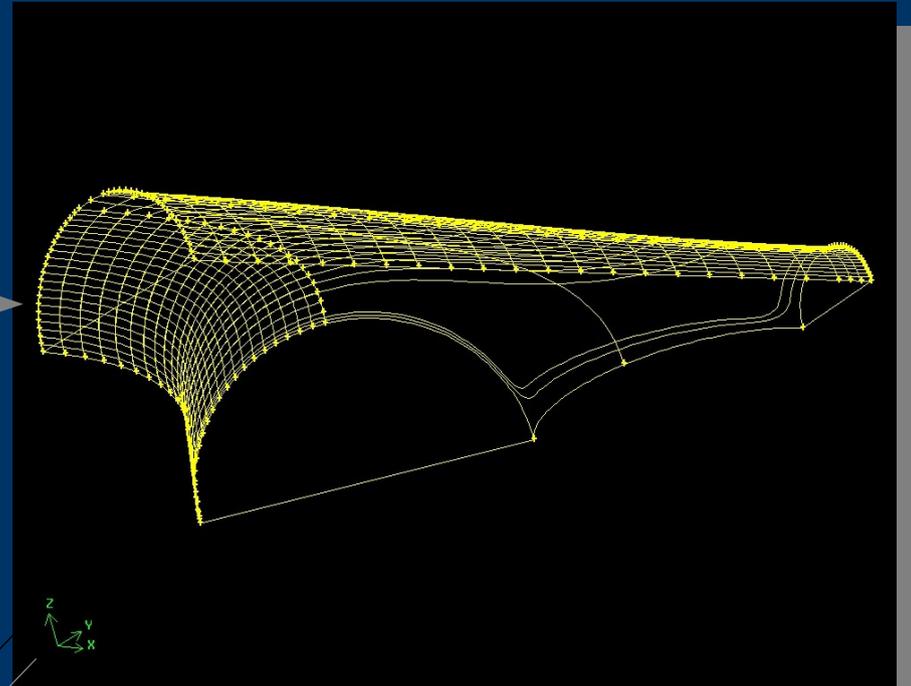
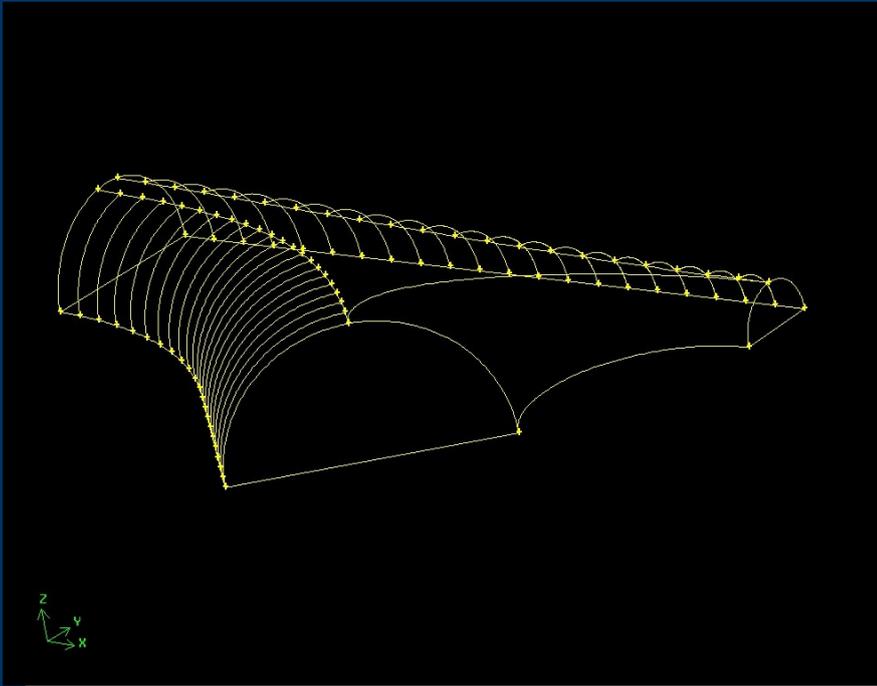


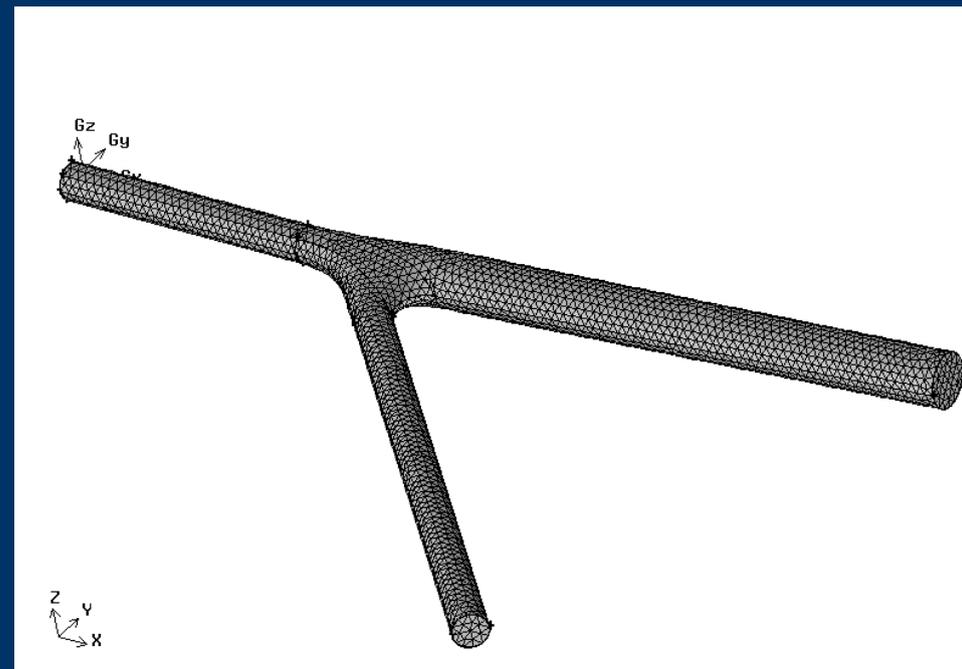
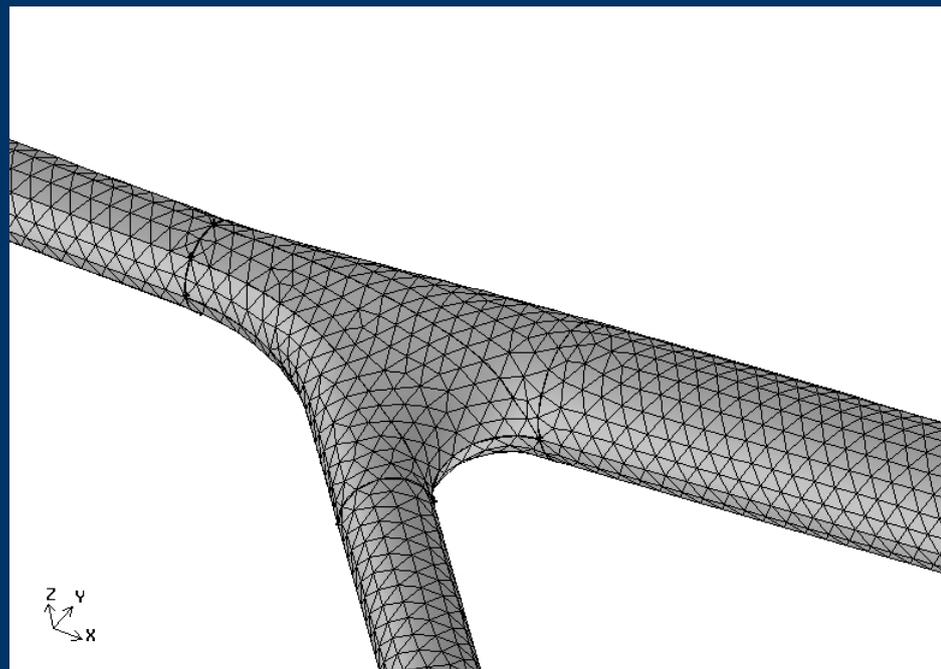
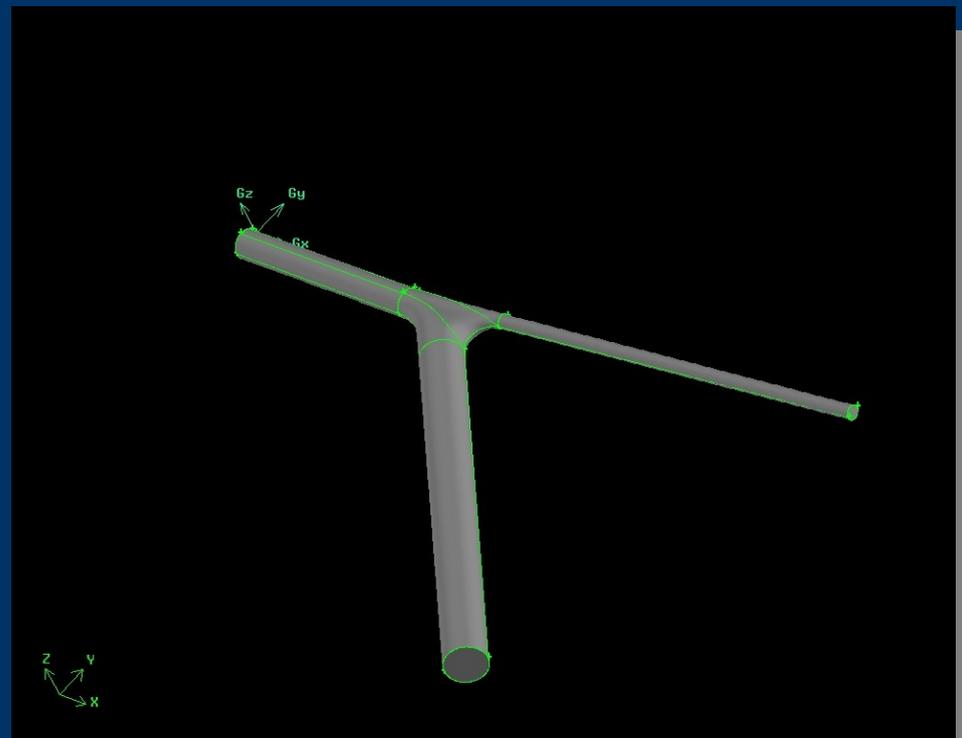
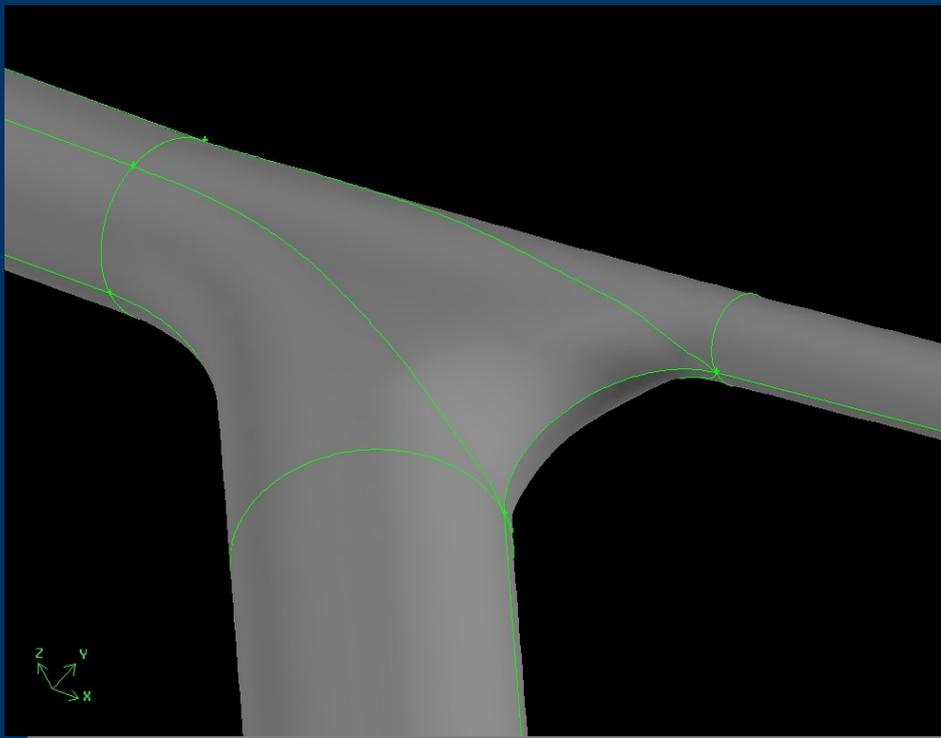
Grid

Jan 26, 2007
FLUENT 6.2 (2d, dp, segregated, lam)



3D Geometry – Mark I

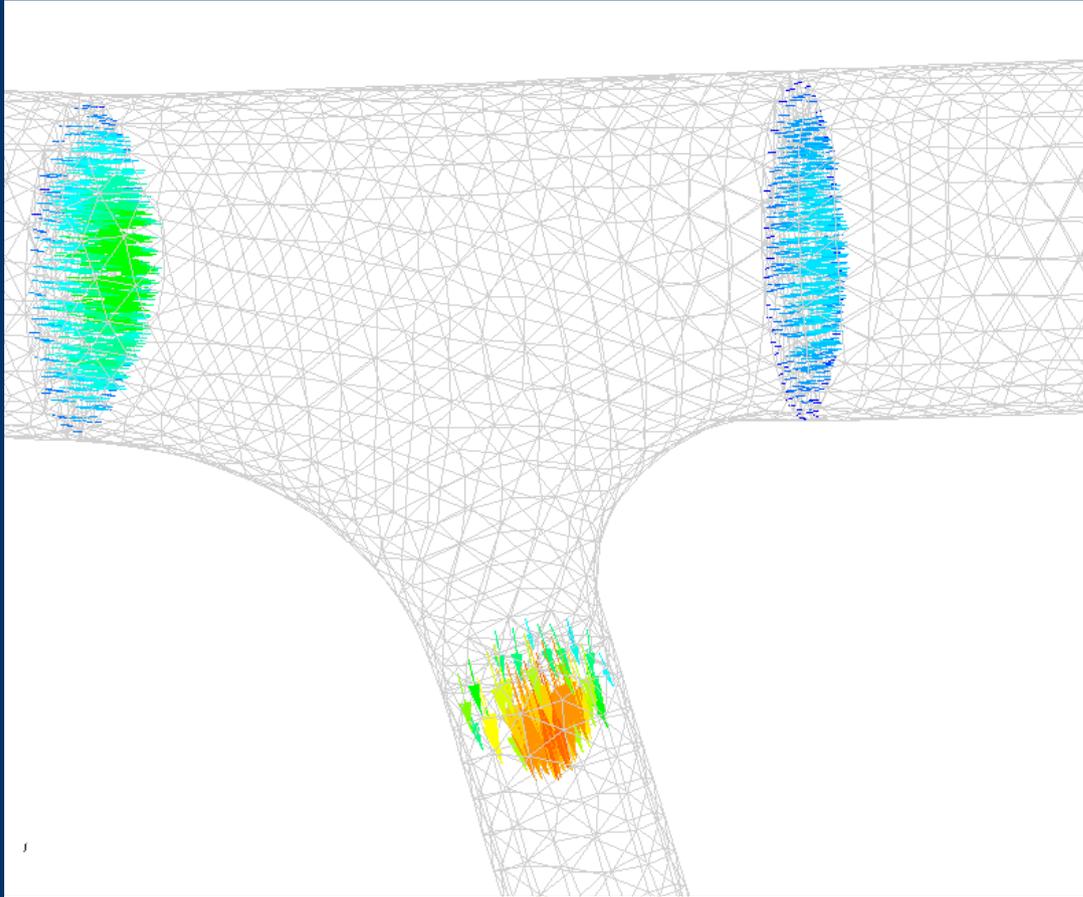




Simulation Parameters

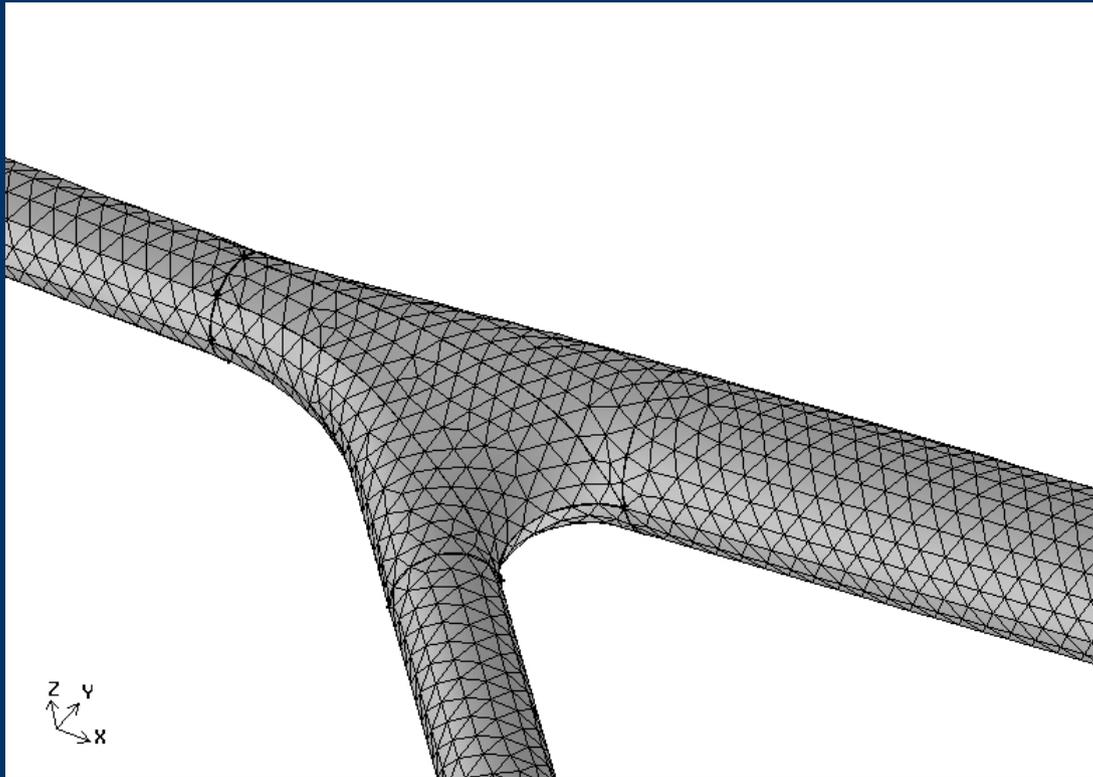
- Re_D (Reynolds Number = proportional to speed) was maintained at constant value at the inlet duct
- d_1 was 30 microns. The fluid was chosen to be liquid water at 20°C. The inlet flow velocity, u_1 , was set to 0.5 m/s giving a Reynolds number of 15
- $Le_D = 0.06Re_D D$ --- gives 0.9 D for $Re_D = 15$
- FLUENT output files contain surface averaged static pressure and magnitude of flow velocity at duct cross-sections at the duct inlets and exits.

Duct Inlet/Outlets



Duct inlet and exit sections considered to be where geometry of duct is the same as the downstream portion for outlets and upstream portion for inlet.

Mesh Automation



Mesh was set to 1/4 of smallest duct

Tetrahedral Mesh

Large number of tests to assess ability to generalize the mesh (1/4 factor determined in this manner)

Some testing to verify no change in results with change in mesh size.

Inlet was specified as velocity inlet

Outlets were outflow boundaries – allowed specification of flow fraction

Numerical Methods

Finite Volume solution of integral Navier Stokes

Steady-State 3D

Implicit

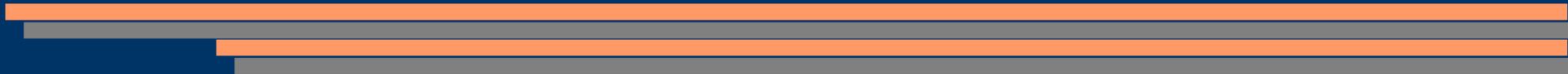
SIMPLE for pressure velocity coupling

1st order upwind scheme of momentum

discretization

Max number of iterations

Convergence criterion = 0.1%



Parameter Values

d_2/d_1 and $d_3/d_1 = 0.5, 1.0, 1.5$

$f_2 = 0.1, 0.3, 0.5, 0.7, 0.9$

θ_2 and $\theta_3 = 5^\circ, 25^\circ, 45^\circ, 65^\circ, 85^\circ$

600 runs attempted – 475 completed (*geometry issues on remainder*)

Suite of C++ procedures to create geometries, input files, read and collate results

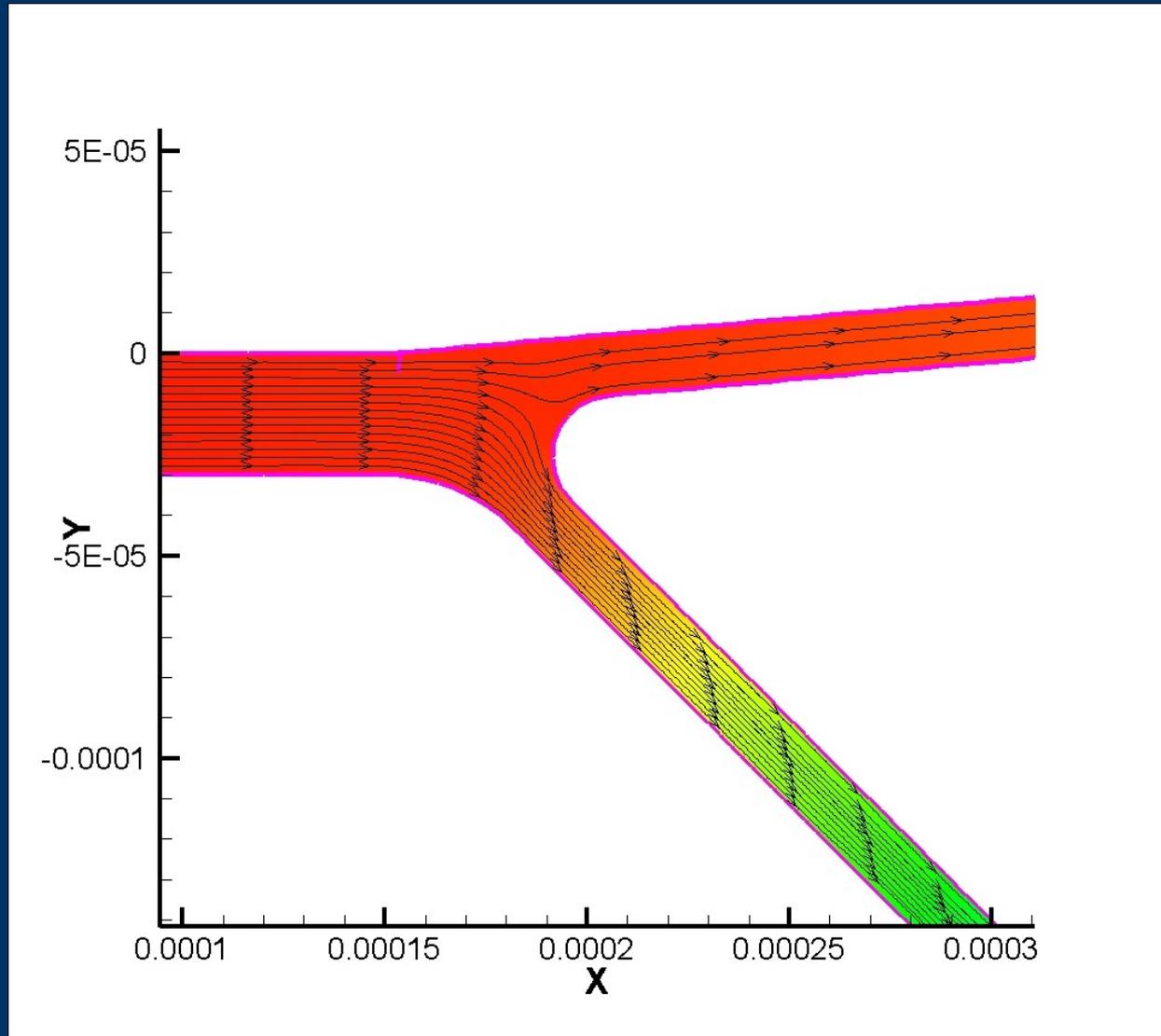
Create GAMBIT script to create geometries

Create input files for GAMBIT and FLUENT

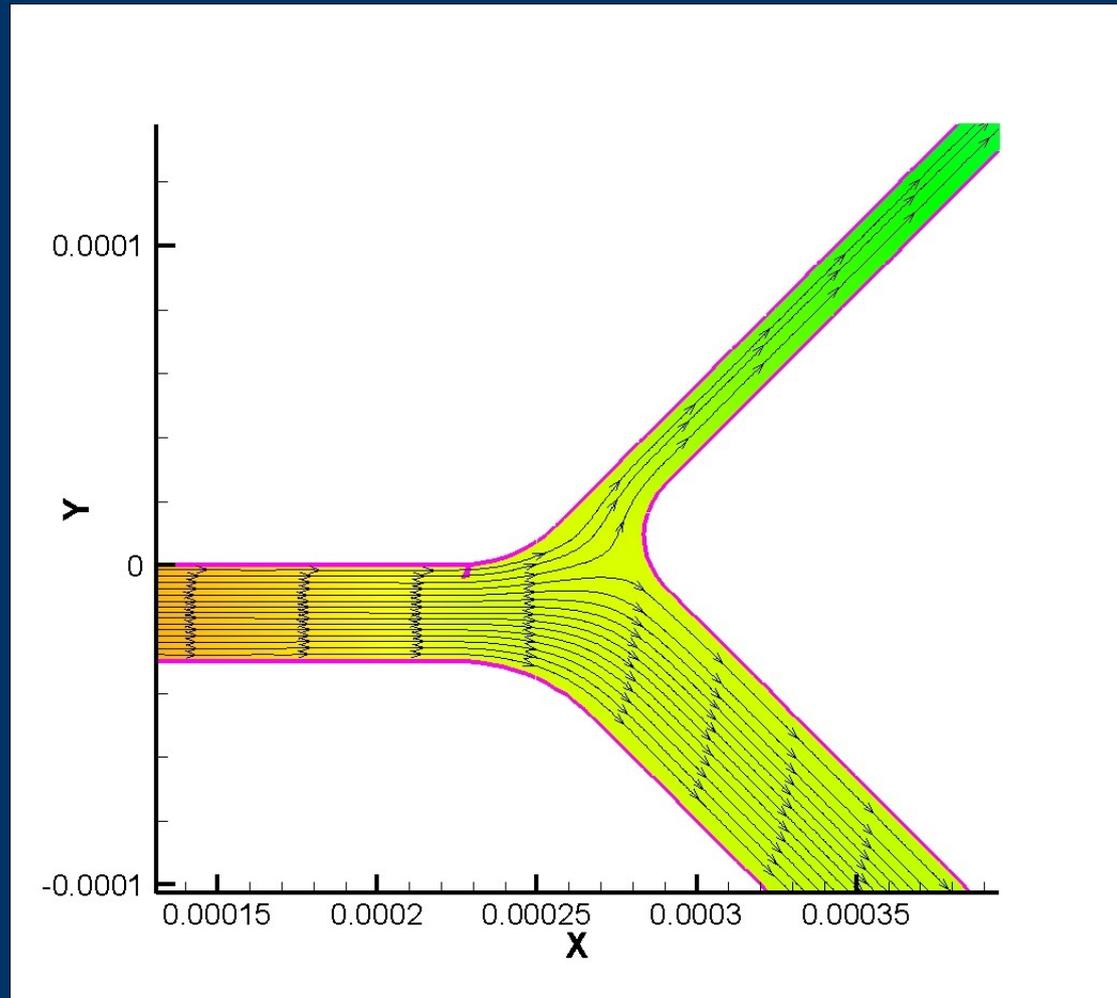
Read results files for static pressures and velocities averaged over surfaces in and out of junctions.

Fluent Result

$f2 = 0.1$, $d2/d1 = 0.5$, $\theta2 = 5^\circ$, $d3/d1 = 0.5$, and $\theta3 = 45^\circ$

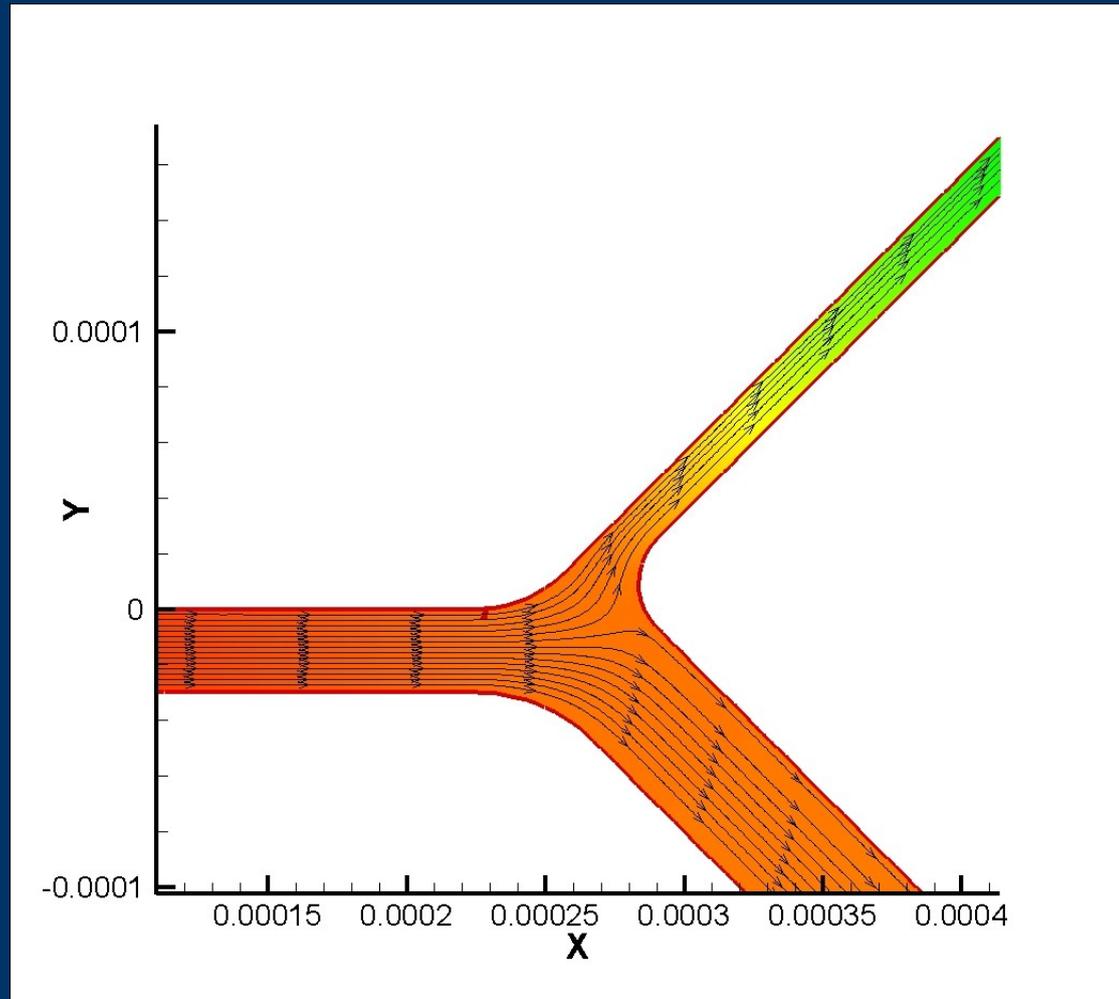


$f_2 = 0.1, \theta_2 = 45^\circ, \theta_3 = 45^\circ, d_2/d_1 = 0.5, d_3/d_1 = 1.5.$



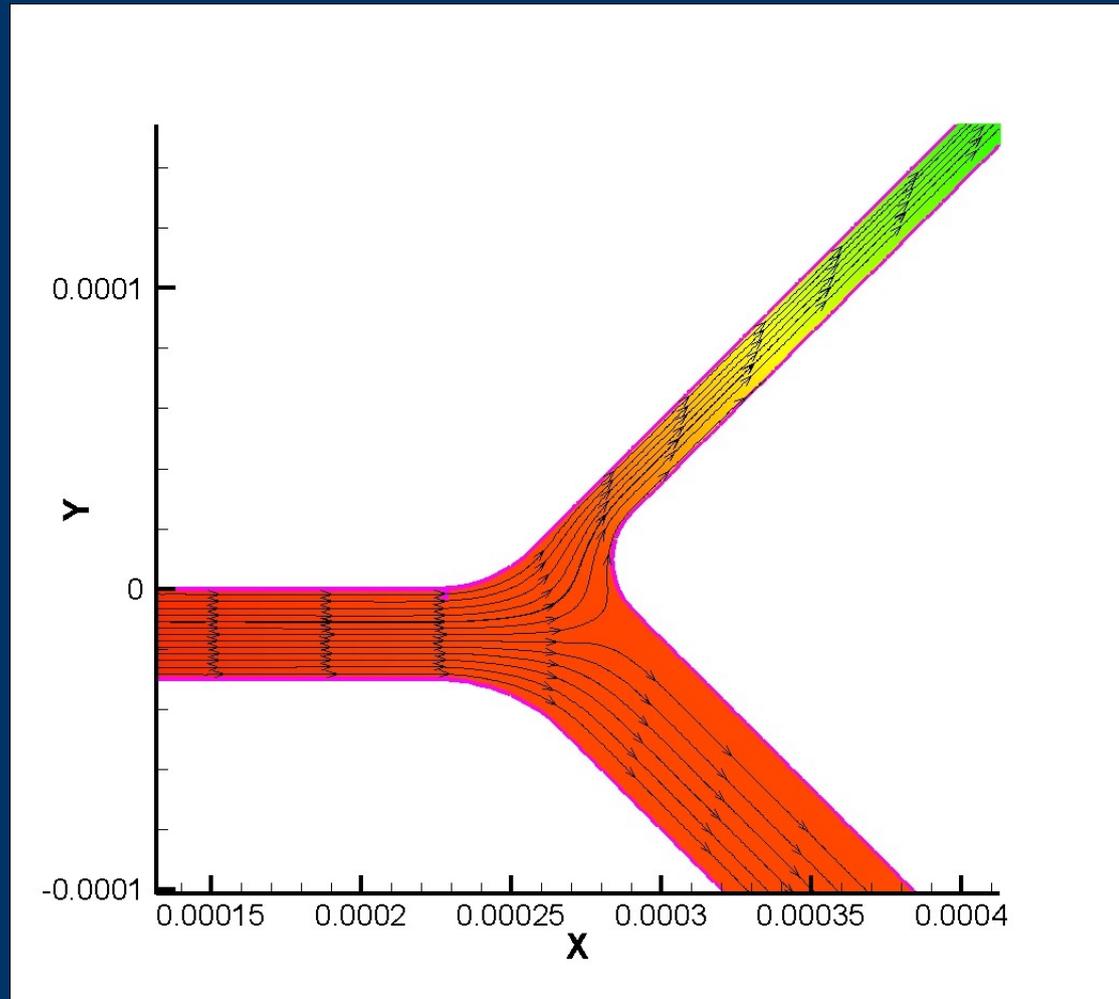
$$K_2 = 5.47$$

$f_2 = 0.3, \theta_2 = 65^\circ, \theta_3 = 45^\circ, d_2/d_1 = 0.5, d_3/d_1 = 1.5.$



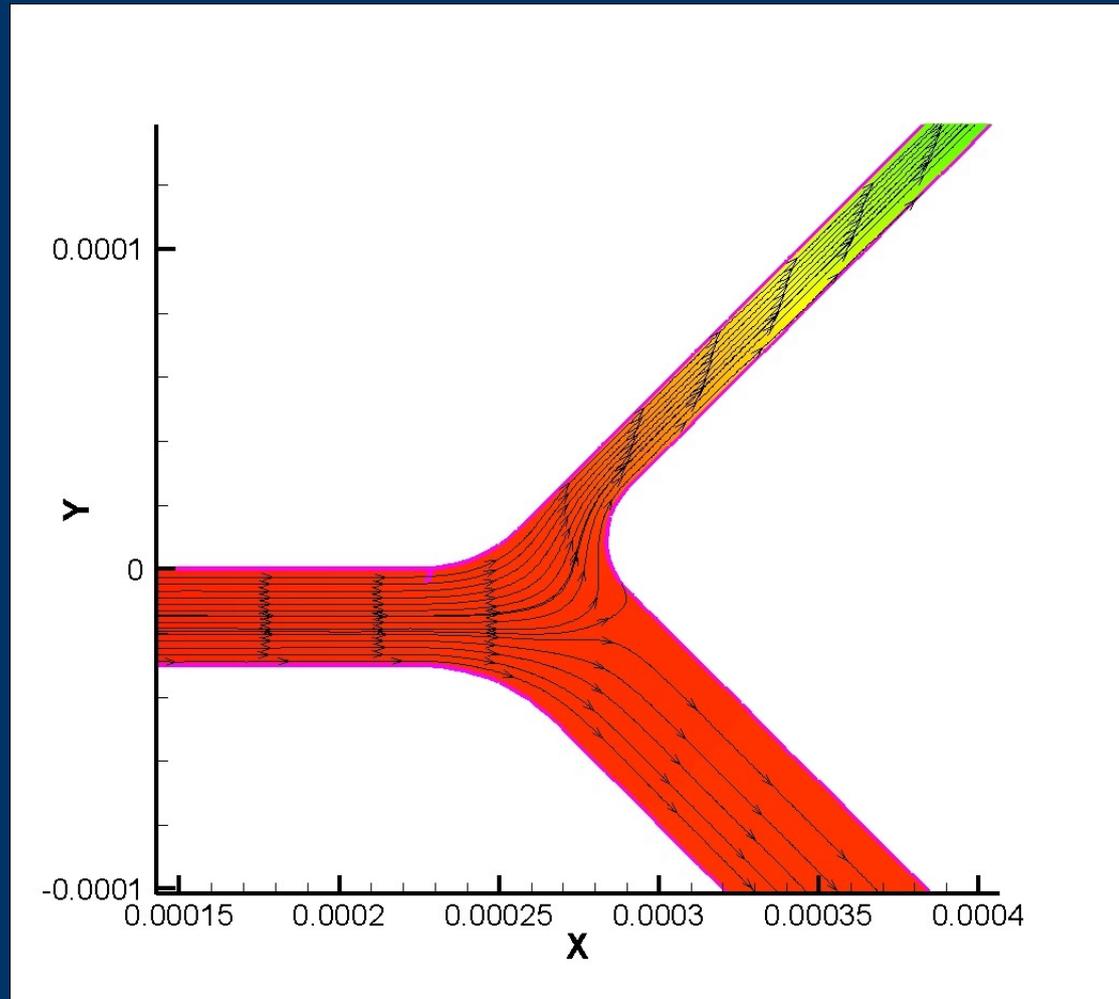
$K_2 = 11.6$

$f_2 = 0.5$, $\theta_2 = 65^\circ$, $\theta_3 = 45^\circ$, $d_2/d_1 = 0.5$, $d_3/d_1 = 1.5$.



$K_2 = 18.4$

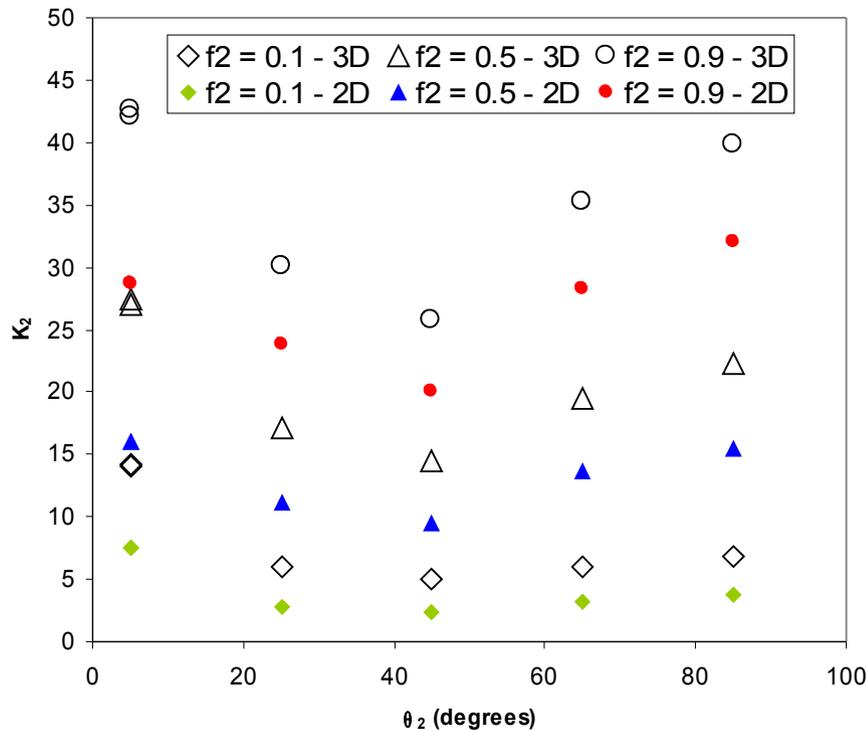
$f_2 = 0.7, \theta_2 = 65^\circ, \theta_3 = 45^\circ, d_2/d_1 = 0.5, d_3/d_1 = 1.5.$



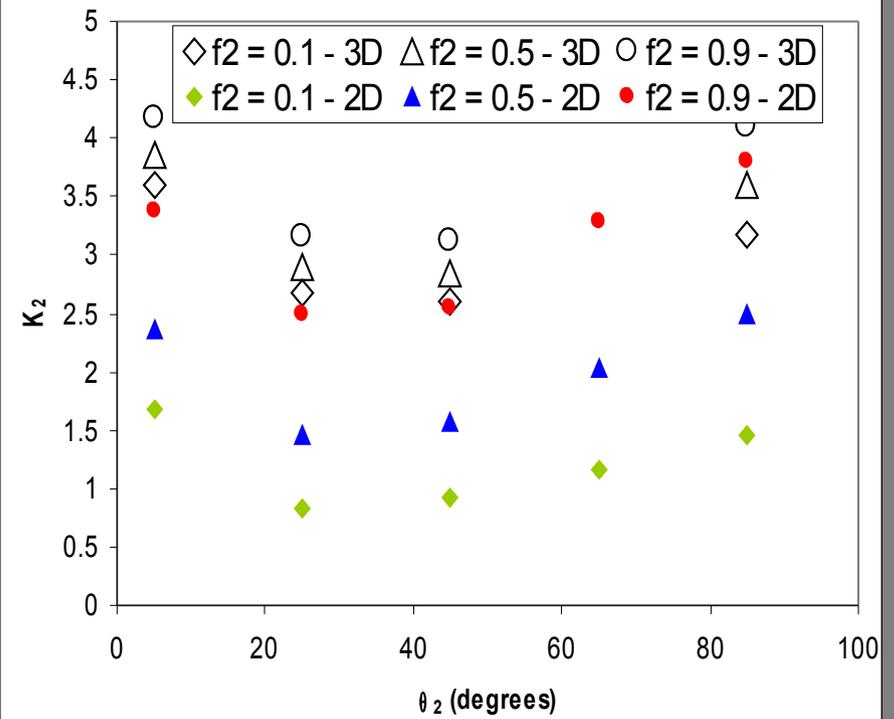
$K_2 = 25.7$

2D vs. 3D Differences

K_2 values for $d_2/d_1 = 0.5$, $d_3/d_1 = 0.5$, $\theta_3 = 5$ degrees



K_2 values for $d_2/d_1 = 0.5$, $d_3/d_1 = 1.5$, $\theta_3 = 45$ degrees



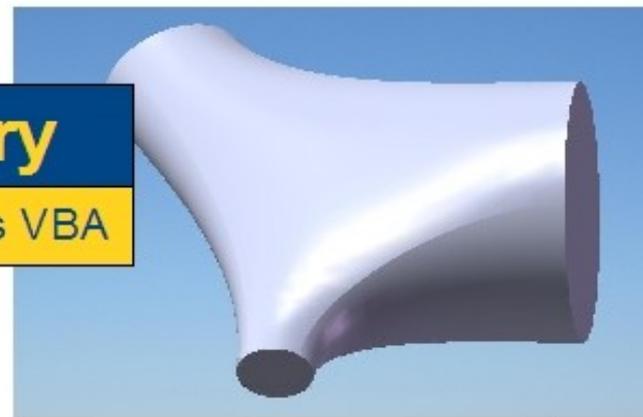
$$\left[\frac{k_2^{3D}}{k_2^{2D}} \right]_{average} = 1.55 \quad (std. \ dev.) = 0.45$$

$$k_2 = \frac{\Delta p / \rho}{u_1^2 / 2} - \frac{u_2^2}{u_1^2} + 1$$

pressure effects kinetic energy effects

Generate Junction Geometry

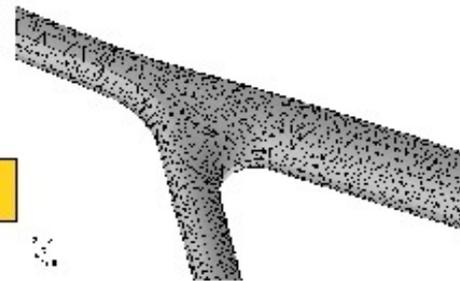
Junction code (C++) → SolidWorks VBA



Simulations

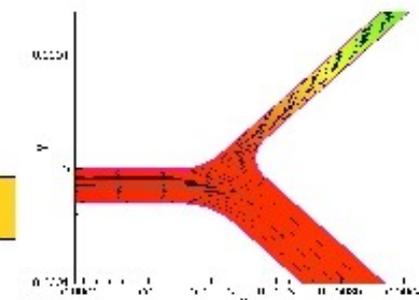
Generate Mesh

Junction Code → GAMBIT



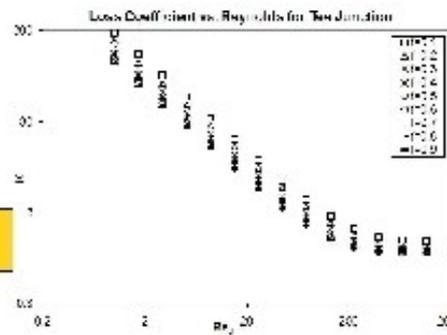
Simulate Flow

Junction Code → FLUENT



Analyze Results

Junction Code → tsv

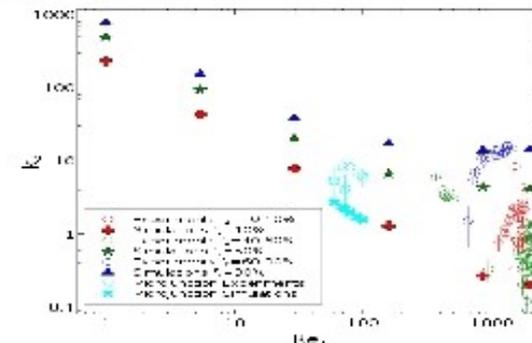
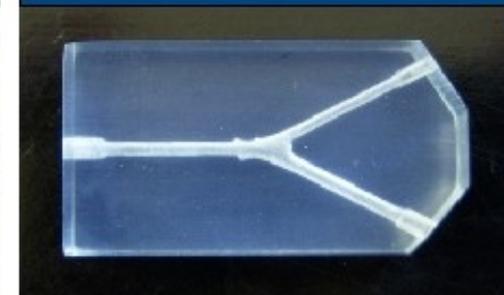


Experiments

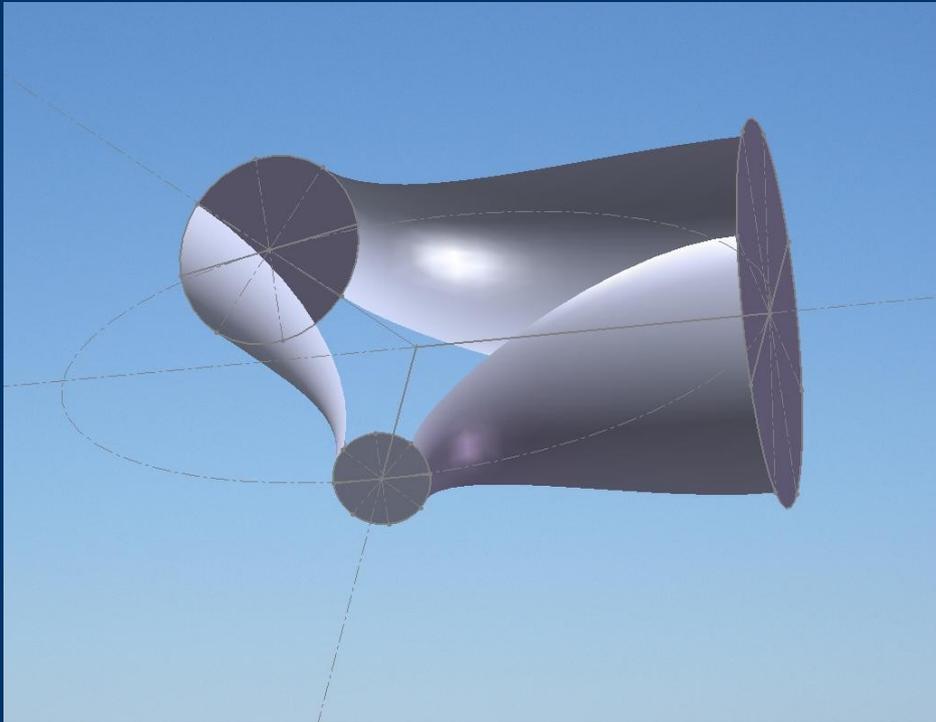
Milliscale



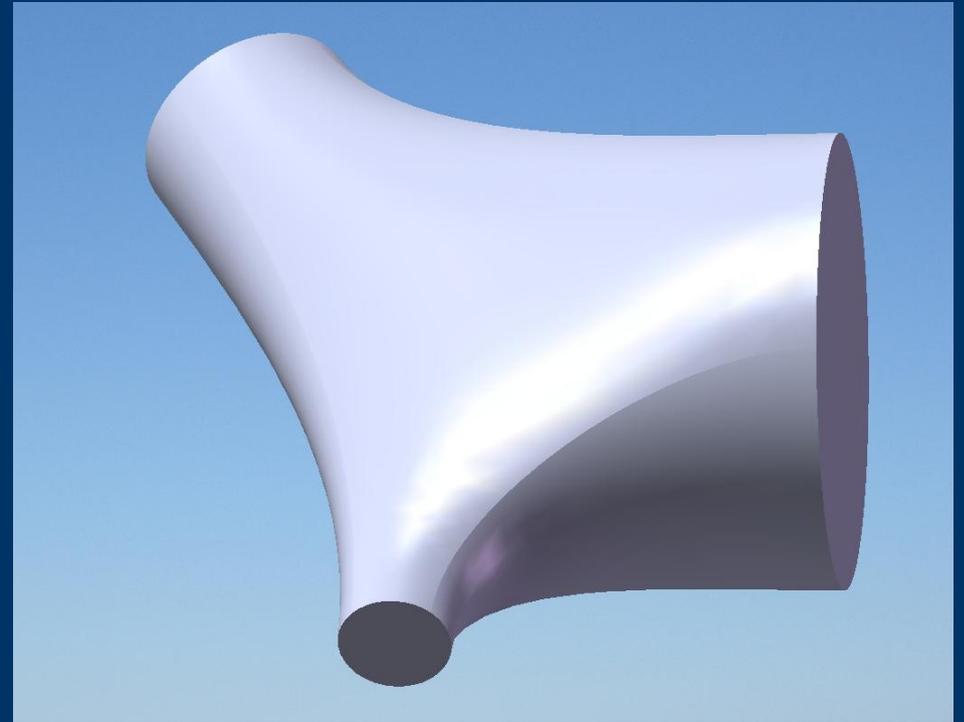
Microscale



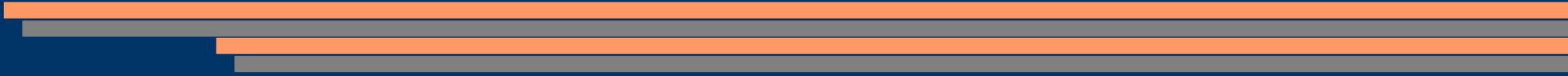
3D Mark II Junction



Junction Without Surface Fills



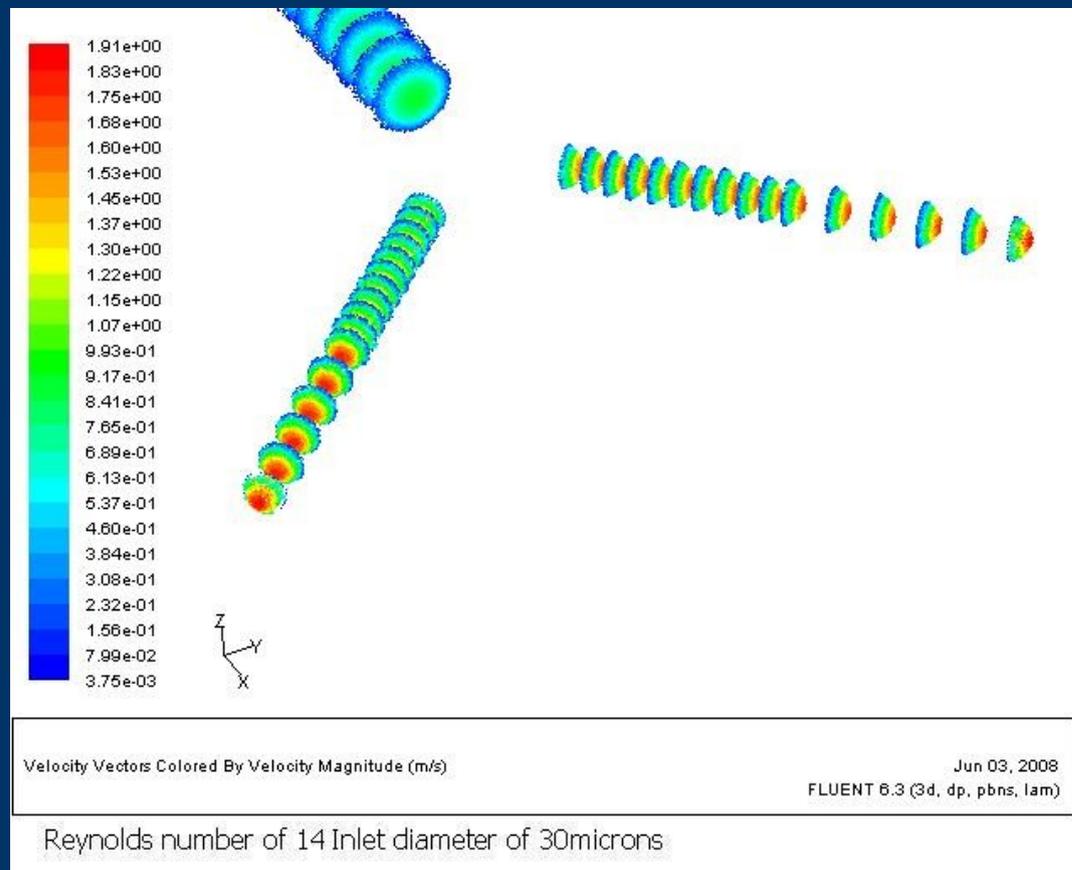
Junction With Surface Fills

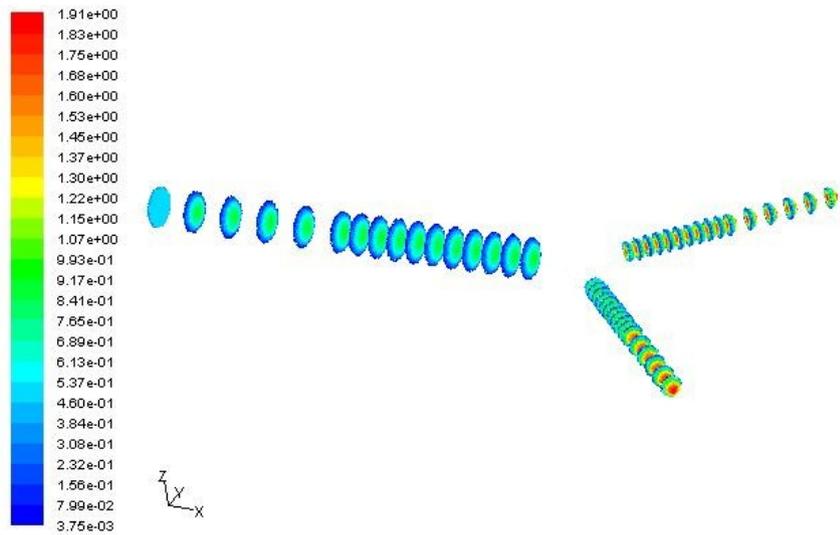


Initial Simulations – 3D Mark II

Small batch

Comparable results to original 3D runs

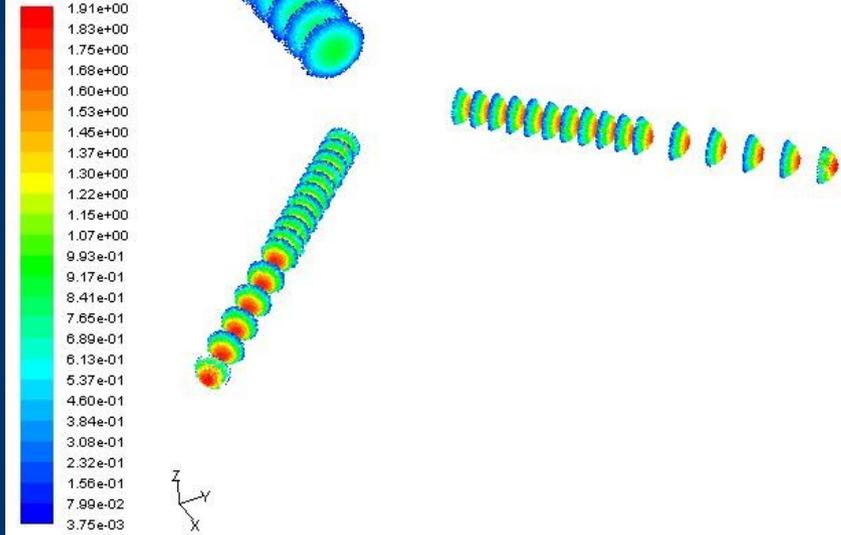




Velocity Vectors Colored By Velocity Magnitude (m/s)

Jun 03, 2008
FLUENT 6.3 (3d, dp, pbns, lam)

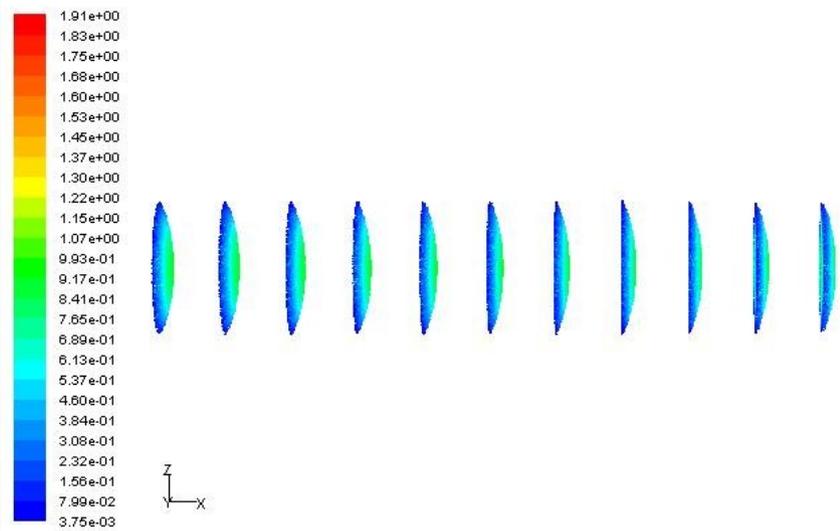
Reynolds number of 14 Inlet diameter of 30microns



Velocity Vectors Colored By Velocity Magnitude (m/s)

Jun 03, 2008
FLUENT 6.3 (3d, dp, pbns, lam)

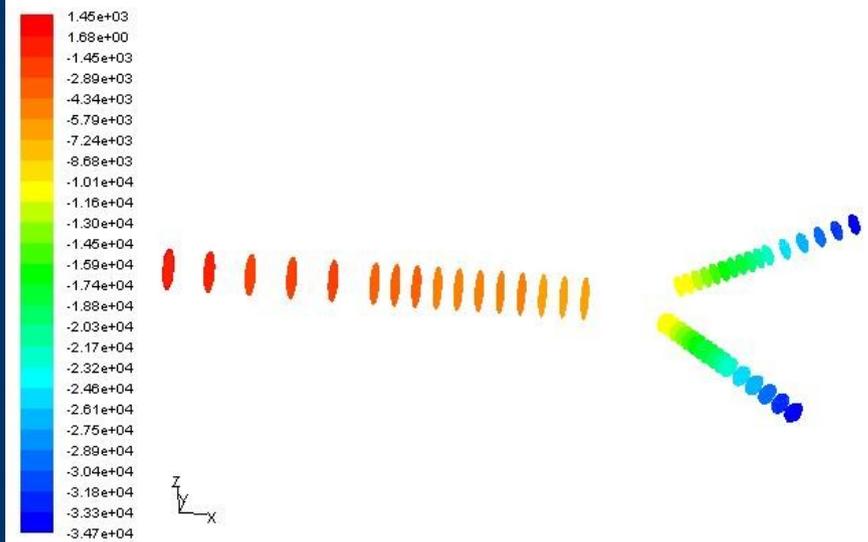
Reynolds number of 14 Inlet diameter of 30microns



Velocity Vectors Colored By Velocity Magnitude (m/s)

Jun 03, 2008
FLUENT 6.3 (3d, dp, pbns, lam)

Reynolds number of 14 Inlet diameter of 30microns



Contours of Static Pressure (pascal)

Jun 03, 2008
FLUENT 6.3 (3d, dp, pbns, lam)

Reynolds number of 14 Inlet diameter of 30microns

Pipe Mesh Testing

Database Interface

Pipe Results Database

Sever | Preview and Download | Open Runs

Flow Parameters

Reynolds	Diameters	Lengths
25	3	30
50		60
100		150
200		300
500		450
1000		600

Edgemesh Parameters

Type	Ratio1	Spacing Type	Spacing
successive bellshape	1 0.8	0	10 20 50 100 30 40

Facemesh Parameters

Type	Spacing Type	Spacing
map	0	20 50 70 90 40 120

Volumemesh Parameters

Type	Spacing Type	Spacing
cooper	1	2

Get Valid Runs

Valid Runs

	Reynolds	Diameter	Length	Edge Params	Face Params	Volume Params
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 80	map 0 90	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 80	map 0 70	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 80	map 0 50	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 60	map 0 90	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 60	map 0 70	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 60	map 0 50	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 50	map 0 90	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 50	map 0 70	cooper 1 2
<input type="checkbox"/>	500	3	150	bellshape 0.8 0 50	map 0 50	cooper 1 2

Get Data Plot Preview

Rakes (Reynolds = 500)

Pipe Mesh Testing

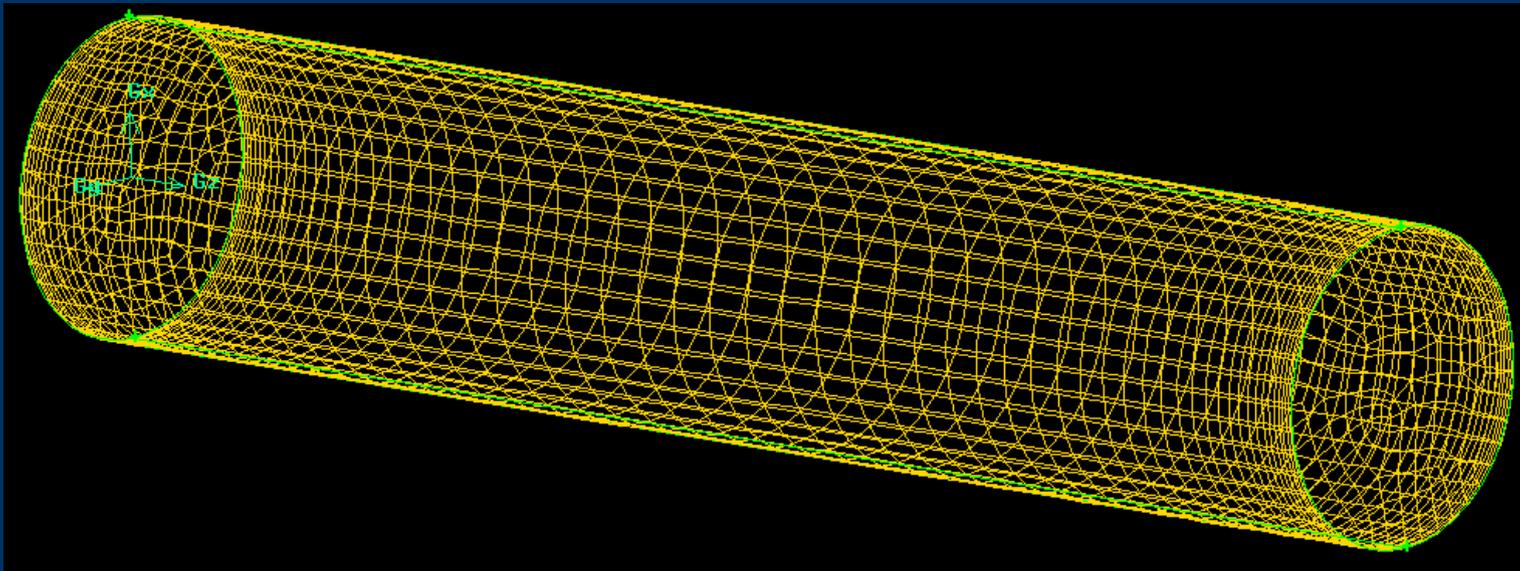
Flow-aligned hex core

Varied axial and radial spacing

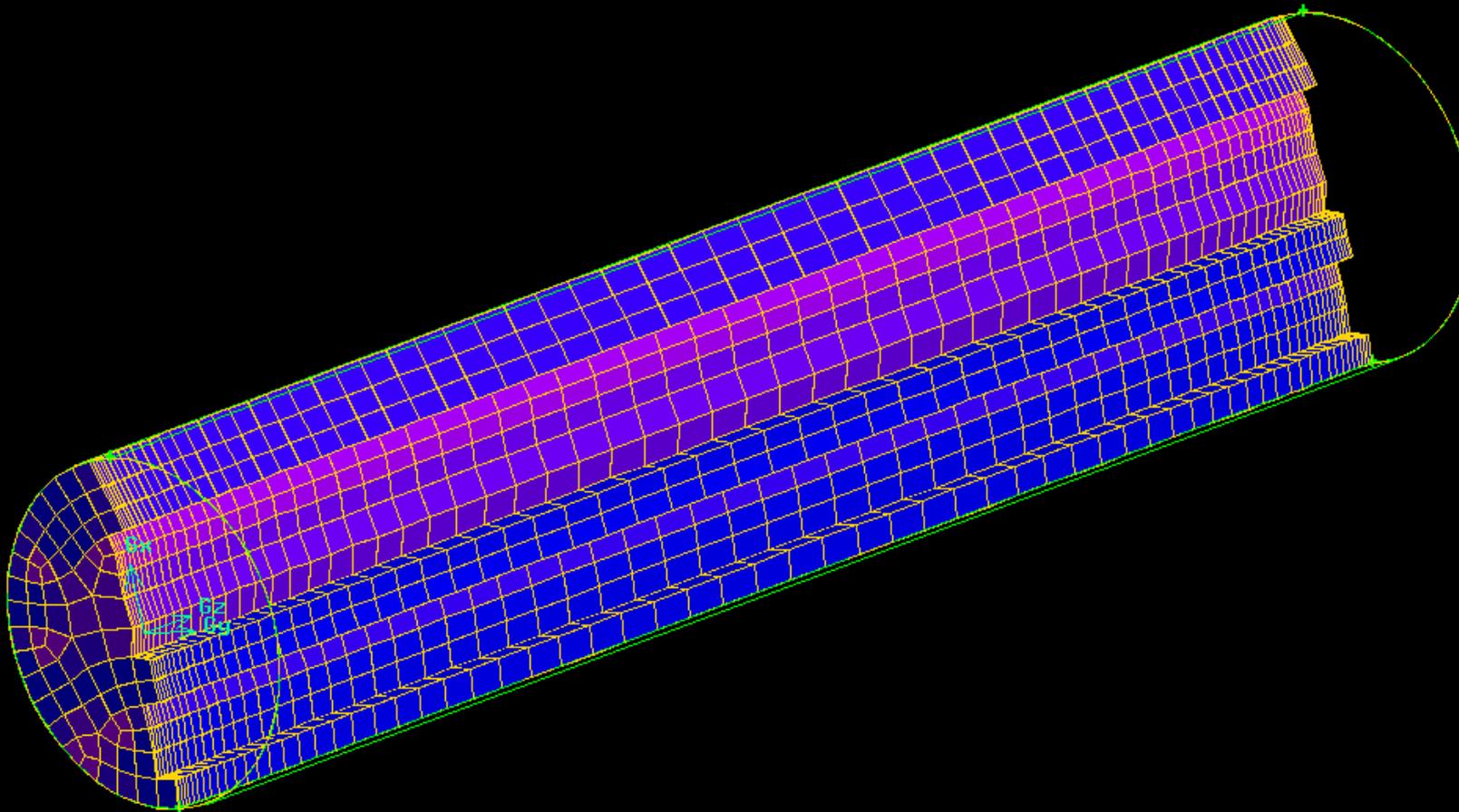
MySQL results database

Automated

Results database easy to setup for junction runs

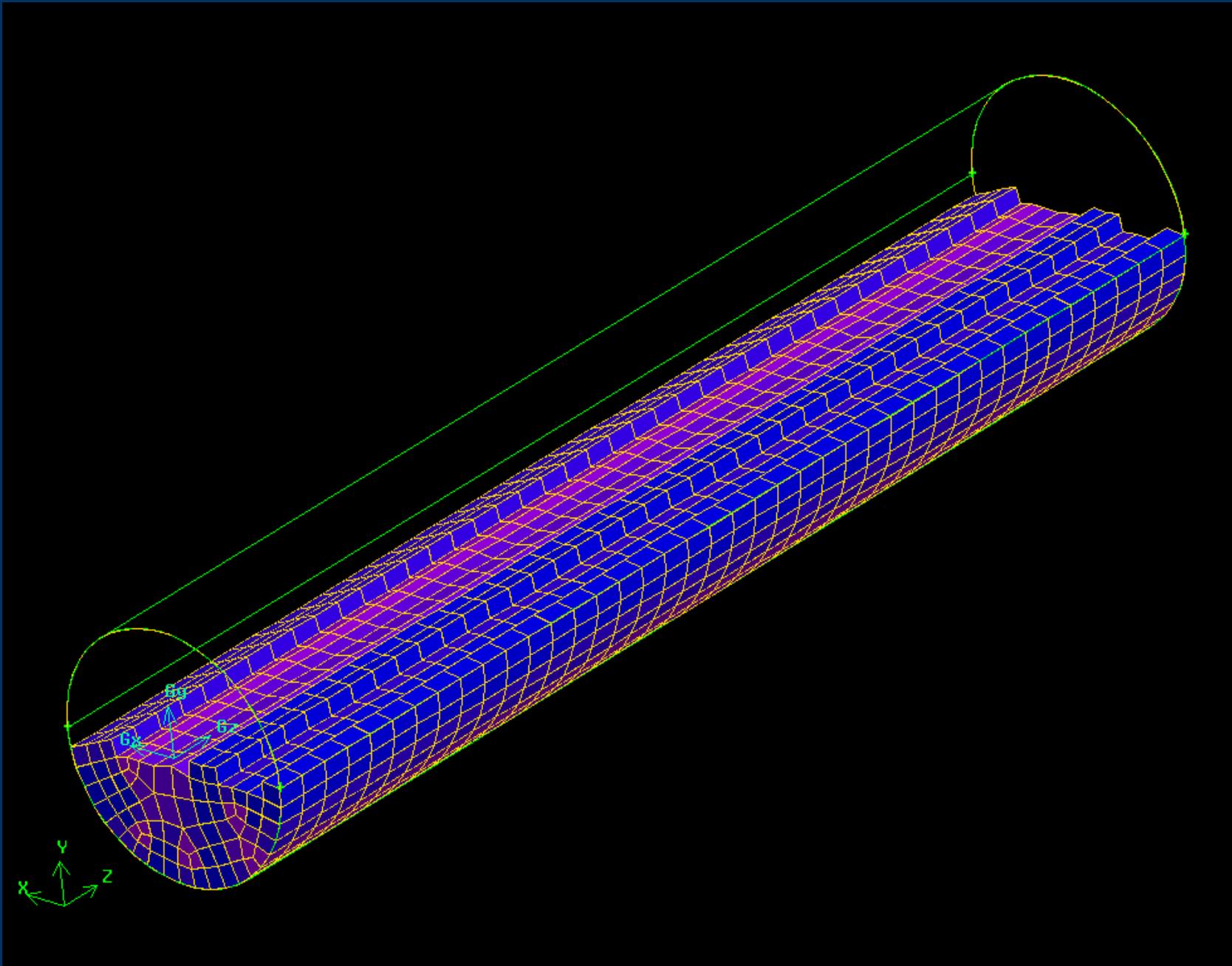


Pipe Mesh Testing



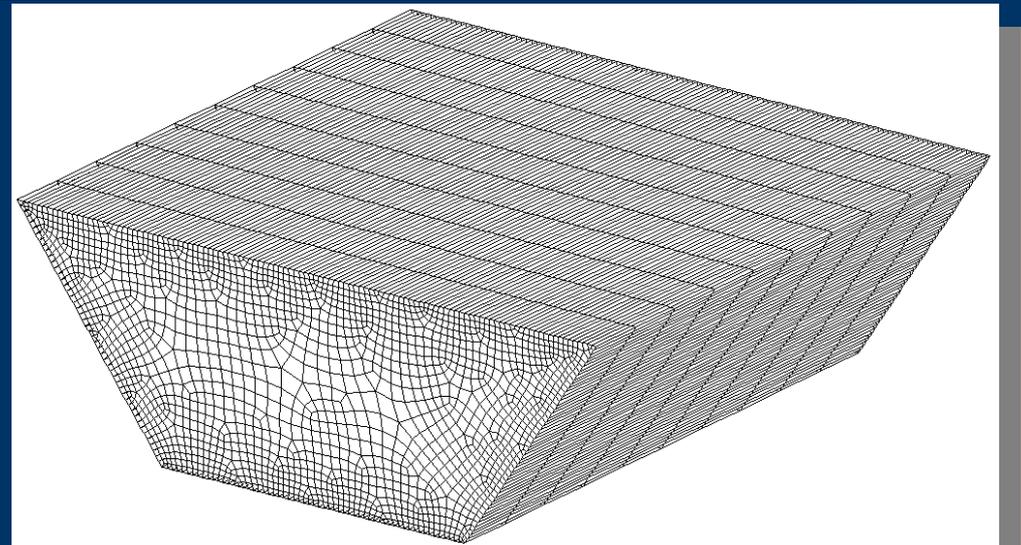
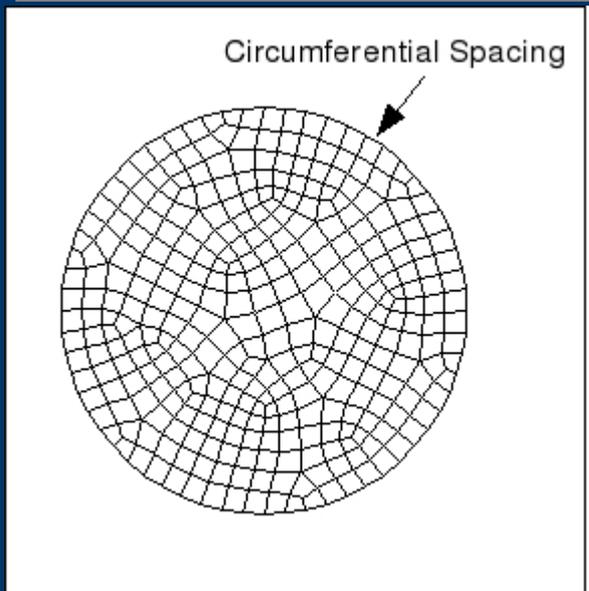
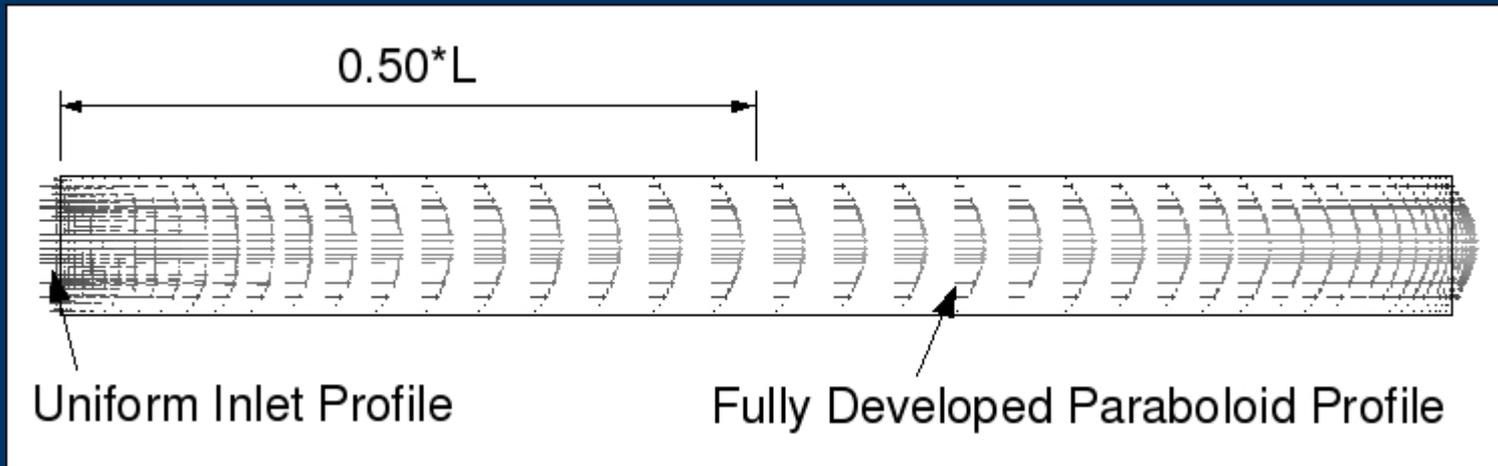
“Bell Shaped” Axial Spacing

Pipe Mesh Testing

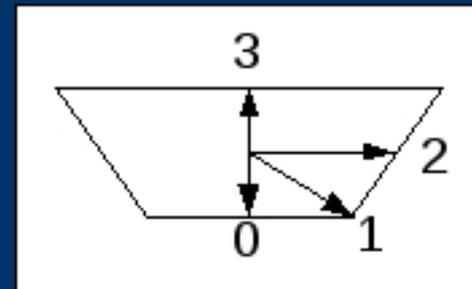
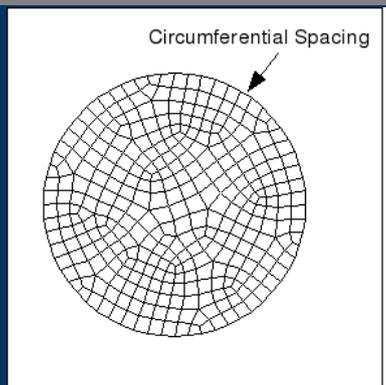
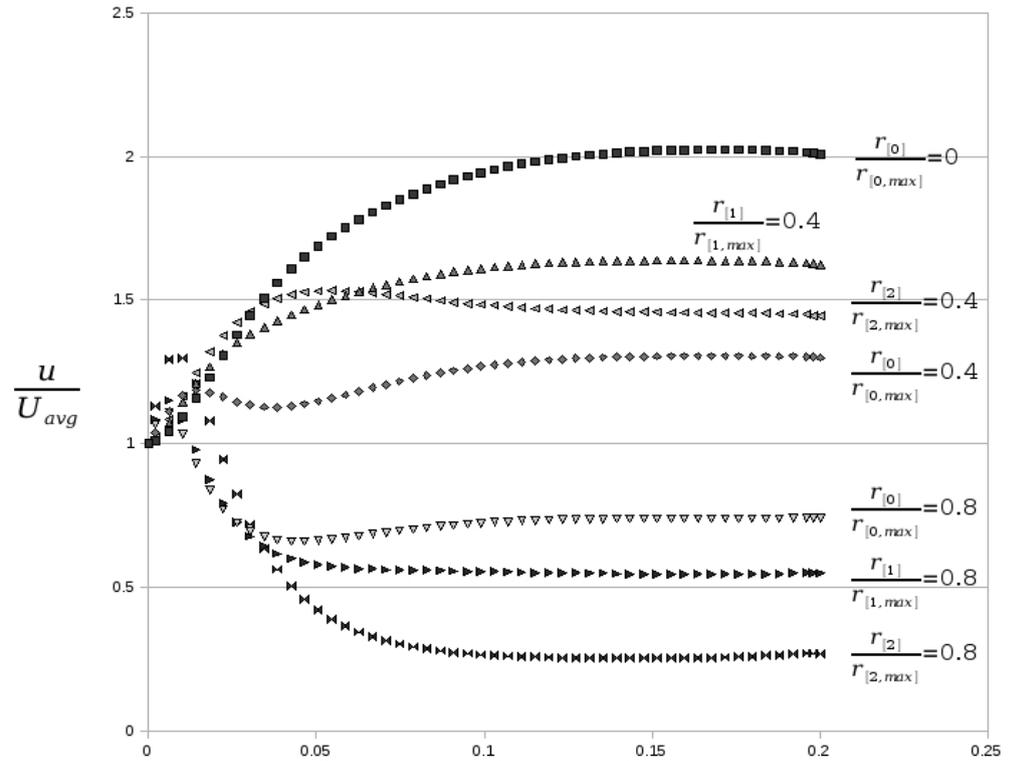
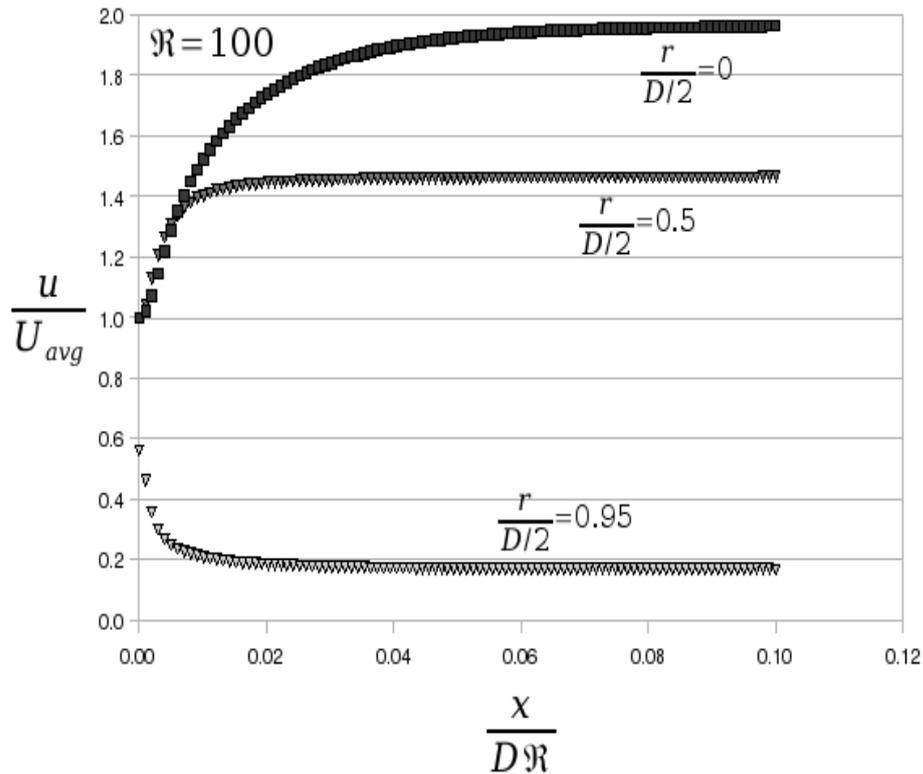


Uniform Axial Spacing

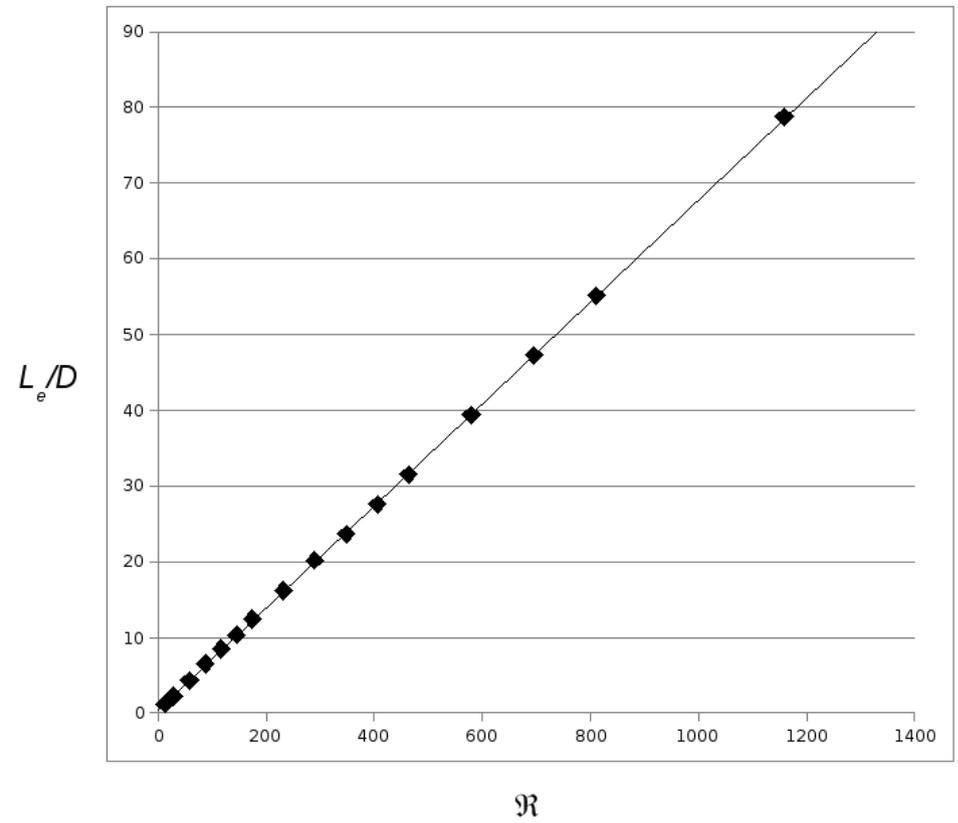
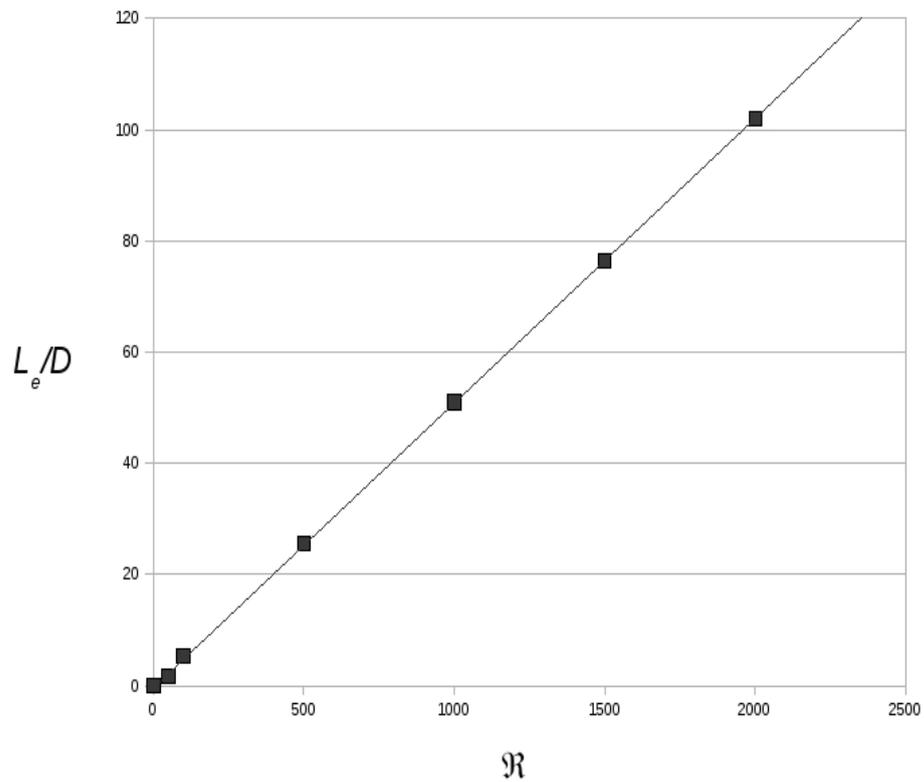
Entrance Length in Microtubes



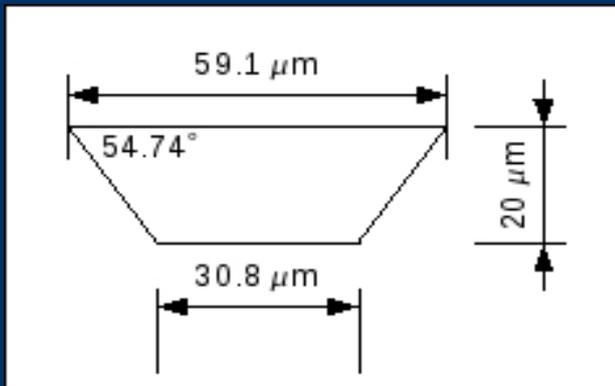
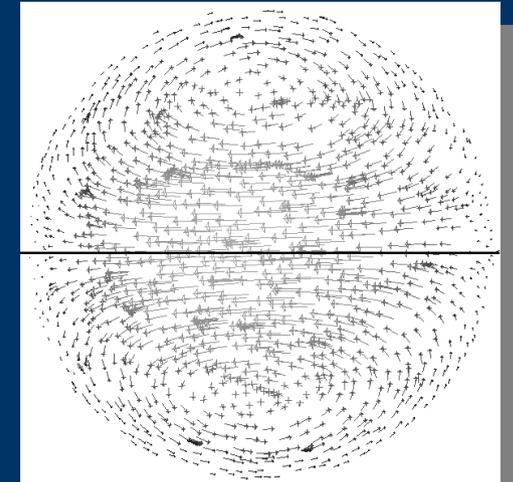
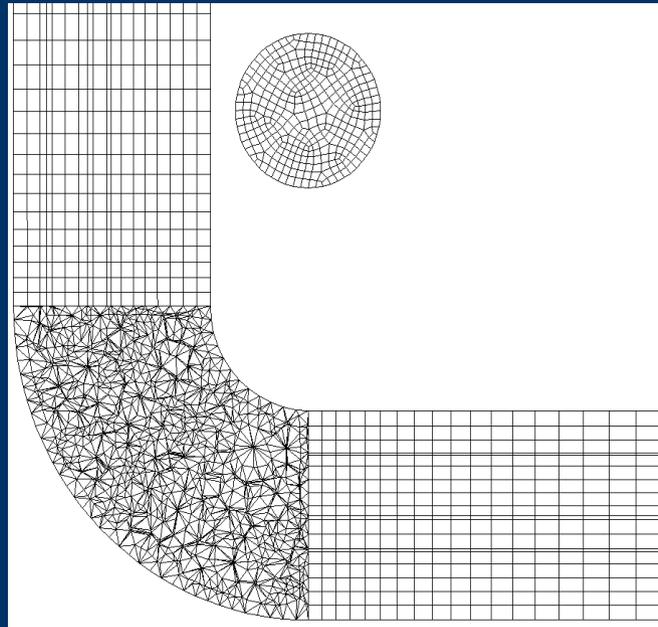
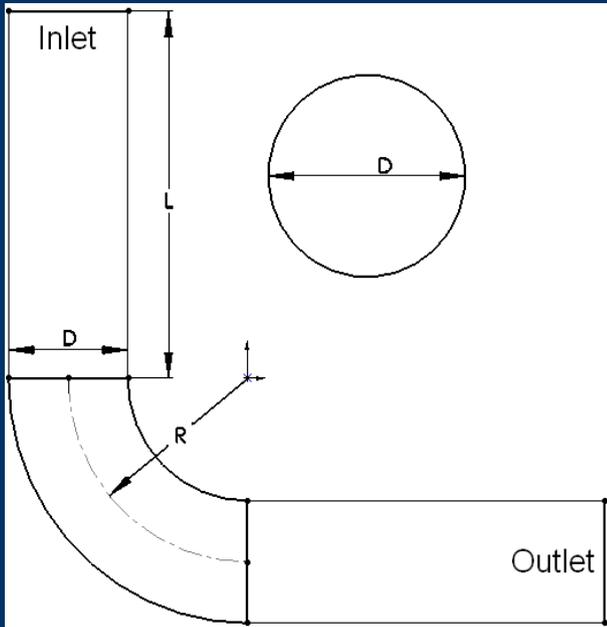
Entrance Length Results



Entrance Length Results



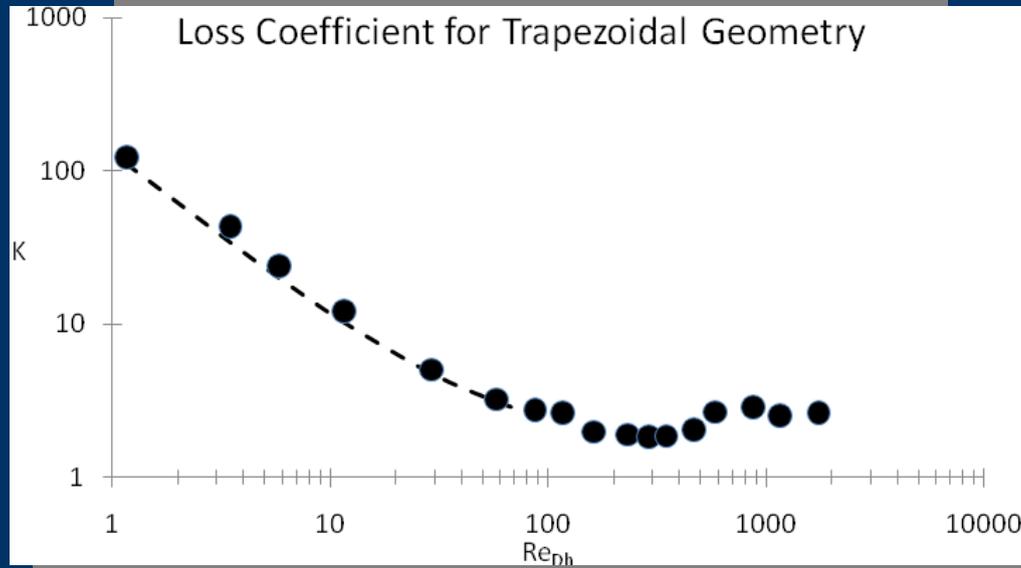
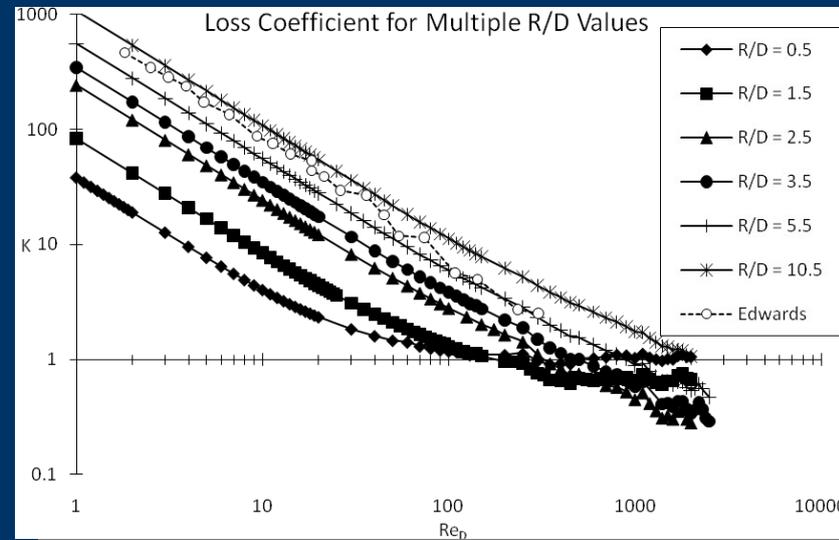
Loss Coefficients in Microelbows



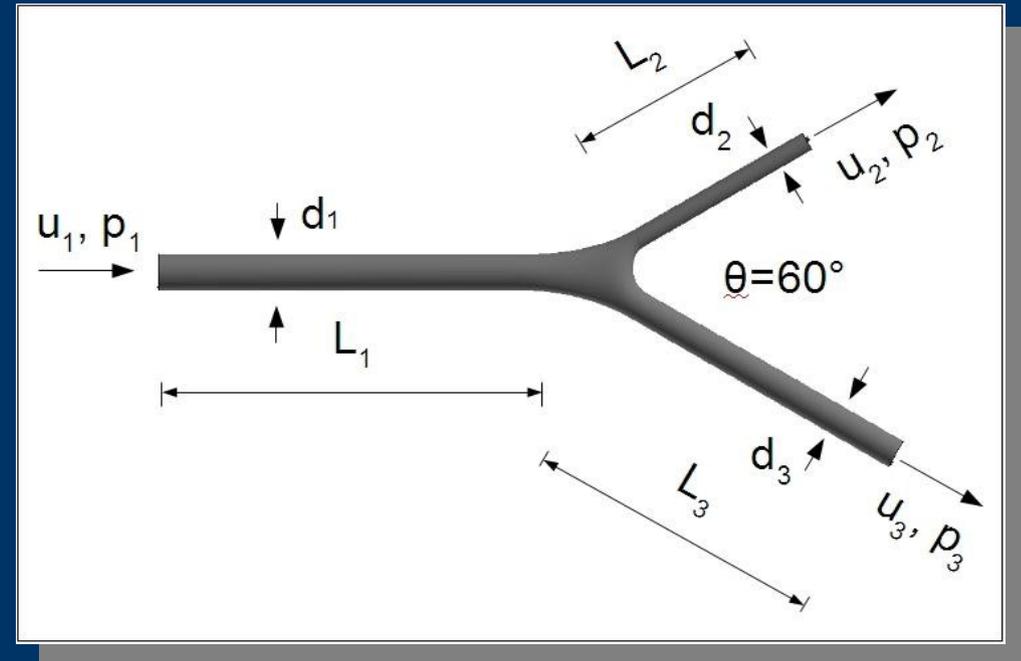
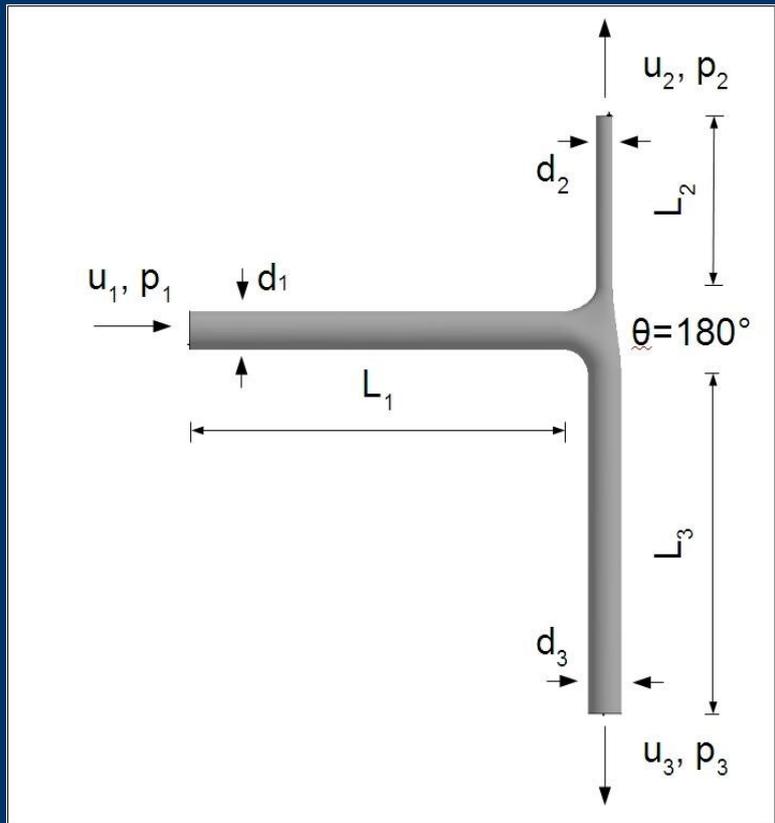
$$K = \frac{\Delta p}{\frac{1}{2} \rho V^2}$$

Microelbow Loss Coefficients

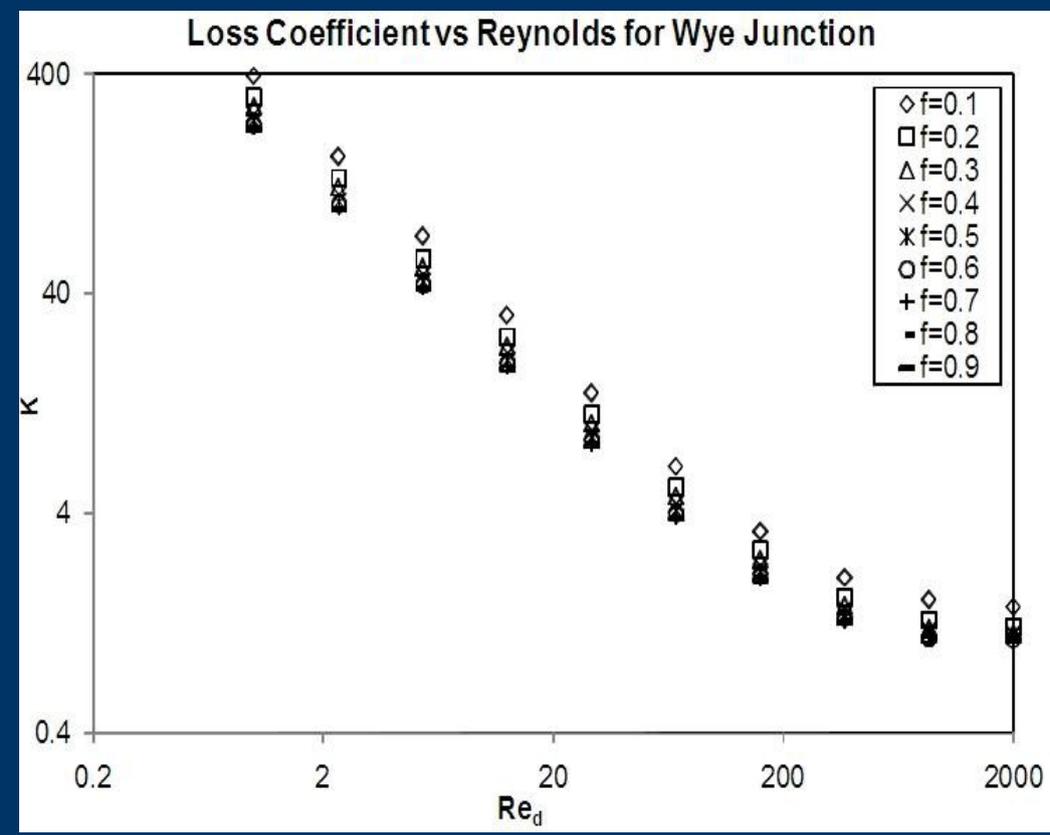
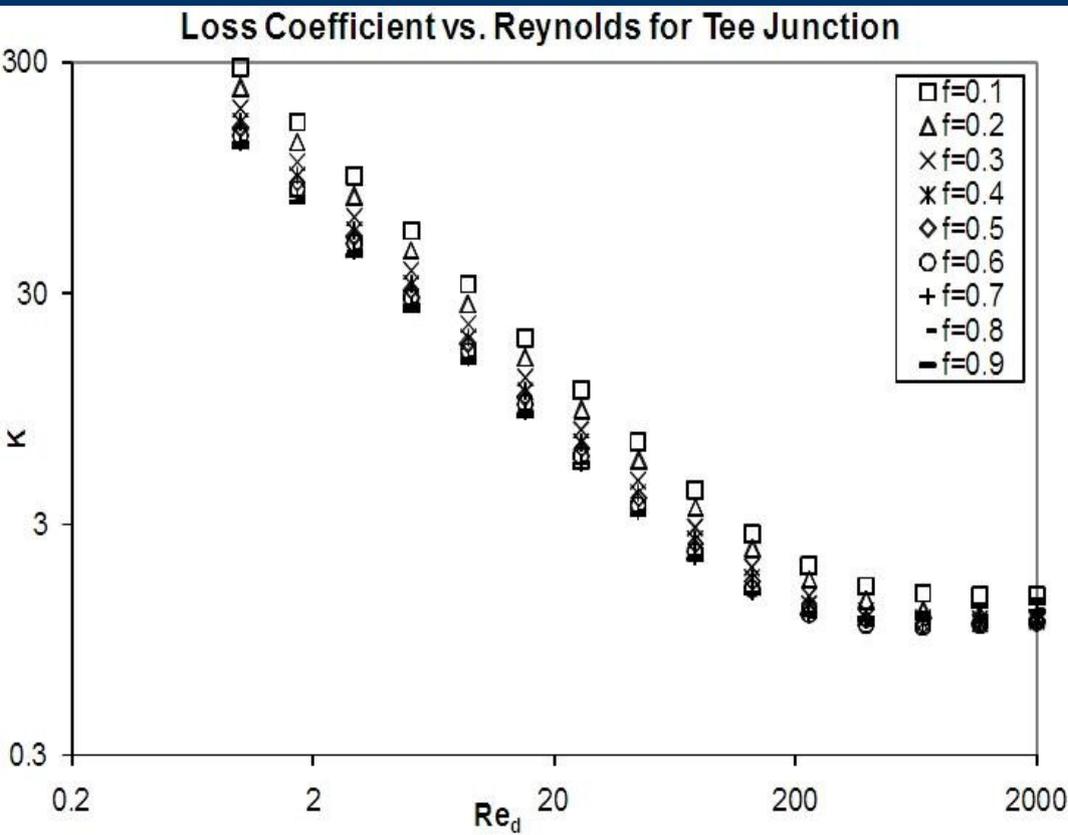
$$K = \frac{\Delta p}{\frac{1}{2} \rho V^2}$$



Tee and Wye Reynolds Number Dependence

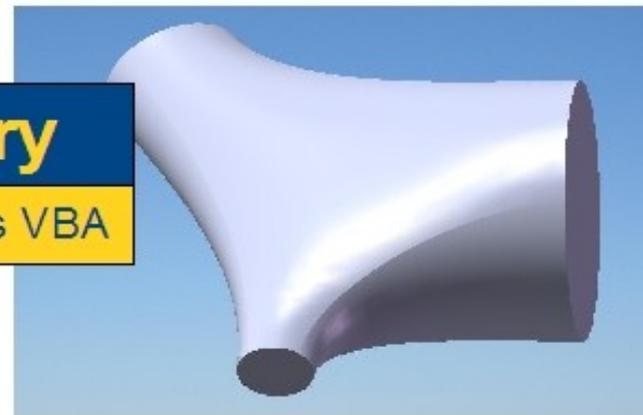


Loss Coefficient Versus $Re =$ *proportional to speed*



Generate Junction Geometry

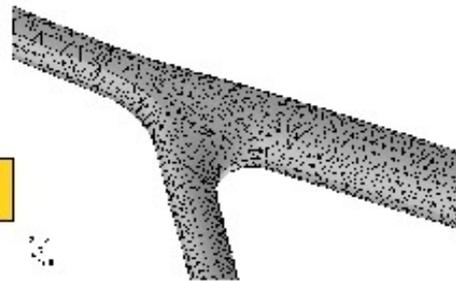
Junction code (C++) → SolidWorks VBA



Simulations

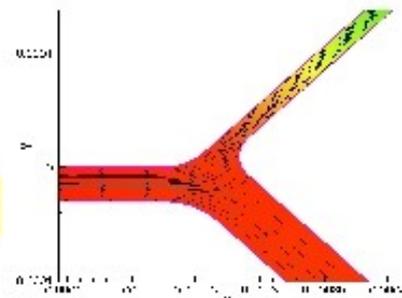
Generate Mesh

Junction Code → GAMBIT



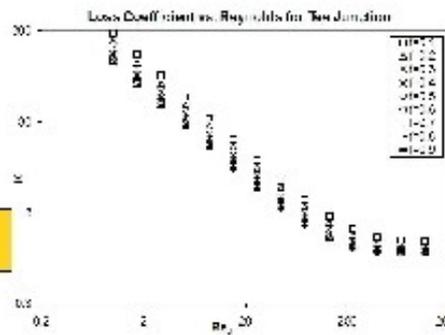
Simulate Flow

Junction Code → FLUENT



Analyze Results

Junction Code → tsv

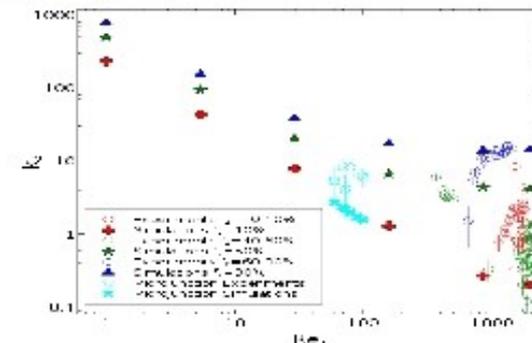
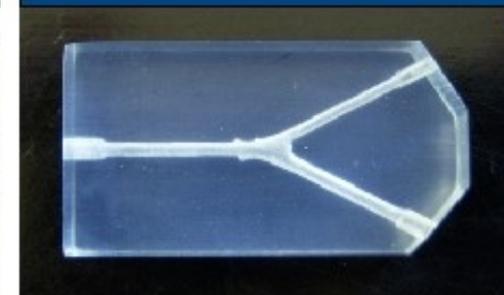


Experiments

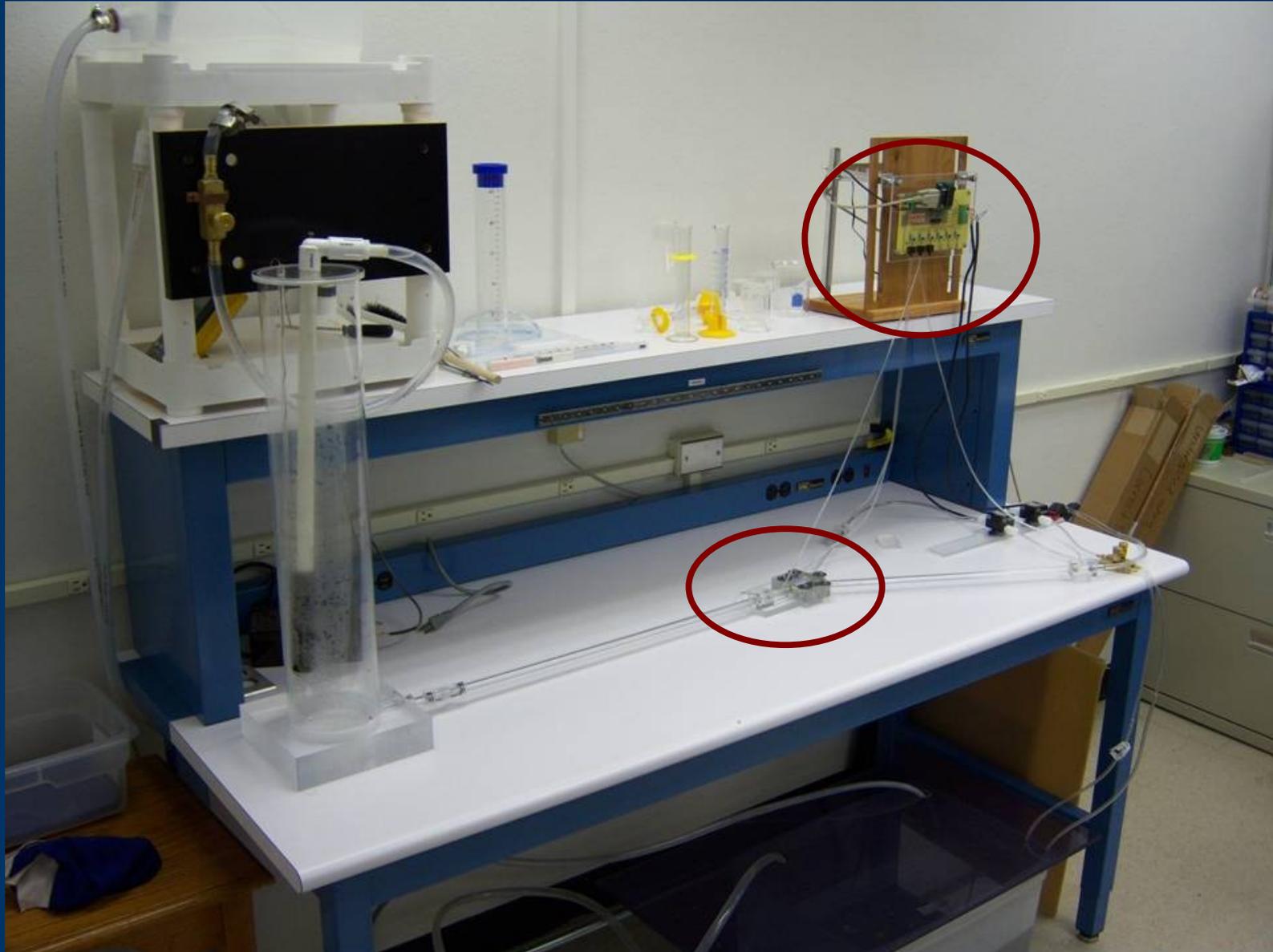
Milliscale



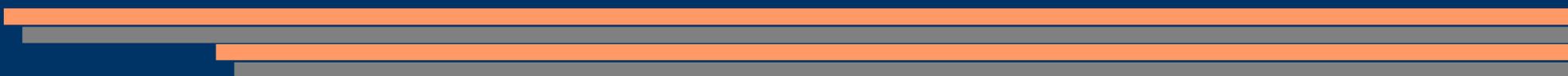
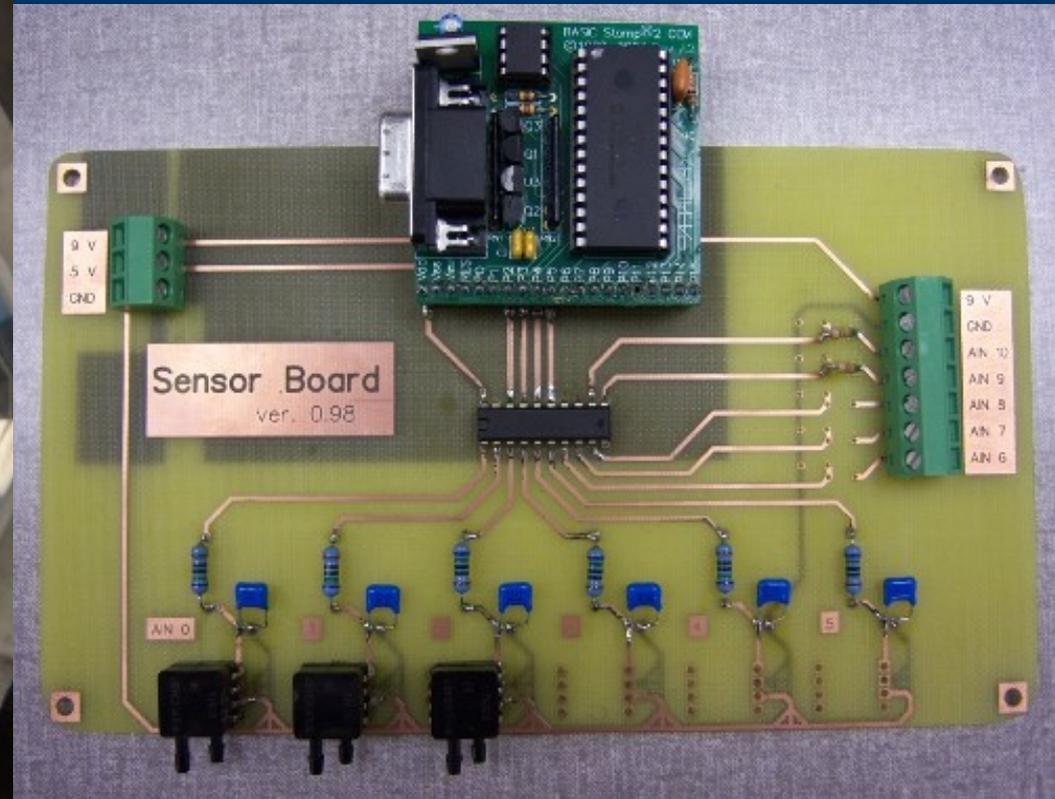
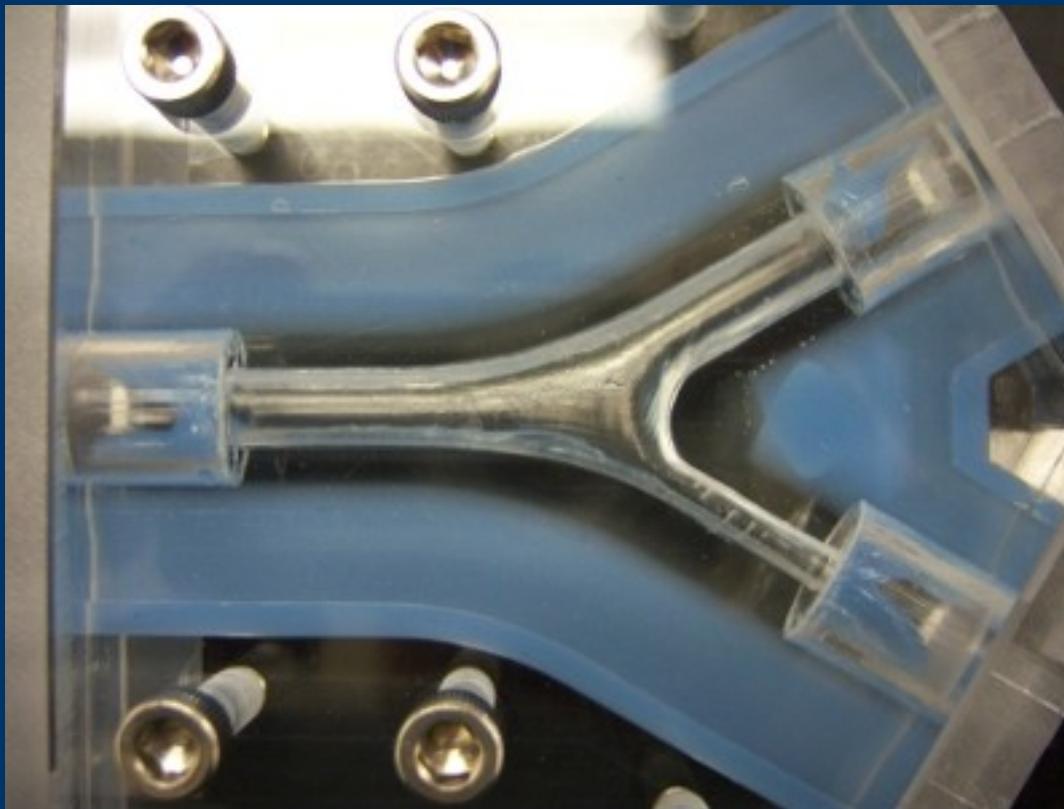
Microscale

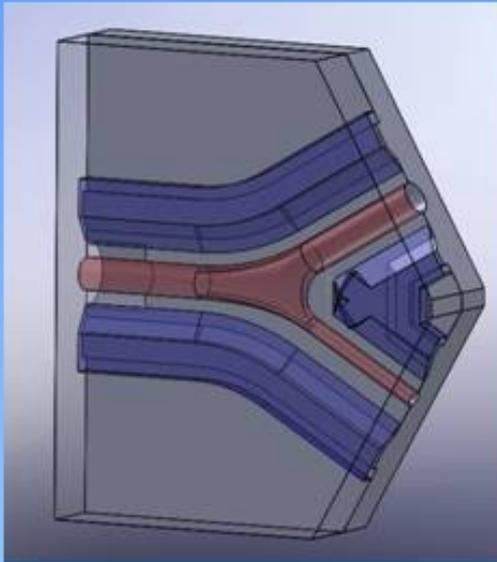


Millijunction Experiments

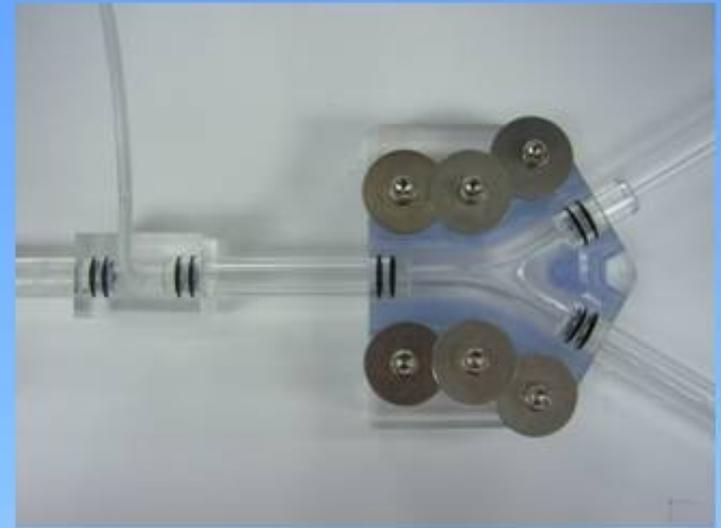


Junction and Measurements





Here is the Solidworks model as well as the physical realization of our efforts to create a useable bifurcation.



Upstream from the head tank and downstream from the junction are our new, highly sensitive needle valves used to adjust the head tank flow and the flow fraction through the bifurcation.



The flow straightener we designed has a replaceable core made of glass micropipettes that can be removed completely for unrestricted flow.



Collecting Data

Sensor

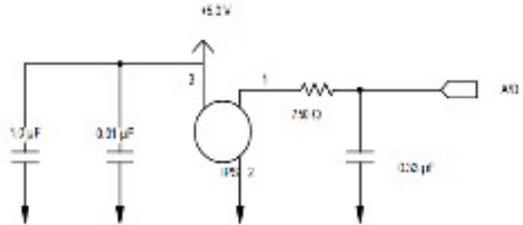
- Submersible
- Internal Instrumentation Op-Amp



MPVZ5004GW7U
CASE 1560-02

Filter

- 750Hz Low Pass
- Reduces Noise from 10mV to 2mV



ADC

- 12 bit
- 11 Channel



Basic Stamp

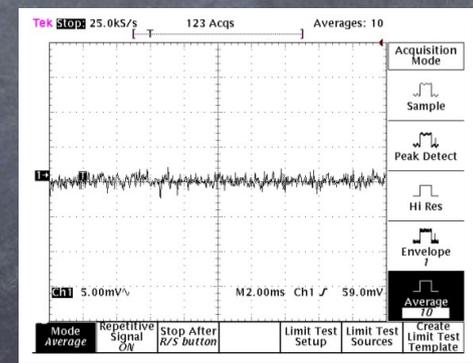
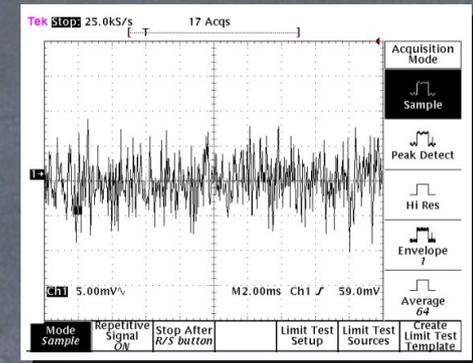
- Easy to program
- Affordable



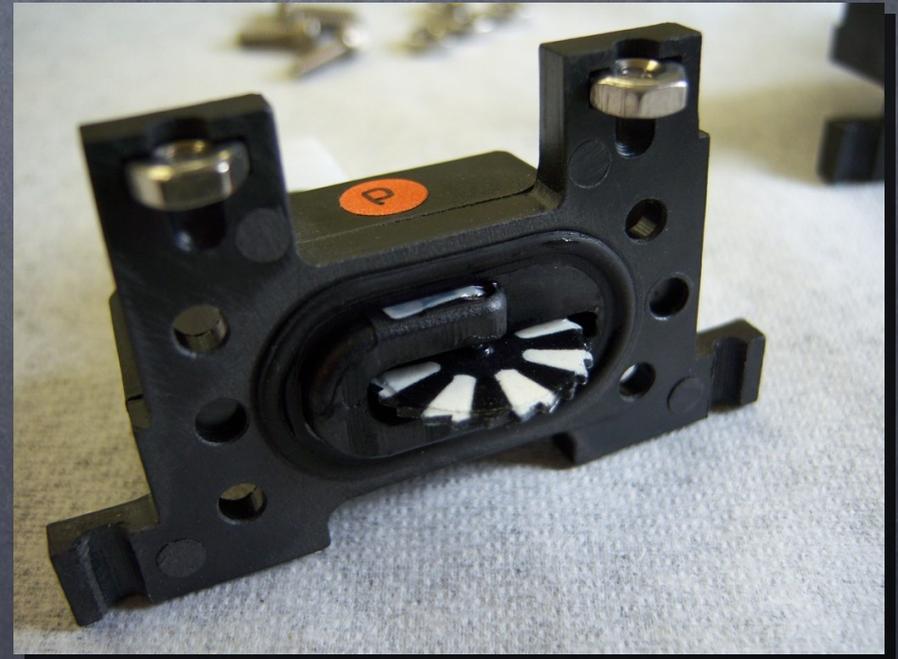
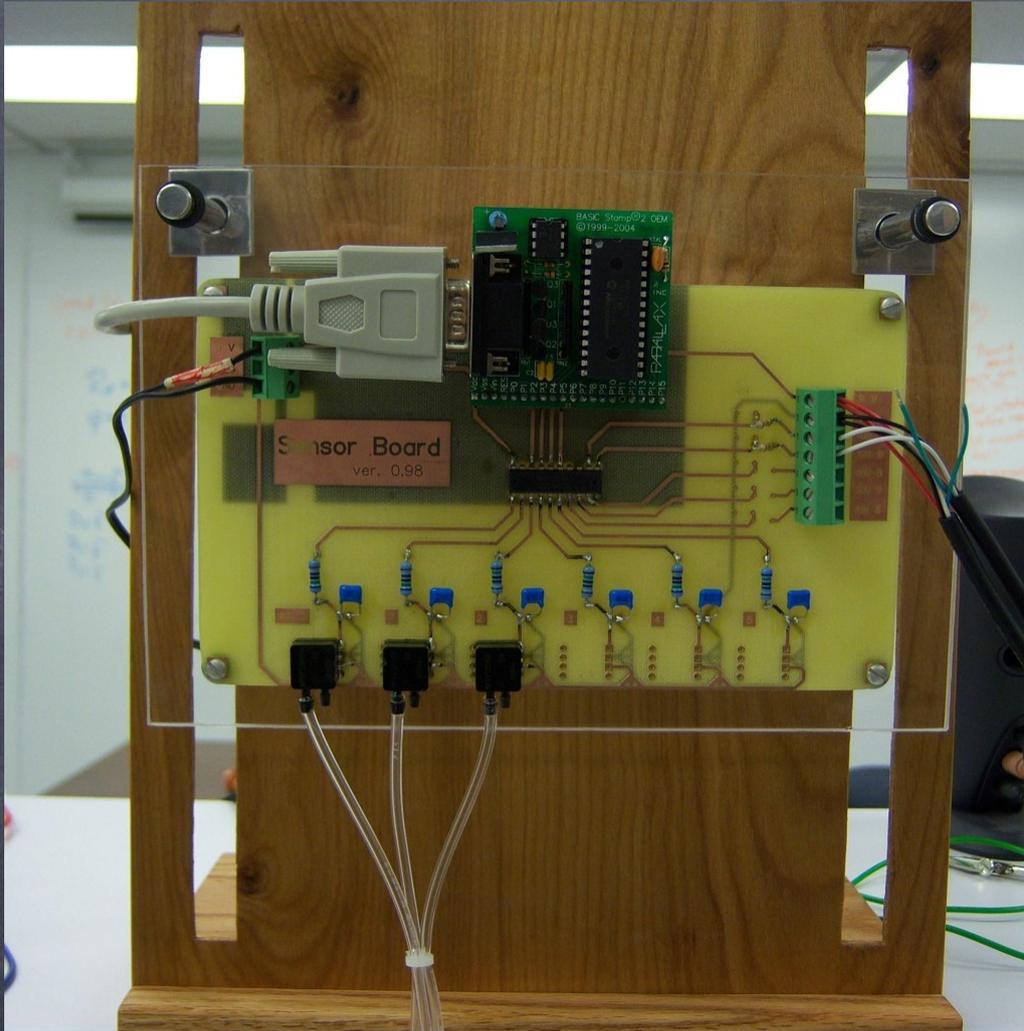
Computer

- Python
- Real Time Plotting with GNU Plot

```
if __name__ == '__main__':  
    serial = connect_serial()  
    filesave = csv.writer(open("data.csv", "wb"))  
    while True:  
        data = serial.readline().split()  
        filesave.writerow(data)  
        plot(data)
```

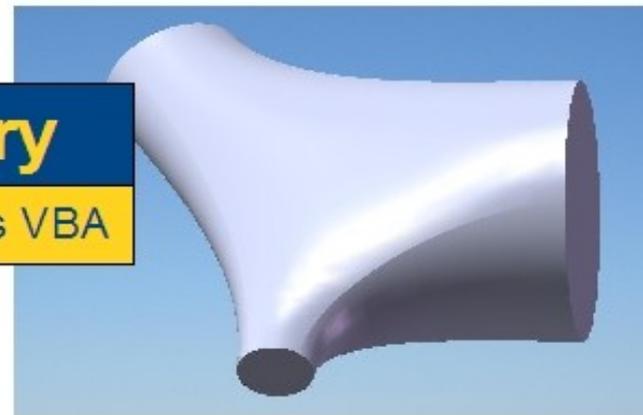


Basic Stamp
Passive Filter
Pressure Sensors
Flow Sensors
Ruby Interface



Generate Junction Geometry

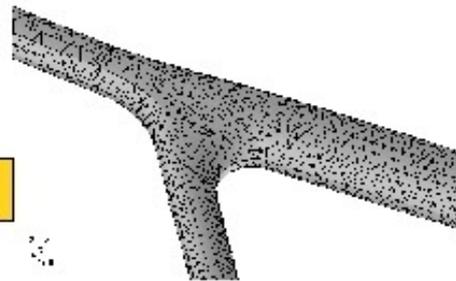
Junction code (C++) → SolidWorks VBA



Simulations

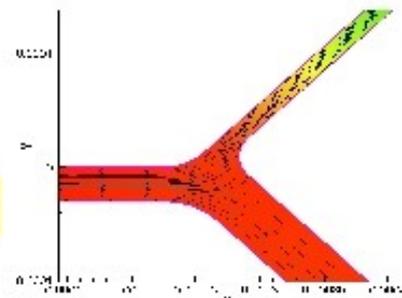
Generate Mesh

Junction Code → GAMBIT



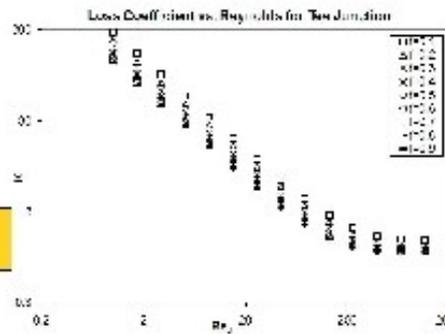
Simulate Flow

Junction Code → FLUENT



Analyze Results

Junction Code → tsv

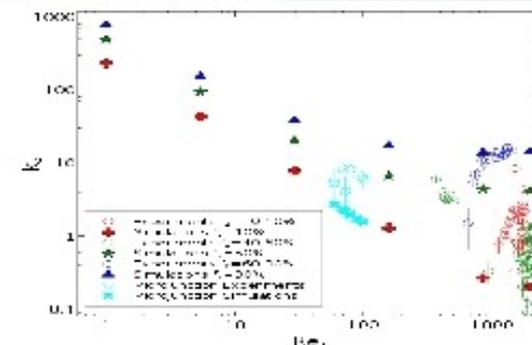
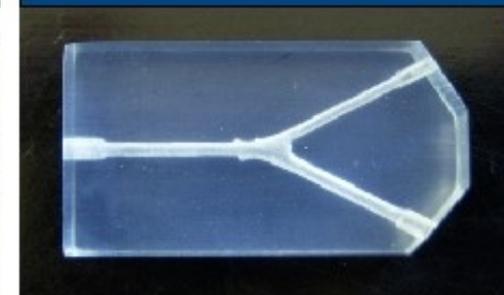


Experiments

Milliscale



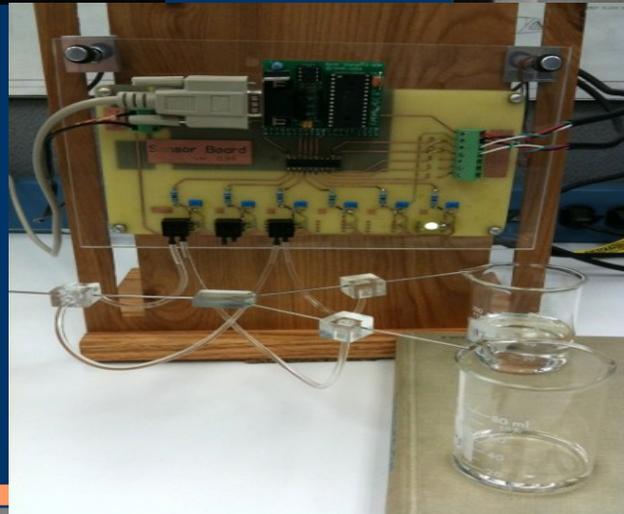
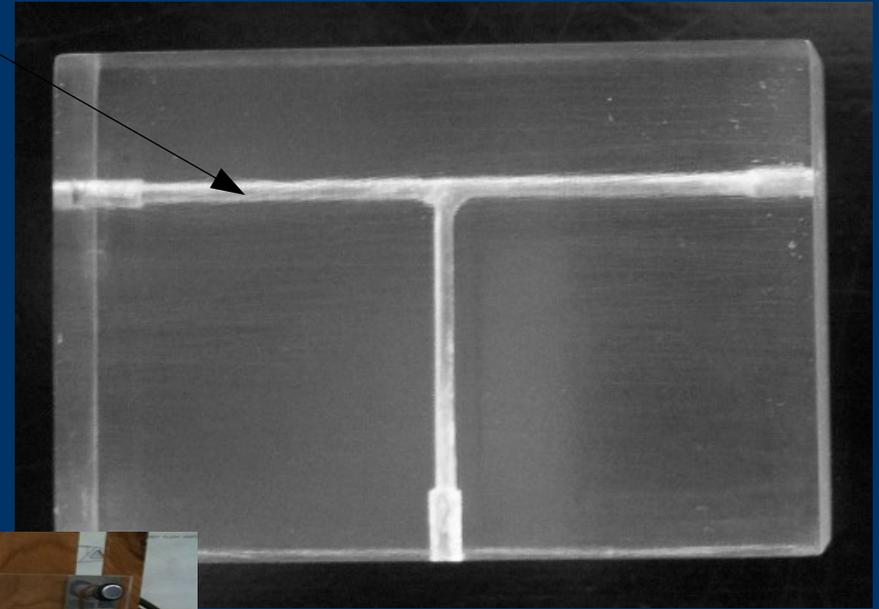
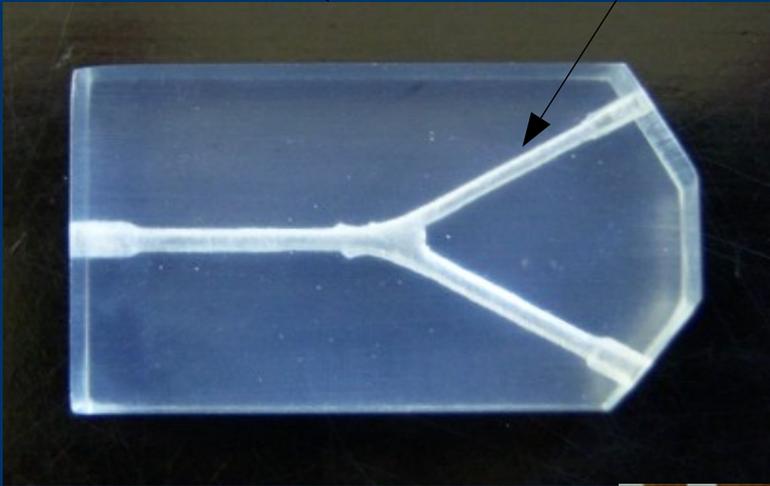
Microscale



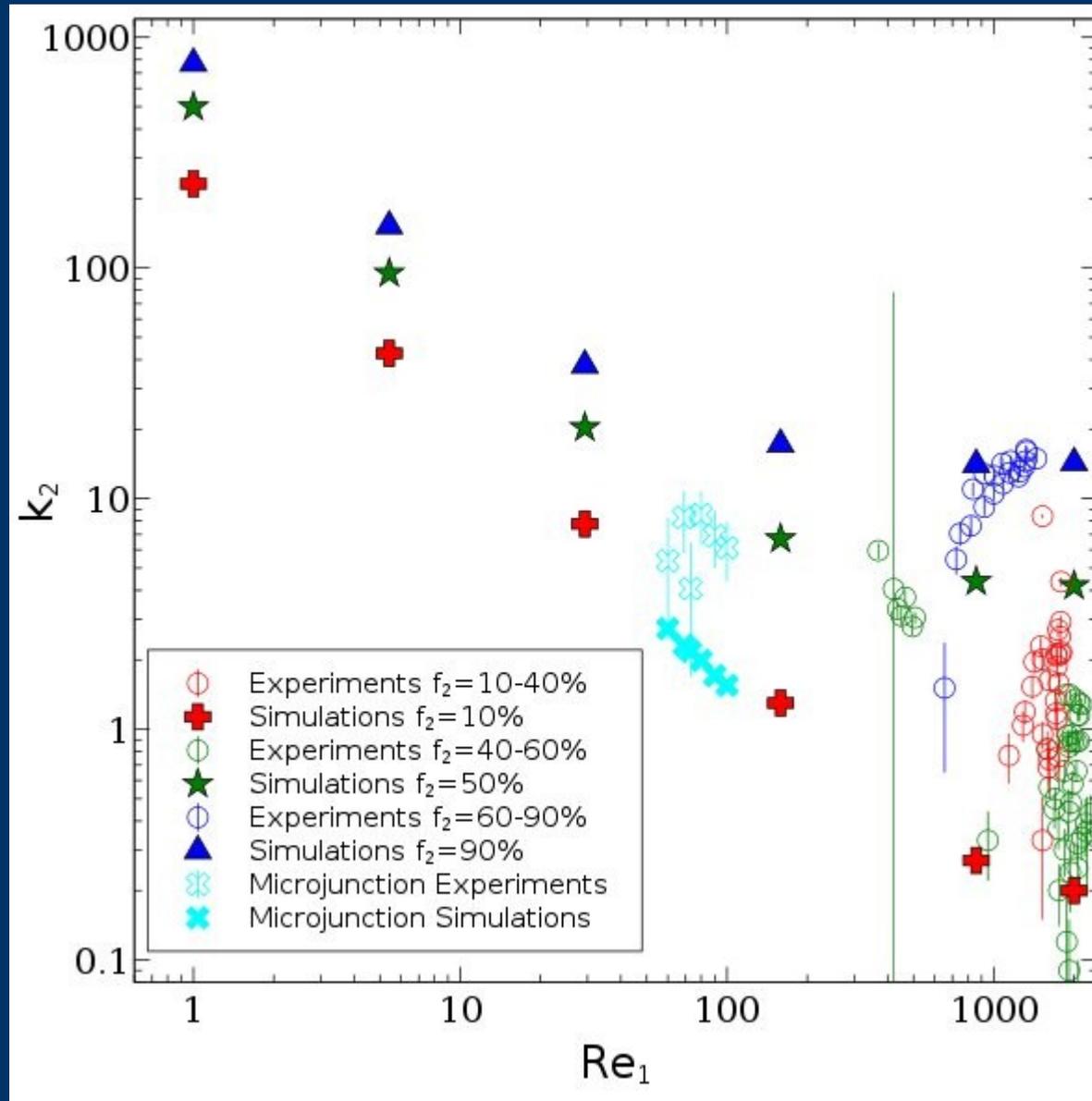
Microscale Junction Experiments

Stereo
Lithography
Technique

Diameter =
0.75 mm

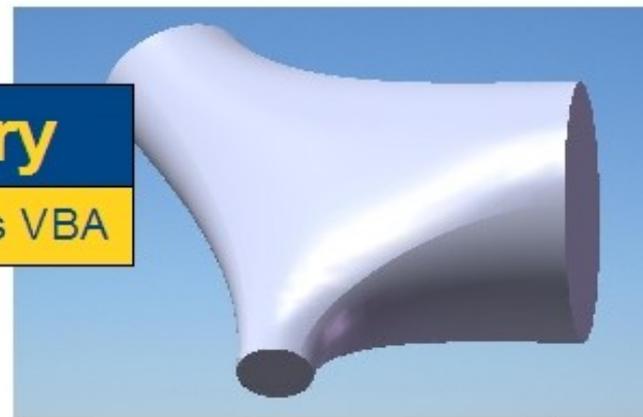


Results - Simulations/Milliscale/Microscale



Generate Junction Geometry

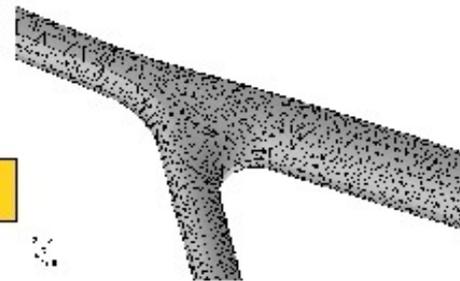
Junction code (C++) → SolidWorks VBA



Simulations

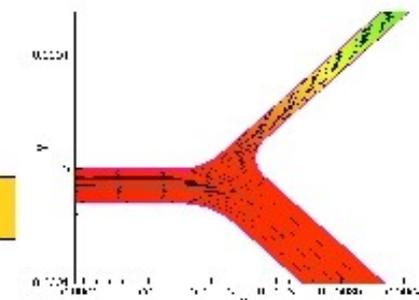
Generate Mesh

Junction Code → GAMBIT



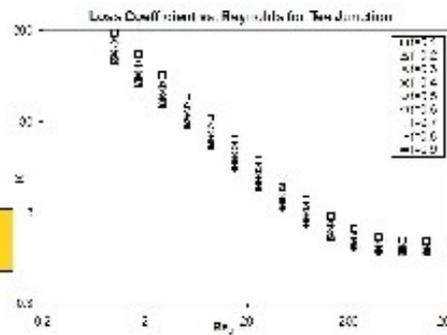
Simulate Flow

Junction Code → FLUENT



Analyze Results

Junction Code → tsv

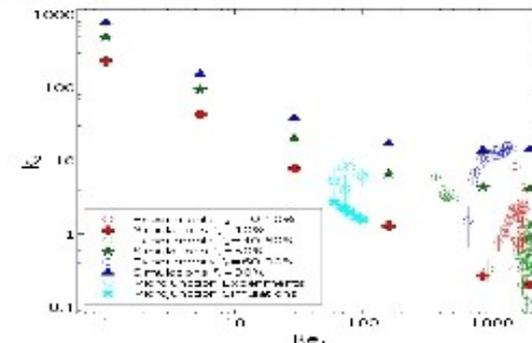
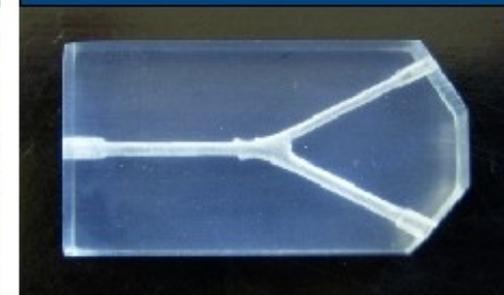


Experiments

Milliscale



Microscale



Using Network Simulations to Understand Non-Darcy Flow

Objectives

Develop an algorithm to create, mesh, and perform CFD on simplified models of real porous media networks.

Equal number of entry and exit pores (no splits)

90° elbows only

No overlap of pores within media

Compare results of CFD to FTPM and empirical data in literature.

Modify algorithm to allow for complex models of real porous media.

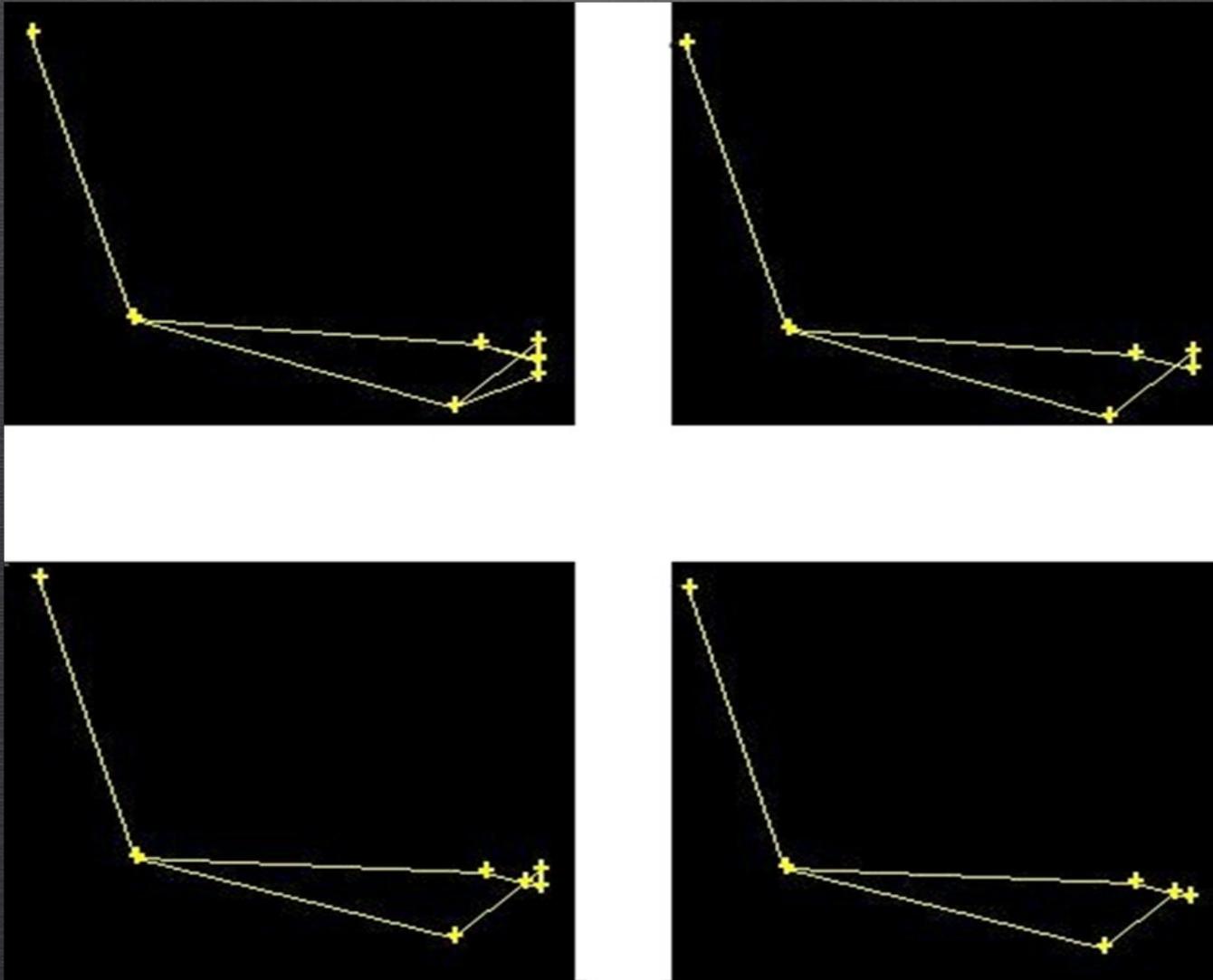
Splits with arbitrary angles

Overlaps

Using Network Simulations to Understand Non-Darcy Flow

Implementation of Objectives

Use custom codes to extract FTPM networks from code

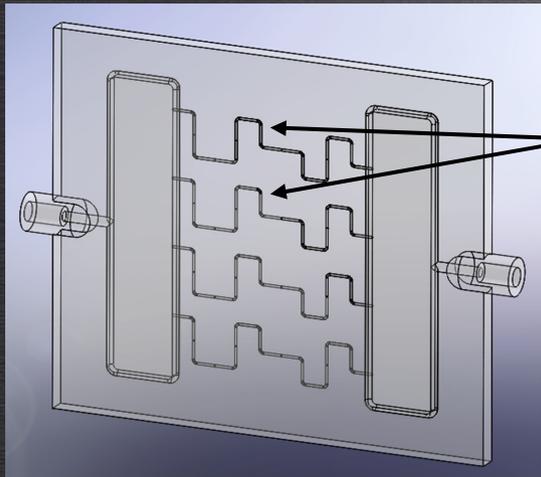


Using Network Simulations to Understand Non-Darcy Flow

Implementation of Objectives

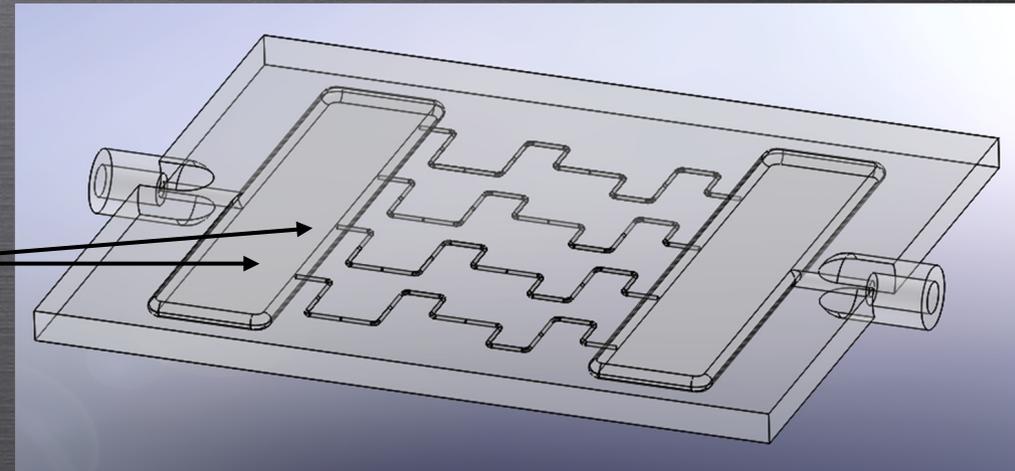
Use Solidworks to create network designs

Phase 1



90° elbows

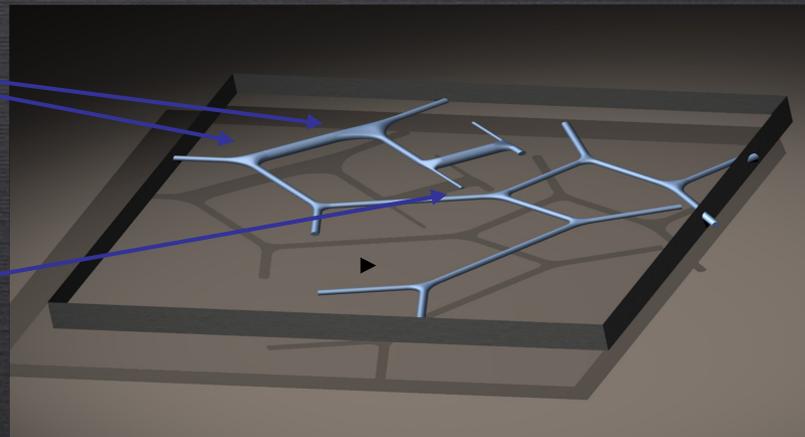
No overlapping



Phase 2

Arbitrary angles

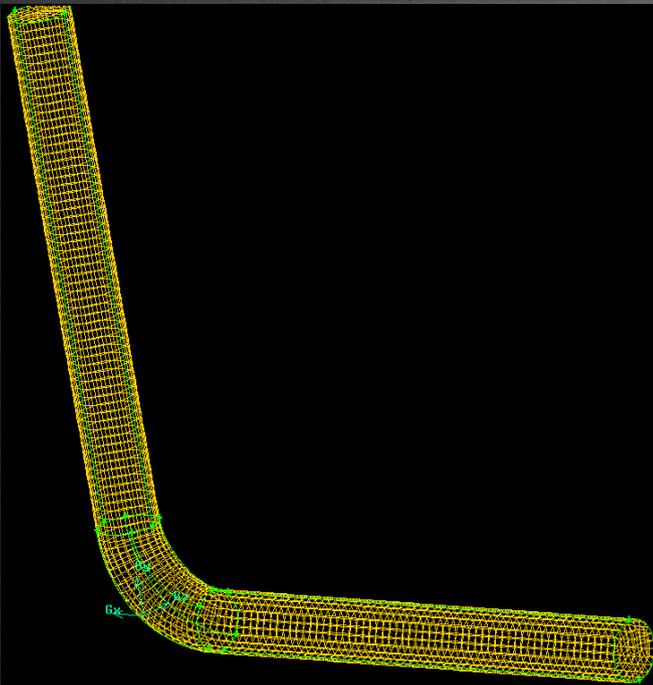
Possible overlaps



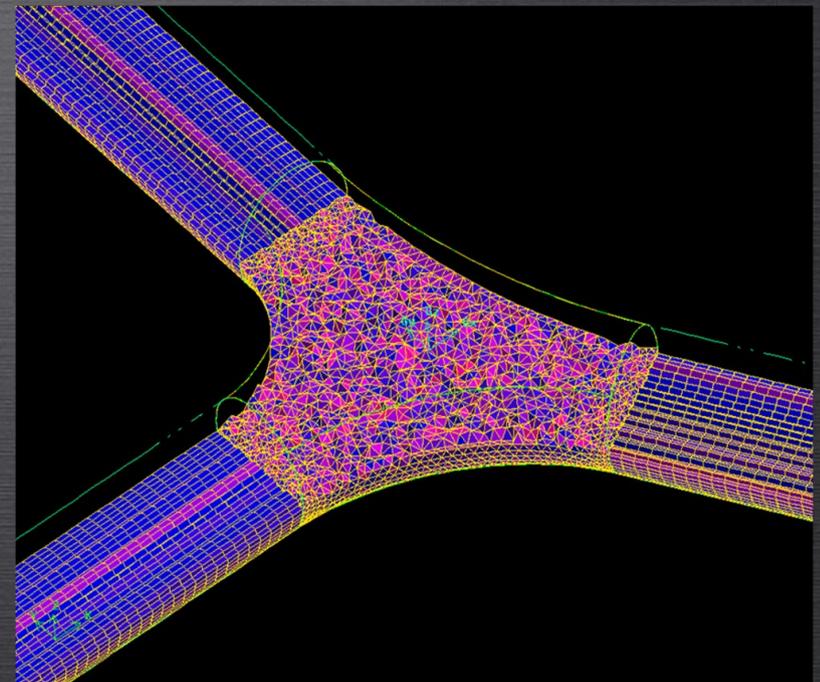
Using Network Simulations to Understand Non-Darcy Flow

Implementation of Objectives

Use Gambit to mesh networks



Close up of 90° elbow

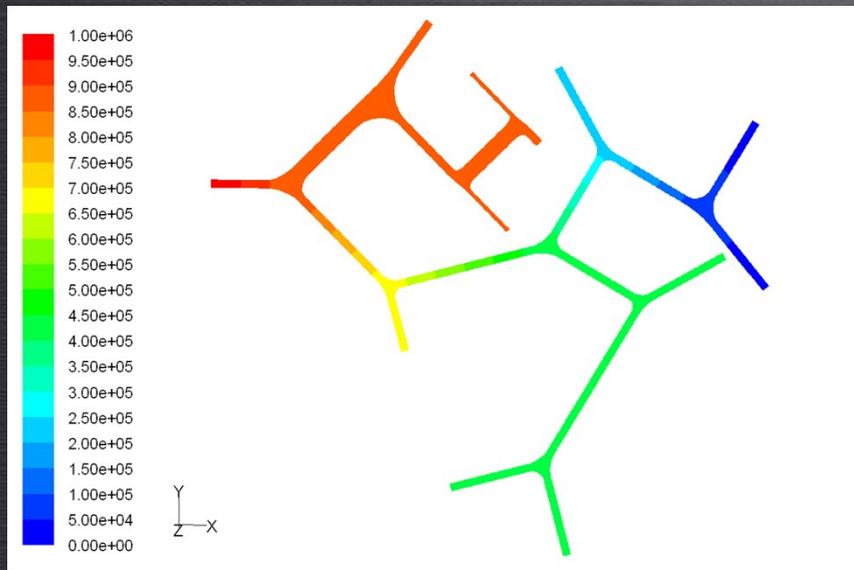


Close up of arbitrary junction

Using Network Simulations to Understand Non-Darcy Flow

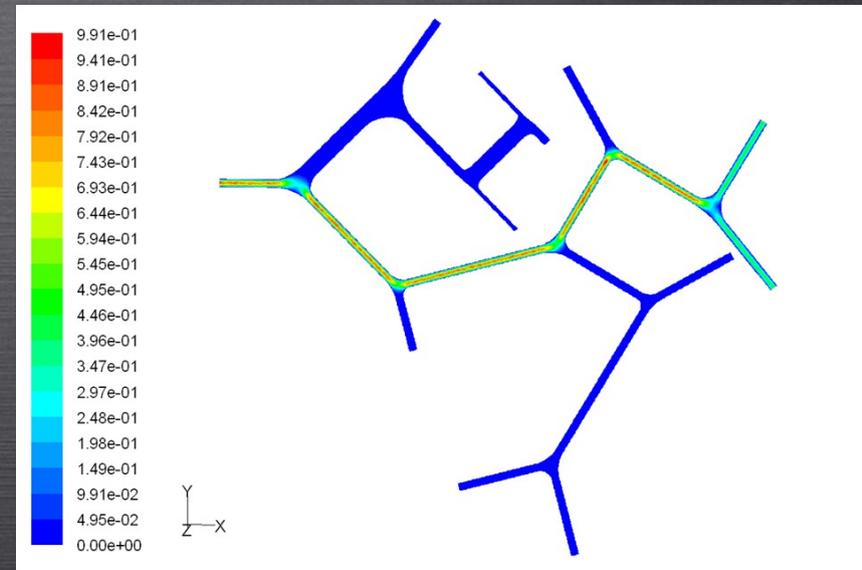
Implementation of Objectives

Use Fluent (CFD) to obtain κ and β from pressure and velocity data



Contours of Static Pressure (pascal)

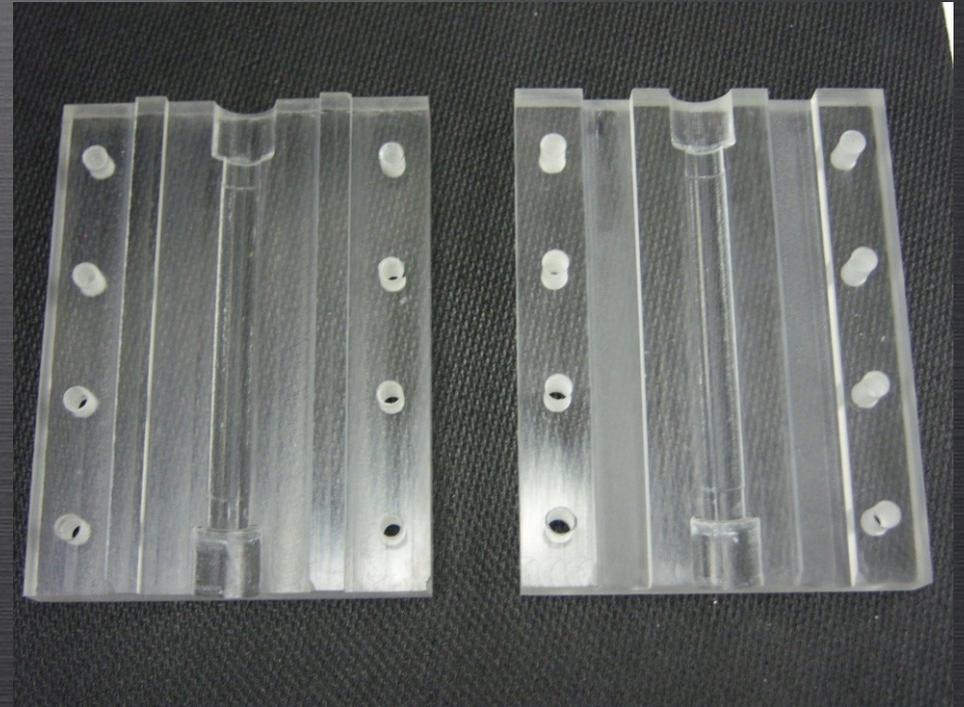
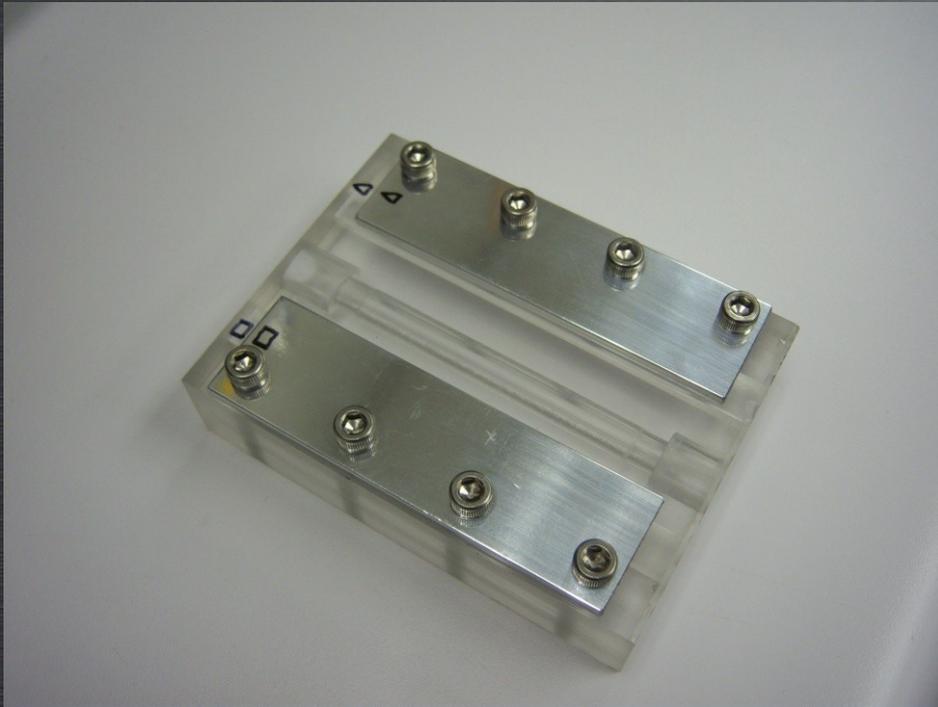
FLUENT 6.3 (3d, pbns, lam)



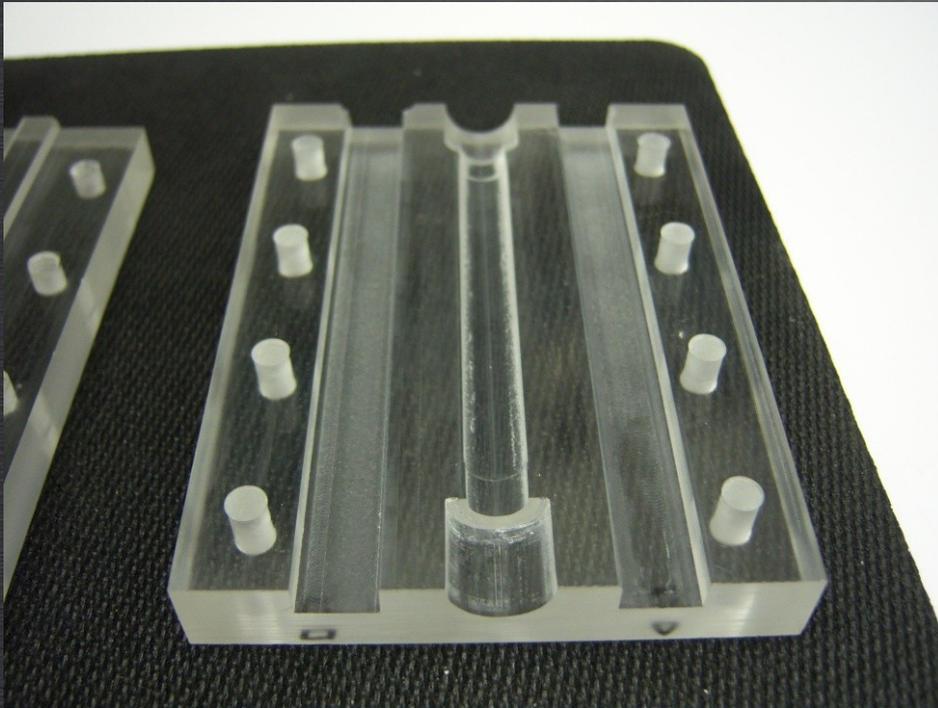
Contours of Velocity Magnitude (m/s)

FLUENT 6.3 (3d, pbns, lam)

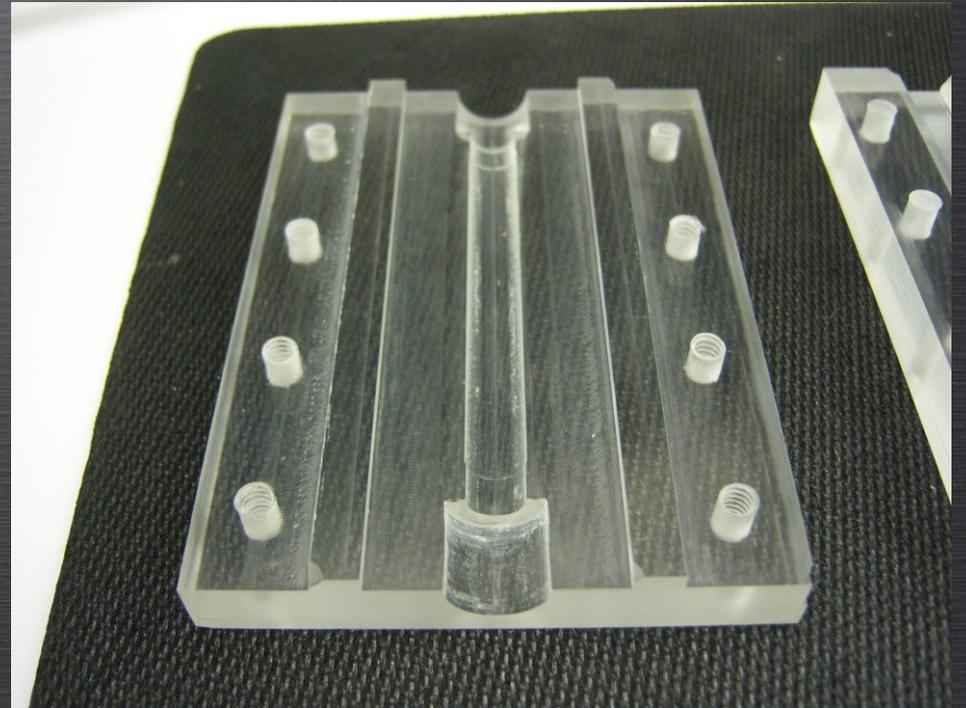
Bio-Scaffold Testing



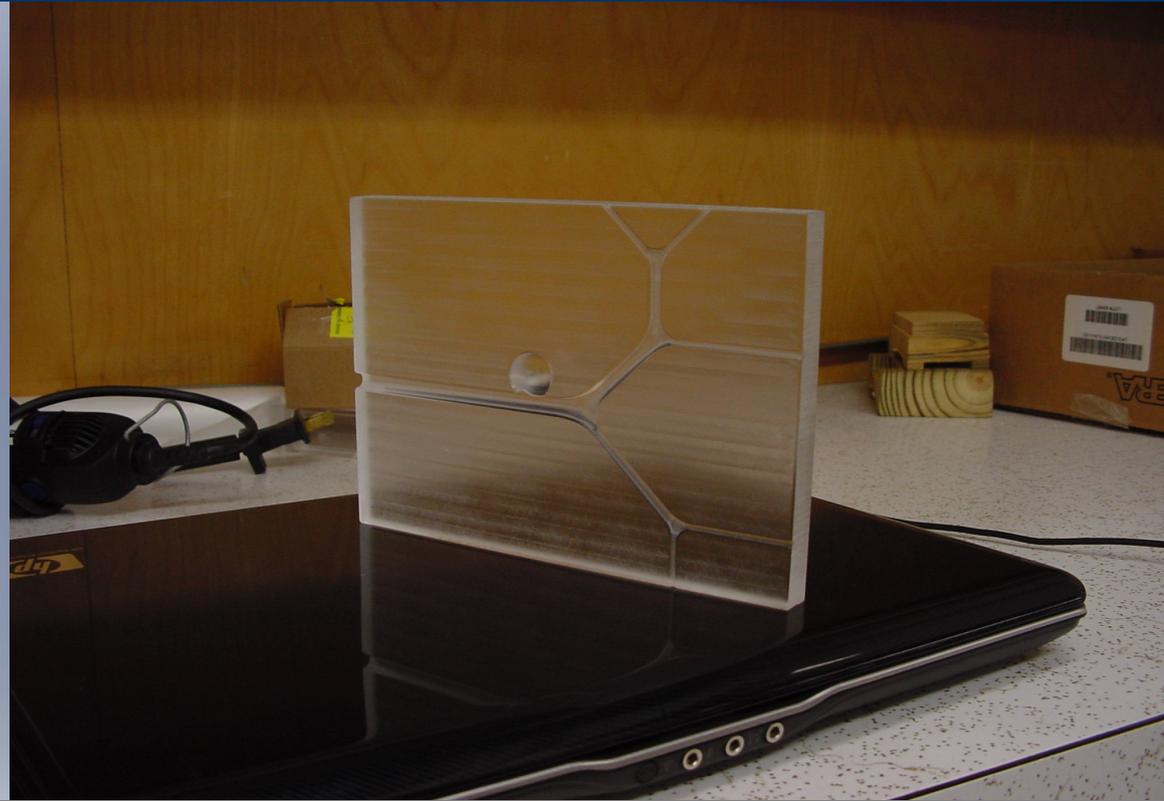
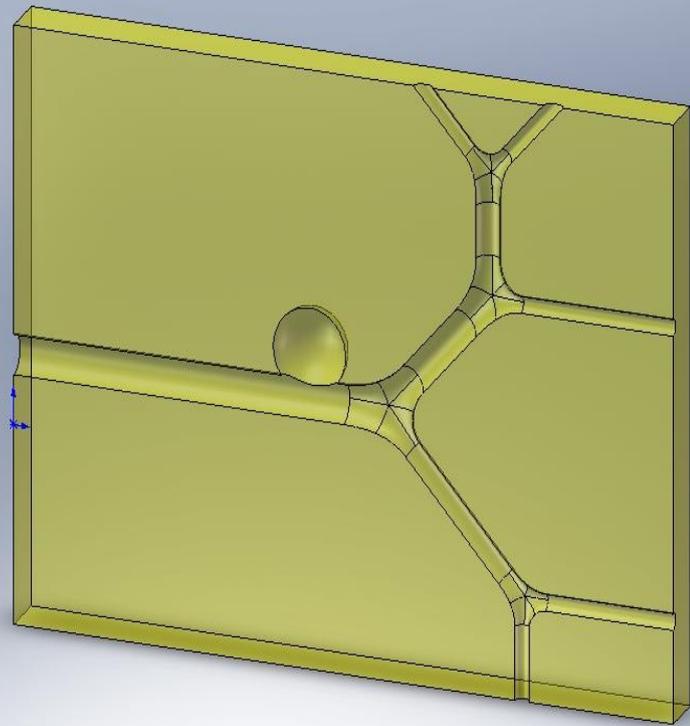
Top



Bottom



Renal Artery Aneurysm Experiments

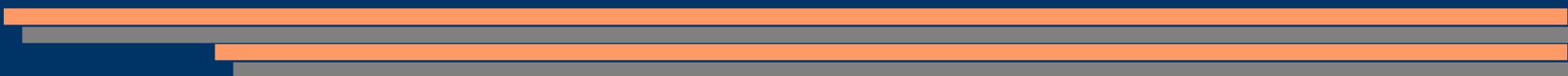


PDMS Microtubes



Acknowledgments

The Office of Research and Grants at the University of Central Oklahoma is acknowledged for support of this research. The Donors of The Petroleum Research Fund, administered by the American Chemical Society, are also be acknowledged for support of this research through grant PRF# 47193-B9. National Science Foundation EPSCoR Research Opportunity Award Program

The bottom of the slide features three horizontal bars of varying lengths and colors. The top bar is a solid orange color and spans the entire width of the slide. Below it are two grey bars, also spanning the width, with the second grey bar being slightly shorter than the first. The bottom-most bar is another orange bar, shorter than the top one, positioned towards the right side of the slide.

Applications

Flow in microfluidic flow networks and flow in porous networks are of interest in many engineering applications.

Applications include porous media, micro-power generation, biomedical, computer chips, chemical separation processes, micro-valves, micro-pumps, and micro-flow sensors

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